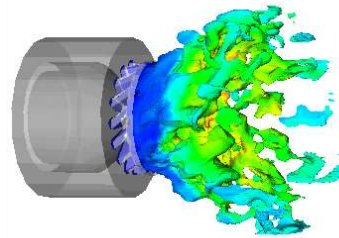


# Center for Advanced Turbomachinery and Energy Research Vasu Lab

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## Effect of Impurities on Supercritical CO<sub>2</sub> Combustion

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Supercritical CO<sub>2</sub> Symposium 2022

Virtual presentation

# Outline

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- Introduction
- Objective
- Approach
- Results and Discussion
- Conclusions

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# Introduction

(Why to think about impurities?)

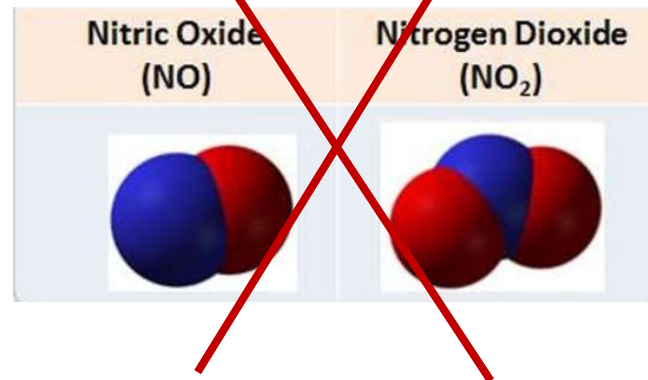
# Important features of direct-fired sCO<sub>2</sub> cycle

## CO<sub>2</sub> capture



Picture source:  
<http://www.21stcentech.com/environment-update-carbon-capture-technique-revealed/>

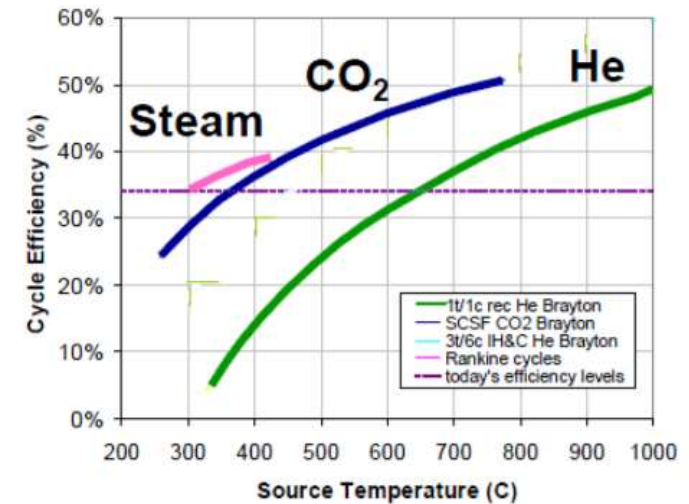
## No Nitrogen oxides



Picture source: <https://www.colorado.edu/aiq/resources/nitrogen-oxides/>

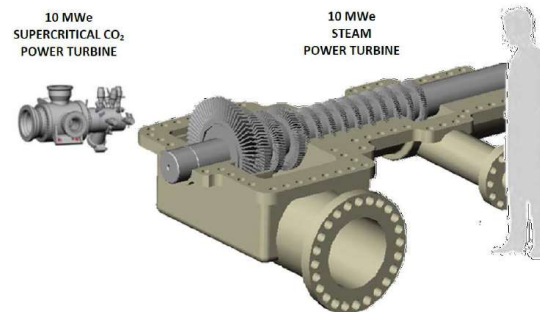
## Higher efficiency

Cycle Efficiencies vs Source Temperature for fixed component efficiency



Source: Jason Wilkes, SwRI tutorials (sCO<sub>2</sub> symposium'18)

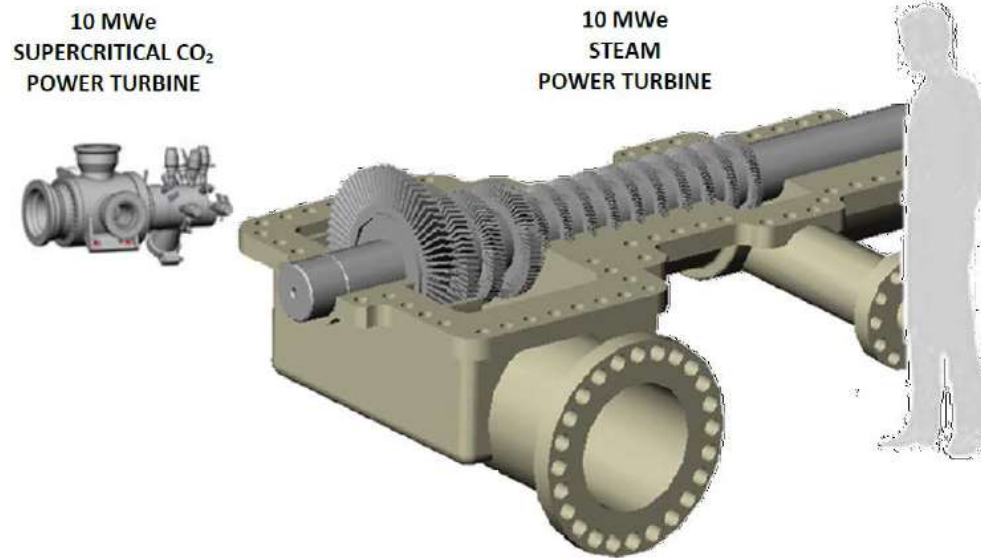
## Compactness



Source: Persichilli et al. (2012) and SwRI tutorials (sCO<sub>2</sub> symposium'16)

# Reason 1: Need of smaller combustors

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Source: Persichilli et al. (2012) and SwRI tutorials (sCO<sub>2</sub> symposium'16)

If turbine design is scaled down from existing ones, combustor may need to be scaled down (for alignment and compatibility).

# Relation between smaller volume and impurities

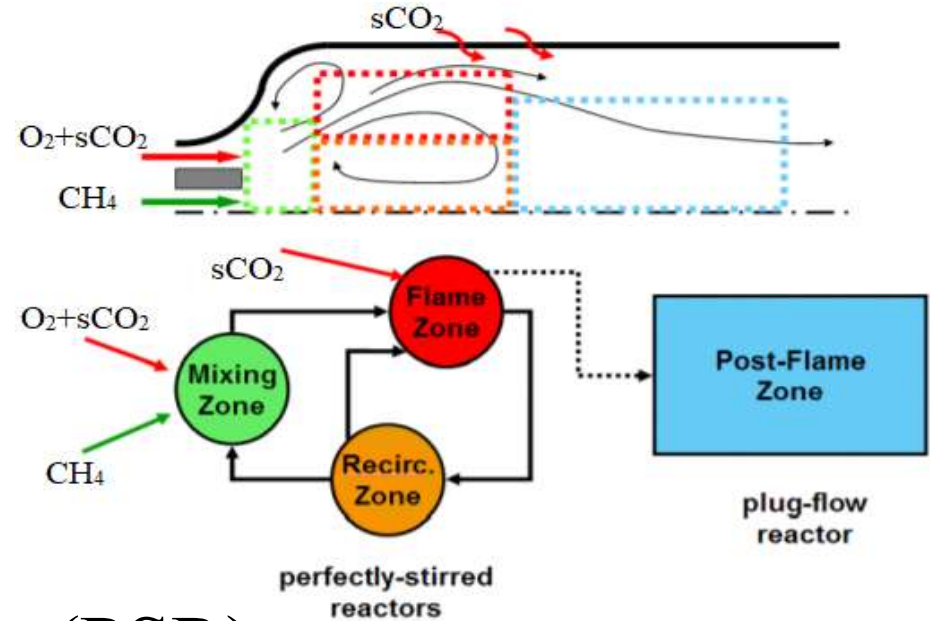
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- Both the volume and recirculation design determines total residence time of combustion mixture
- As we reduce the residence time, complete oxidation of fuel becomes challenging.



# A reactor network analysis

- PSR (also known as WSR) and PFR modeling was extensively used in 1950s to develop gas turbine combustors.



- Perfectly-Stirred Reactor (PSR) accounts the time associated with the molecules to enter and exit the reactor, called reactor residence time.

# Reactor network modeling

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- The UCF 1.1 (derived from Aramco 2.0 and updated with MD reaction rates) mechanism is used.
- All real gas corrections are incorporated through CHEMKIN PSR and PFR models.
- Soave-Redlich-Kwong (SRK) equation of state is used based on our prior work<sup>2</sup>.

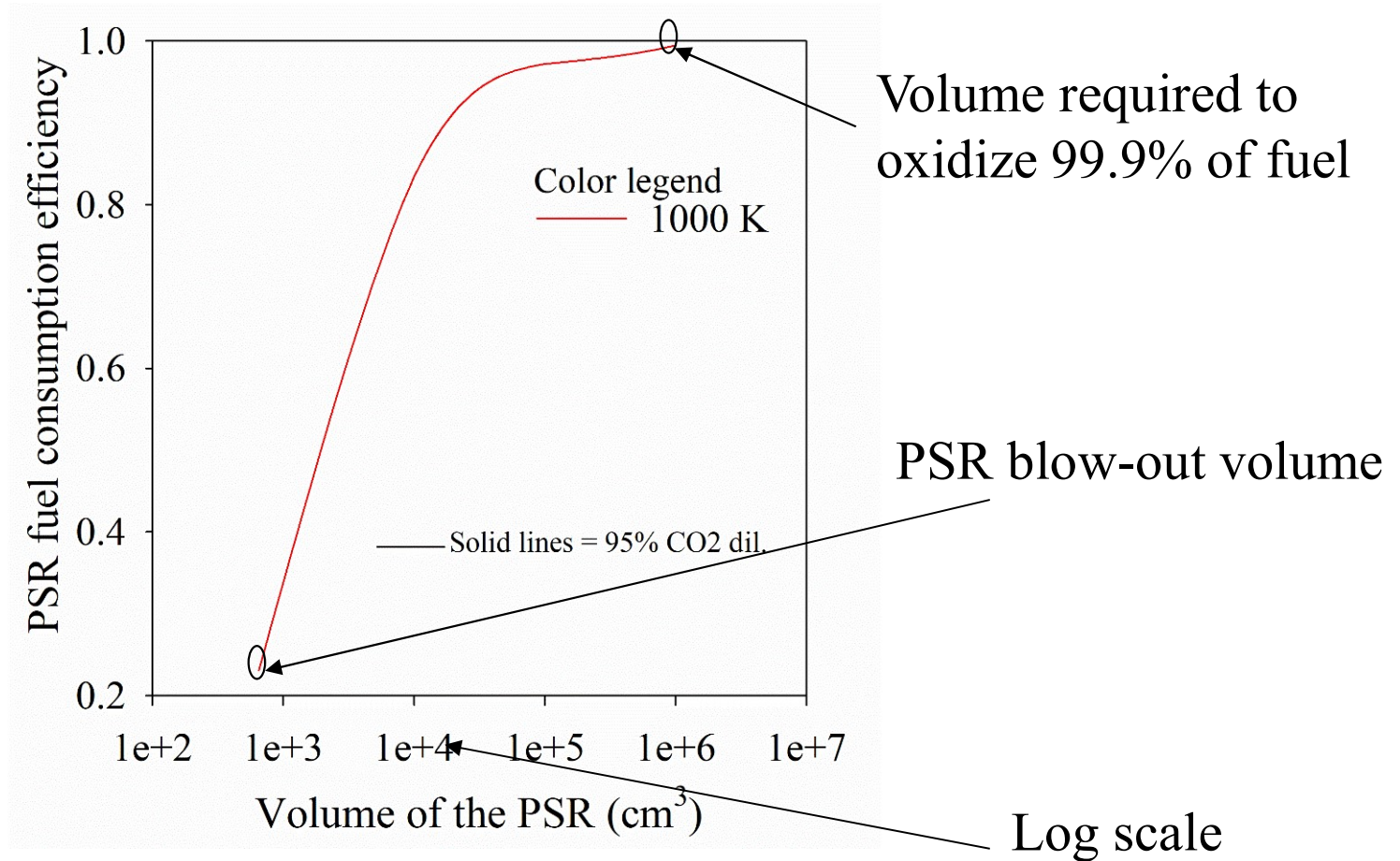
1. Schmitt, R., Butler, P., and French, N. B., 1993, "CHEMKIN Real Gas," UIME PBB, pp. 93-006.

2. Manikantachari, K., Martin, S., Bobren-Diaz, J., and Vasu, S., 2017, "Thermal and transport Properties for the simulation of Direct-Fired sCO<sub>2</sub> Combustor," Journal of Engineering for Gas Turbines and Power. GTP-17-1210, 139(12), 121505.



# Volume of sCO<sub>2</sub> reactor Vs Fuel oxidation

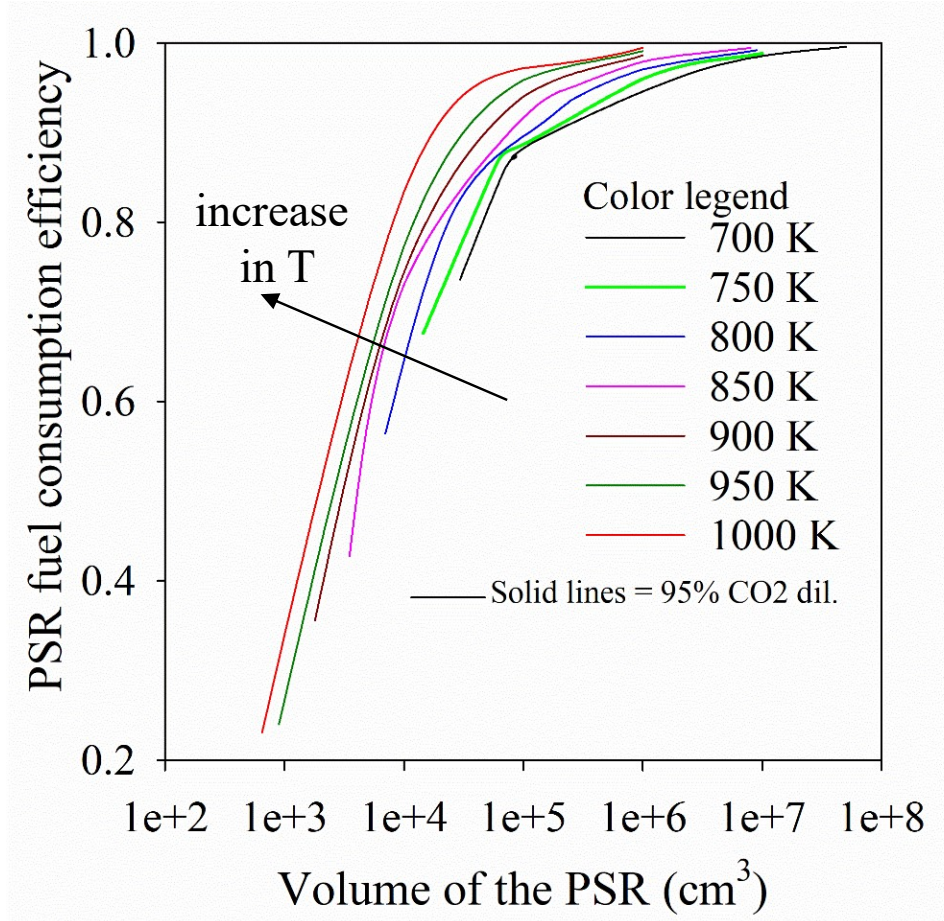
**At 95% primary  
dilution**



- Burning final trace of fuel needs large reactor volume.

# Volume of sCO<sub>2</sub> reactor Vs Fuel temperature

**At 95% primary  
dilution  
&  
Various reactant  
temperatures**



- As the reactant temperature increases the required blow-out volume and required reactor volume decreases.



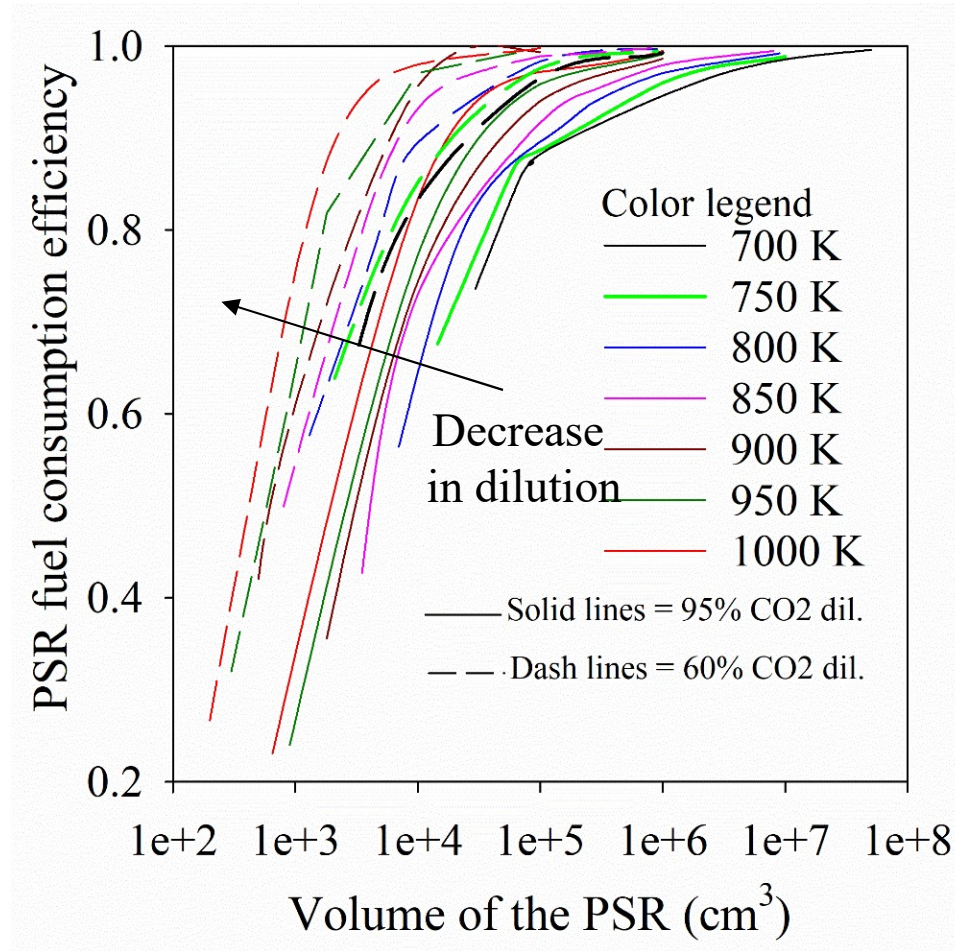
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# Volume of sCO<sub>2</sub> reactor Vs sCO<sub>2</sub> fraction

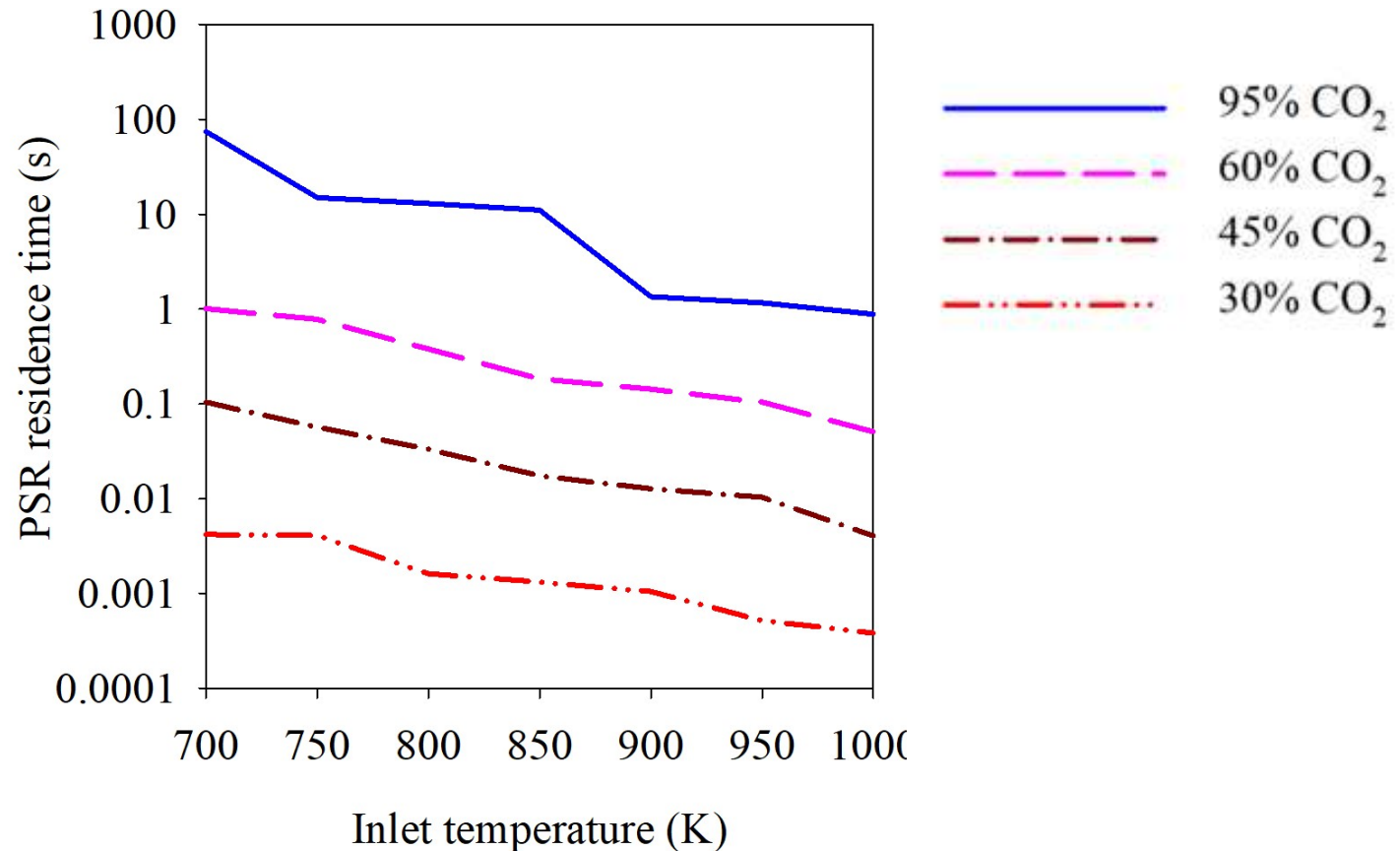
## Influence of primary dilution level of combustor volume



Dash lines → 60% dilu.  
Solid lines → 95% dilu.

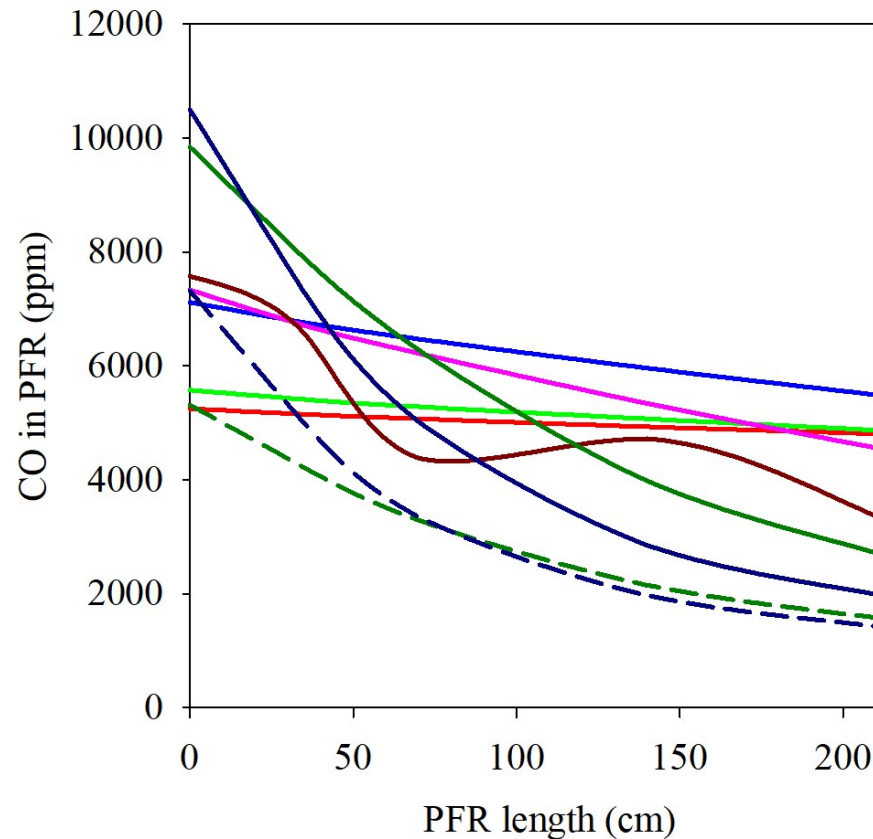
- Lesser primary dilution is more advantageous during cold start.

# Residence time requirement

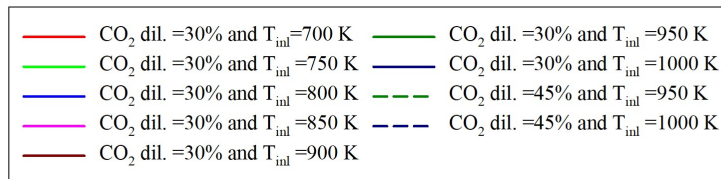


- With 95 % CO<sub>2</sub> dilution it is not practical to burn methane within reasonable residence time

# Emissions at the combustor exit

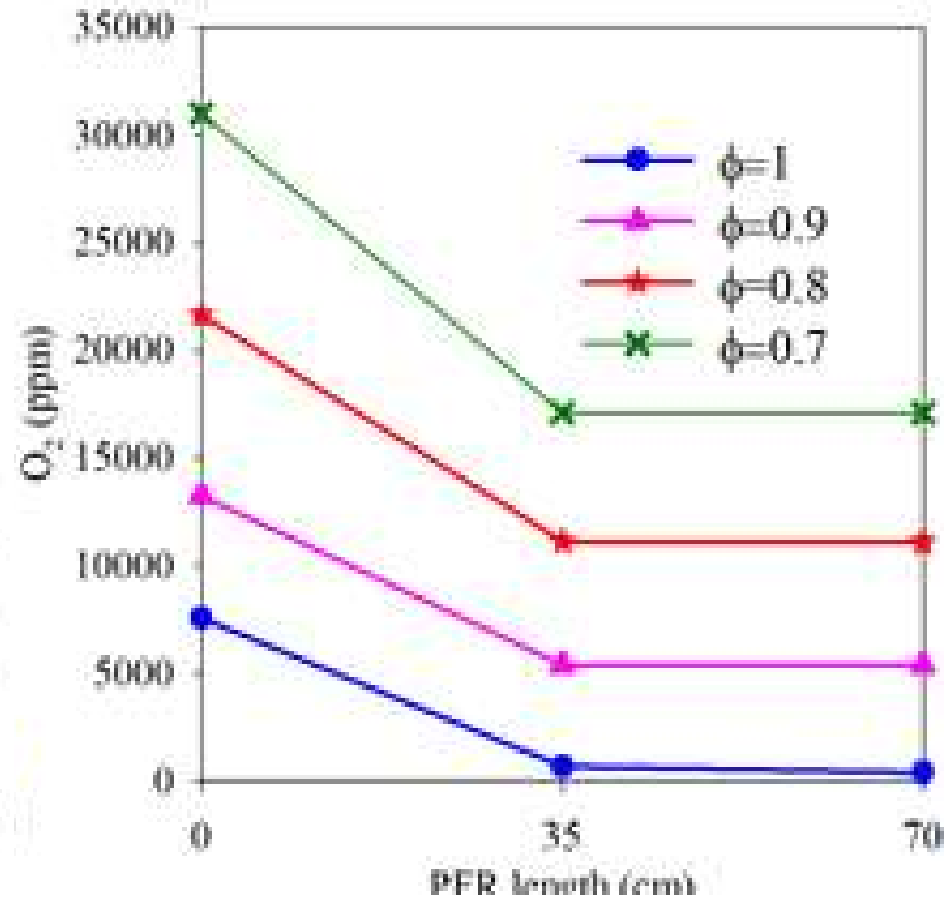
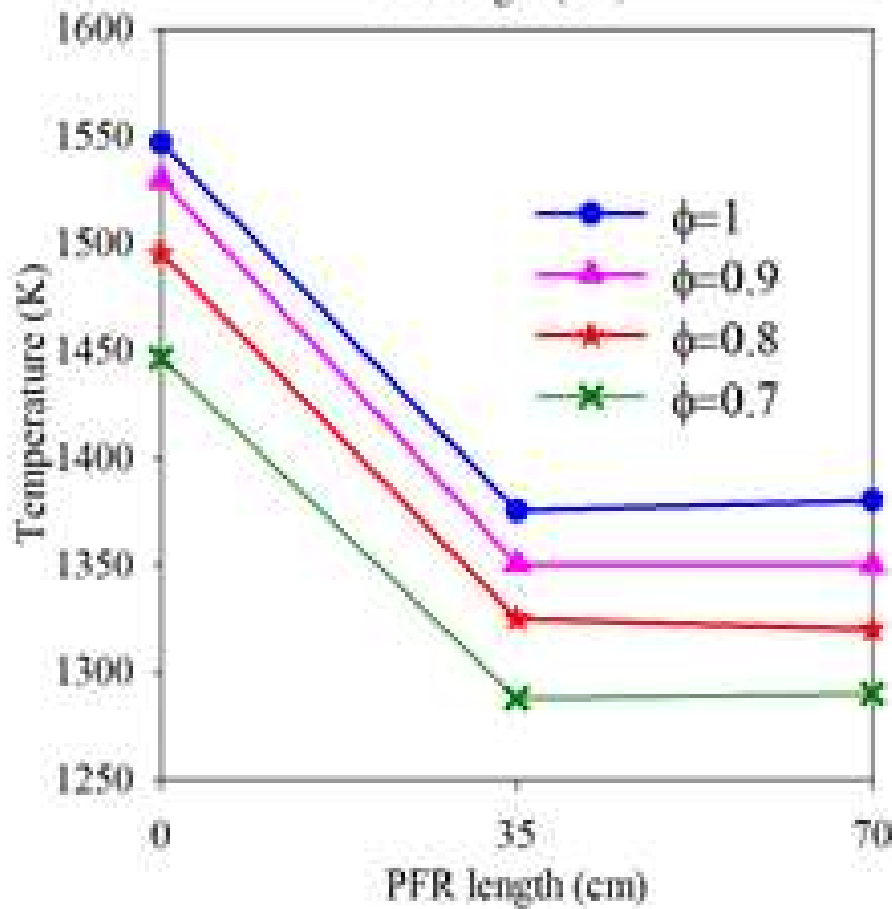


- CO can be a major concern when we try to scale down sCO<sub>2</sub> combustor



# Complete oxidation of CO with lean strategy

## The effect of lean combustion on CO

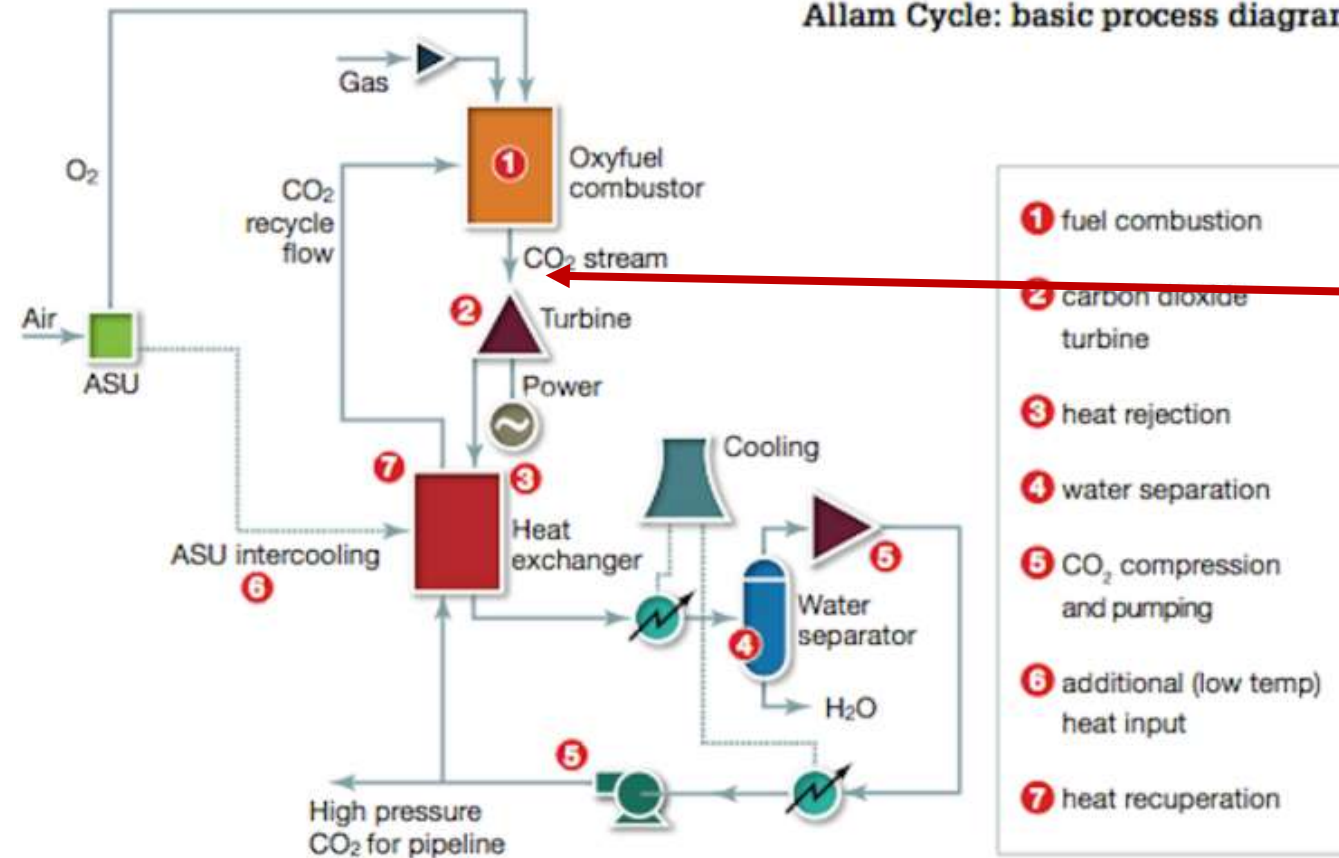


- If CO is oxidized with excess  $O_2$ , then  $O_2$  will remain at the exit



# Emissions can become impurities

Allam Cycle: basic process diagram

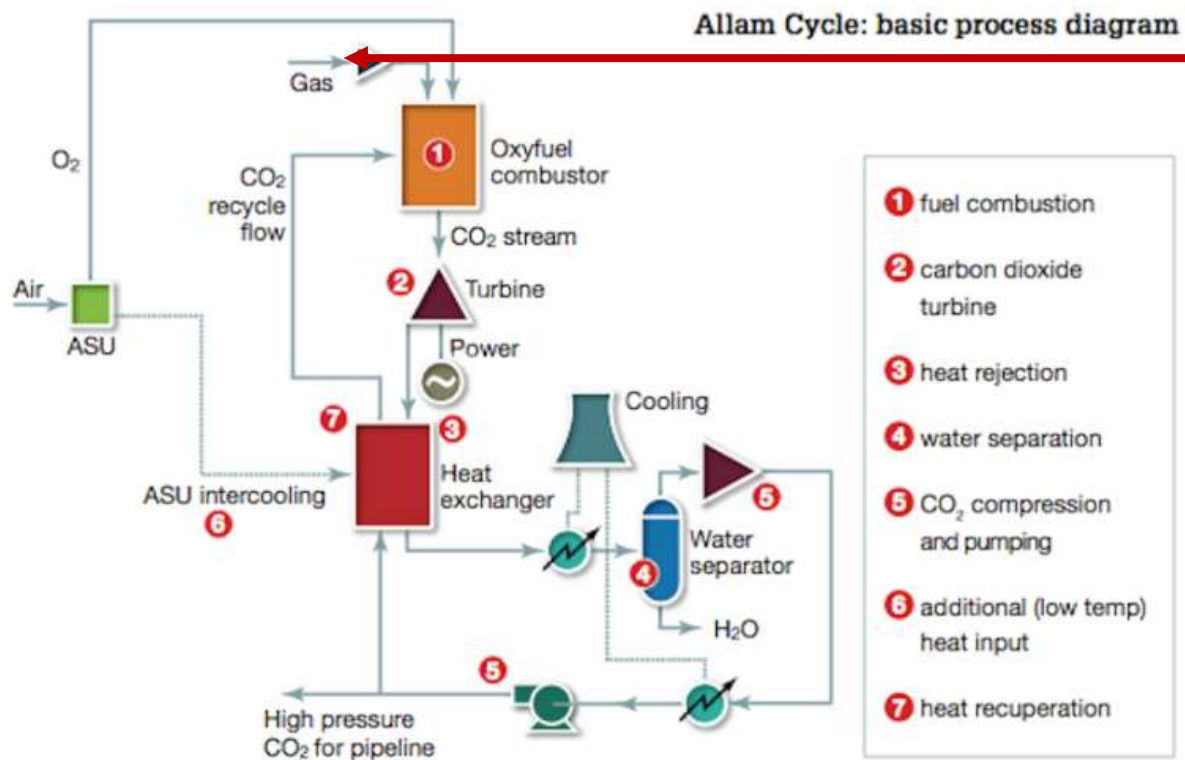


- Emissions other than pure H<sub>2</sub>O and CO<sub>2</sub> in the exhaust must be treated as impurities in a semi closed loop operation



# Reason 2: Impurities in fuel

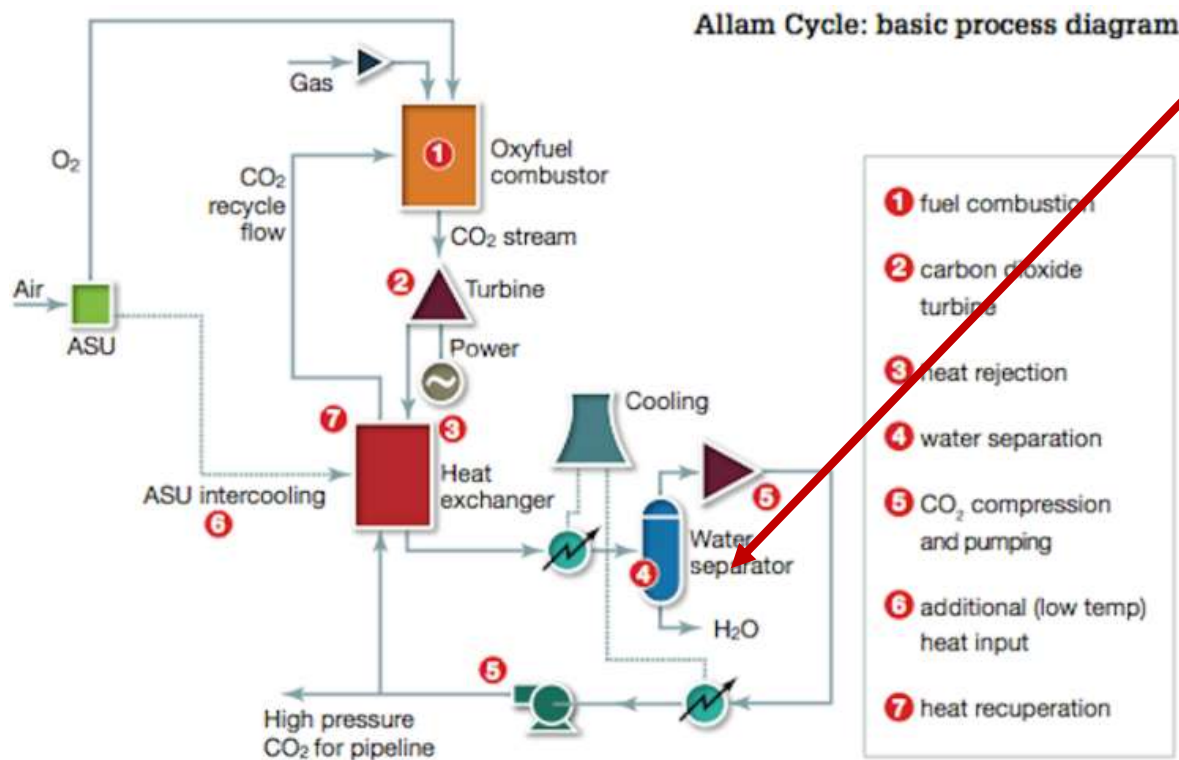
- Natural gas can have impurities such as  $H_2S$ ,  $H_2O$ ,  $N_2$ ,  $CO_2$  etc. and they can



Fuel impurities

## Reason 3: Inefficient water separation

- Natural gas can have impurities such as  $\text{H}_2\text{S}$ ,  $\text{H}_2\text{O}$ ,  $\text{N}_2$ ,  $\text{CO}_2$  etc. and they can



Inefficient water separation can also add traces of  $\text{H}_2\text{O}$  to incoming  $\text{CO}_2$

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# Objective

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# Objective

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High-pressure operation is the major challenges for these combustor development.

Experimentation is expensive, time consuming and even dangerous at these pressures. Therefore, high fidelity simulation tools play a major role in the initial development of this combustor.

High fidelity LES simulations are performed to understand the influence of impurities on sCO<sub>2</sub> combustion

# Cases Considered

Cases investigated in this study	What does the case represent?
<b>Case-1</b>	The re-cycled stream consists of pure CO <sub>2</sub> .
<b>Case-2</b>	The re-cycled CO <sub>2</sub> stream consists of O <sub>2</sub> impurity by 5000 ppm
<b>Case-3</b>	The re-cycled CO <sub>2</sub> stream consists of H <sub>2</sub> O impurity by 5000 ppm

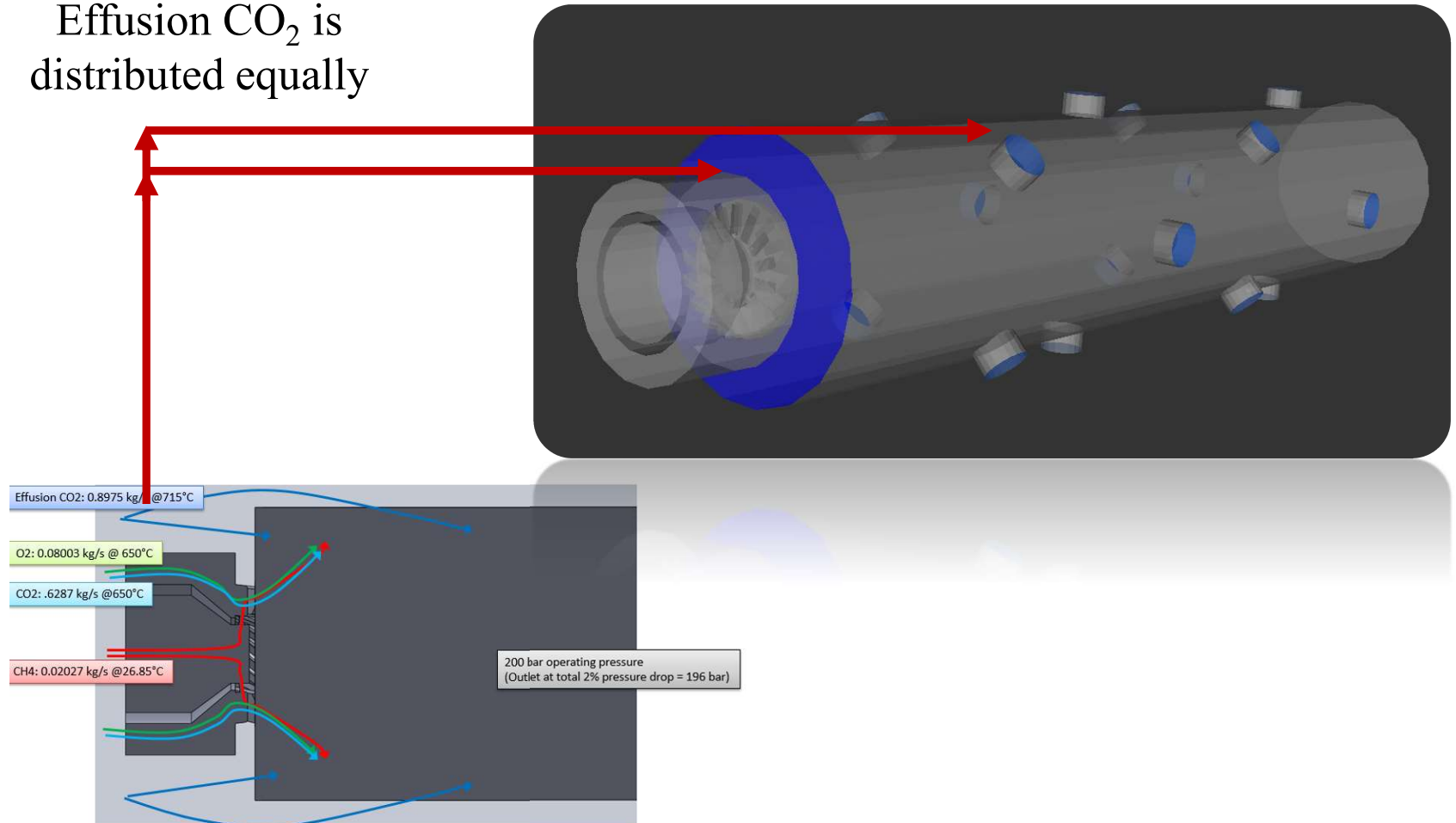
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# Approach

# Geometry

- SwRI sCO<sub>2</sub> combustor design is used

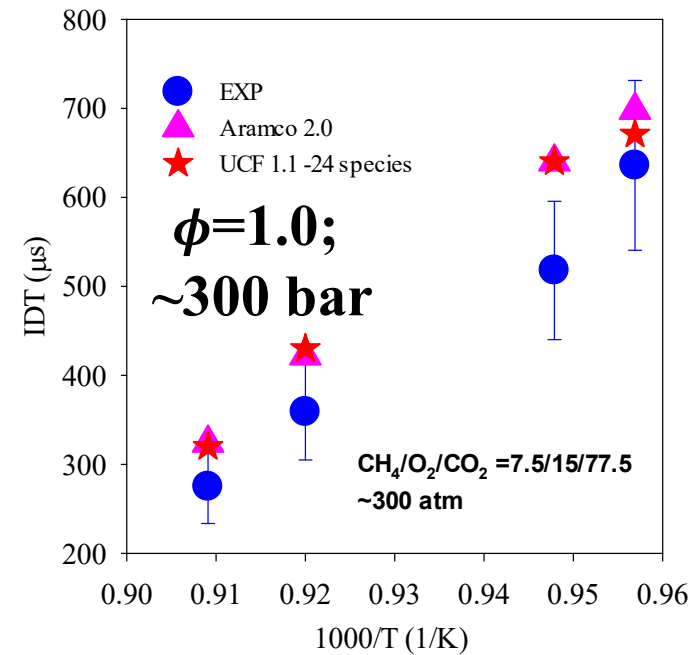
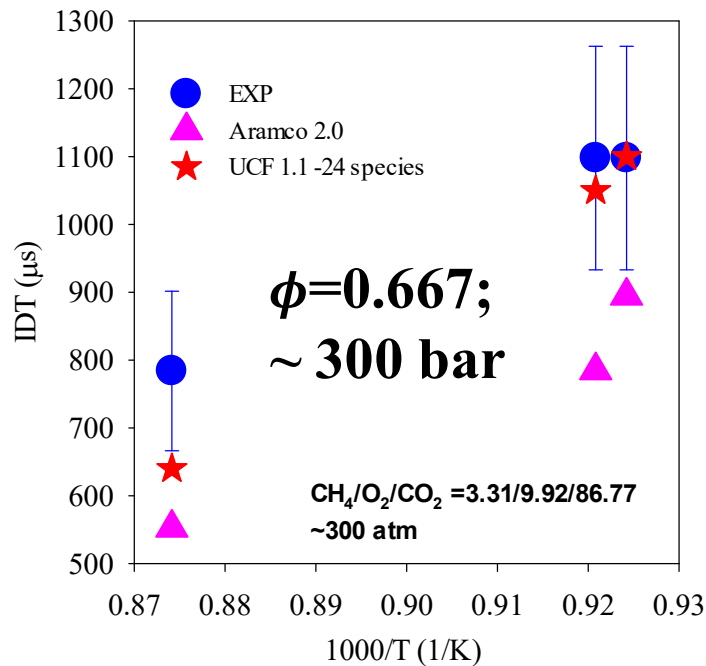
Effusion CO<sub>2</sub> is distributed equally





# Chemical Kinetic Mechanism

## High pressure Ignition Delay Times:



- The UCF 1.1 mechanism (Raghu et al., 2018)
  - based on **Aramco 2.0**
  - has important reaction rates calculated by **molecular level simulations.**

# EOS implementation in Converge

- To consider thermodynamic quantities such as internal energy, enthalpy, entropy and specific heats as functions of both temperature and pressure:

Ideal gas		Real gas		Departure function
$U^0(T)$		$U(T, p)$		$\Delta U(T, p)$
$H^0(T)$		$H(T, p)$		$\Delta H(T, p)$
$S^0(T)$	■	$S(T, p)$	■	$\Delta S(T, p)$
$C_v^0(T)$		$C_v(T, p)$		$\Delta C_v(T, p)$
$C_p^0(T)$		$C_p(T, p)$		$\Delta C_p(T, p)$

- These departure functions are correlated, can be conveniently evaluated

through departure function for Helmholtz energy:  $\Delta A(T, p) = \int_{\infty}^v \left( p - \frac{RT}{v} \right) dv + RT \ln \frac{v}{v^0}$

$$\Delta S(T, p) = -\frac{\partial}{\partial T} \Delta A(T, p)$$

$$\Delta U(T, p) = \Delta A(T, p) + T \Delta S(T, p)$$

$$\Delta H(T, p) = \Delta A(T, p) + T \Delta S(T, p) - RT(Z - 1)$$

$$\Delta C_v(T, p) = \frac{\partial}{\partial T} \Delta U(T, p)$$

$$\Delta C_p(T, p) = \frac{\partial}{\partial T} \Delta H(T, p)$$

Source: Converge manual

# Inputs

- Converge<sup>®</sup> is used for simulations

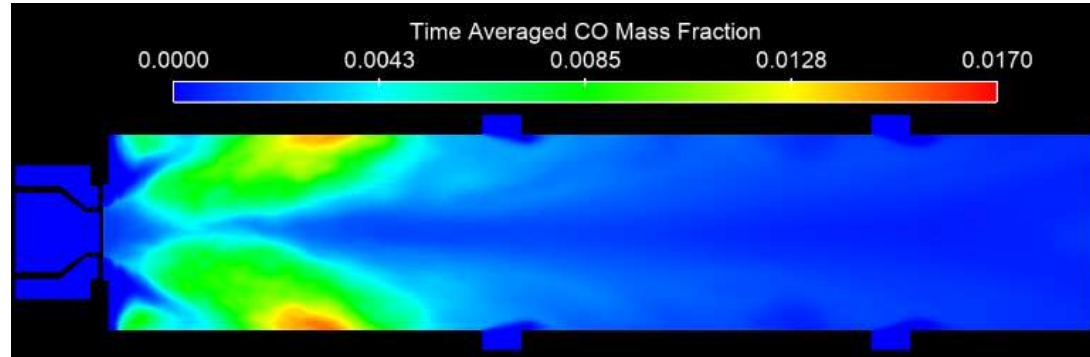
Modeling	Parameter /model chosen
Turbulence modeling	Large-eddy simulation → Viscous One Equation. This model uses sub-grid kinetic energy in modeling the turbulent viscosity
Wall heat transfer modeling	O'Rourke and Amsden
Combustion modeling	SAGE detailed chemistry (all species transport equations are solved).
Number of cells	Approximately six million cells (Adaptive mesh refinement is used)
Equation of state	Soave-Redlich-Kwong equation of state
Viscosity and Thermal conductivity	Pure CO <sub>2</sub> properties between 800 -1600 K from REFPROP are used.
Chemical kinetic mechanism	A UCF 1.1 (23-species) mechanism derived from Aramco 2.0
Simulation time	6 follow-through times

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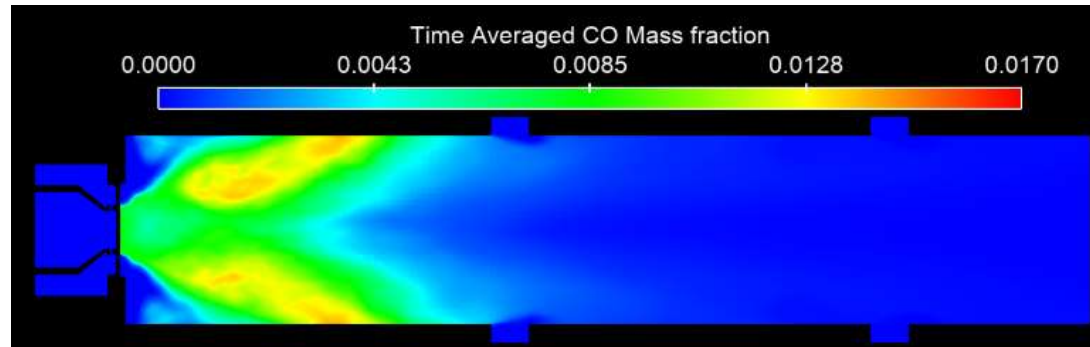
# Results and Discussion

# Time averaged CO mass fraction

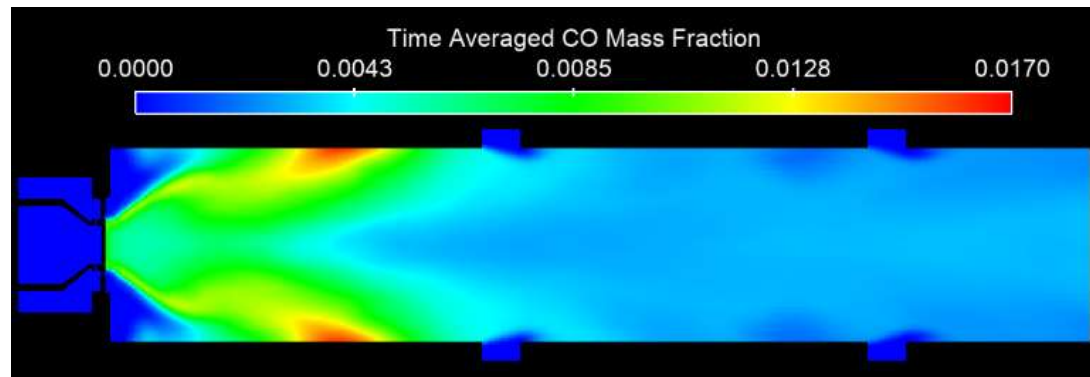
Pure CO<sub>2</sub>



O<sub>2</sub>+CO<sub>2</sub>



H<sub>2</sub>O+CO<sub>2</sub>



# Moles at the Combustor Exit

	CO (ppm)	O2 (ppm)	CH4 (ppm)	Temperature (K)
CO <sub>2</sub>	1194	351	0	1402.3
CO <sub>2</sub> +O <sub>2</sub>	185	2000	0	1415.9
CO <sub>2</sub> +H <sub>2</sub> O	1782	248	0	1408.4

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# Conclusions



# Conclusions

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- Complete oxidation of fuel is challenging while scaling down the size of combustor
- Small impurities can influence the flame structure
- Not much impact on the exit temperature (if sufficient residence time is given)
- Having excess  $O_2$  (5000 ppm) can oxidize CO and minimize CO at the outlet

# Acknowledgments

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Special thanks to Gaurav Kumar and Daniel Lee from Converge Inc., for supporting in CFD simulations.

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# Questions

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