

# Printed Circuit Heat Exchanger and Finned-Tube Heat Exchanger Modeling for a Supercritical CO<sub>2</sub> Power Cycle

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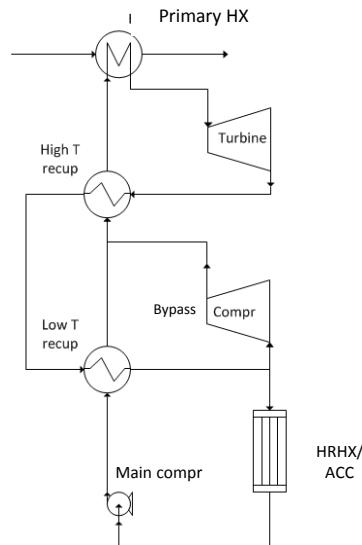
# Objectives of today's discussion

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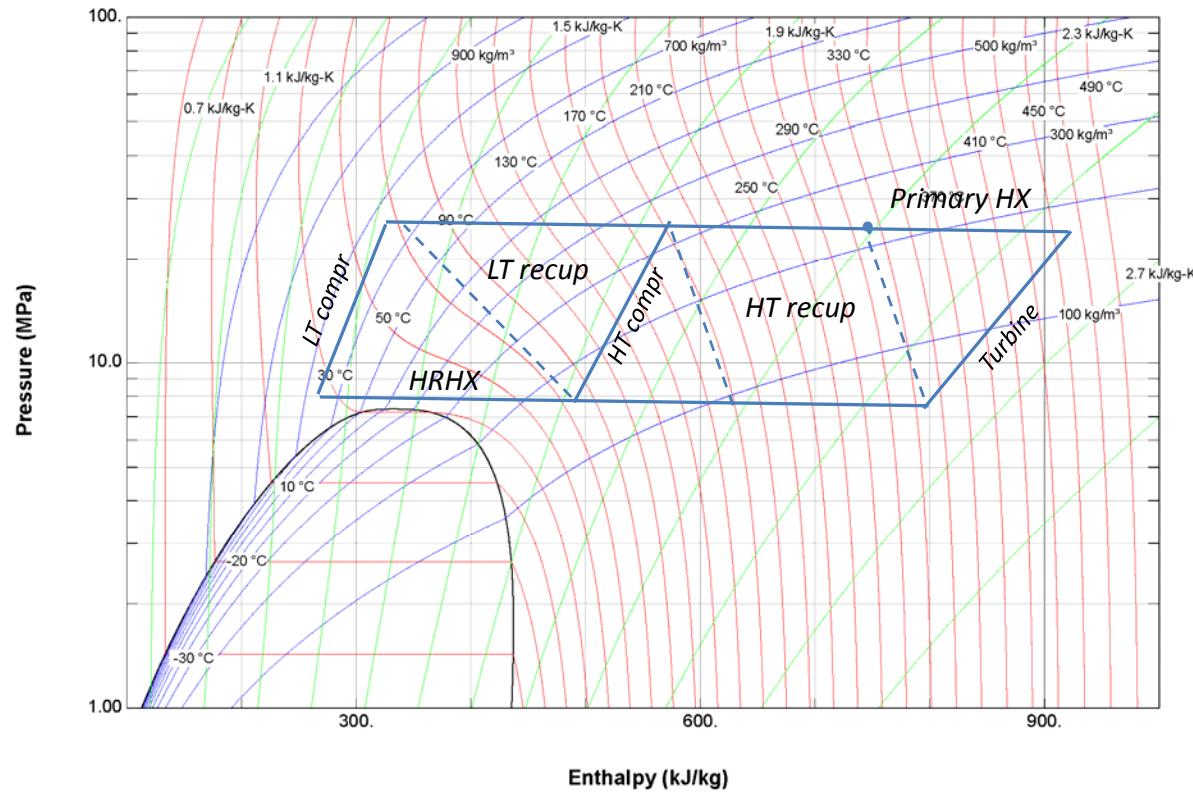


1. Heat exchangers in present study
  - a) Printed circuit HX (HRHX and RHX)
  - b) Finned-tube HX (PHX and ACC)
2. Test system configuration
3. HX modeling in GT-SUITE
4. HX model validation
5. Results and discussion
6. Summary

# sCO<sub>2</sub> cycles are highly dependent upon recuperation



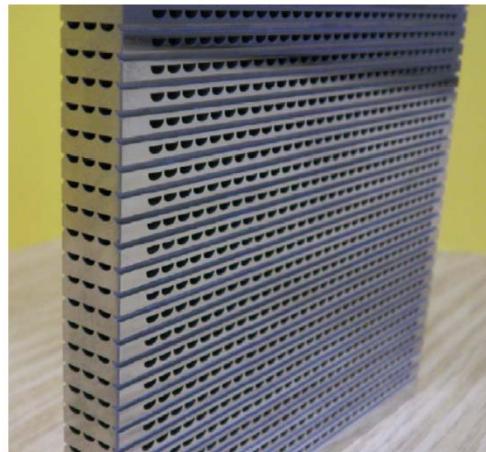
*Low PR = high turbine outlet temperature, large amount of unused enthalpy*



# Printed Circuit Heat Exchangers (PCHE)



- Recuperators (RHX): This high degree of recuperation requires high conductance (UA) and high operating pressures demand for advanced compact HX technology.
- Used as heat rejection HX (HRHX) in  $s\text{CO}_2$  cycles for direct water cooling option.
- Consists of layers of chemically etched metal plates in a diffusion-bonded assembly.
- RHX and HRHX models are validated for transient and steady-state performance against measured data from EPS100 testing.



From Southall, D., 2009, "Diffusion Bonding in Compact Heat Exchangers,"  *$s\text{CO}_2$  Power Cycle Symposium*.

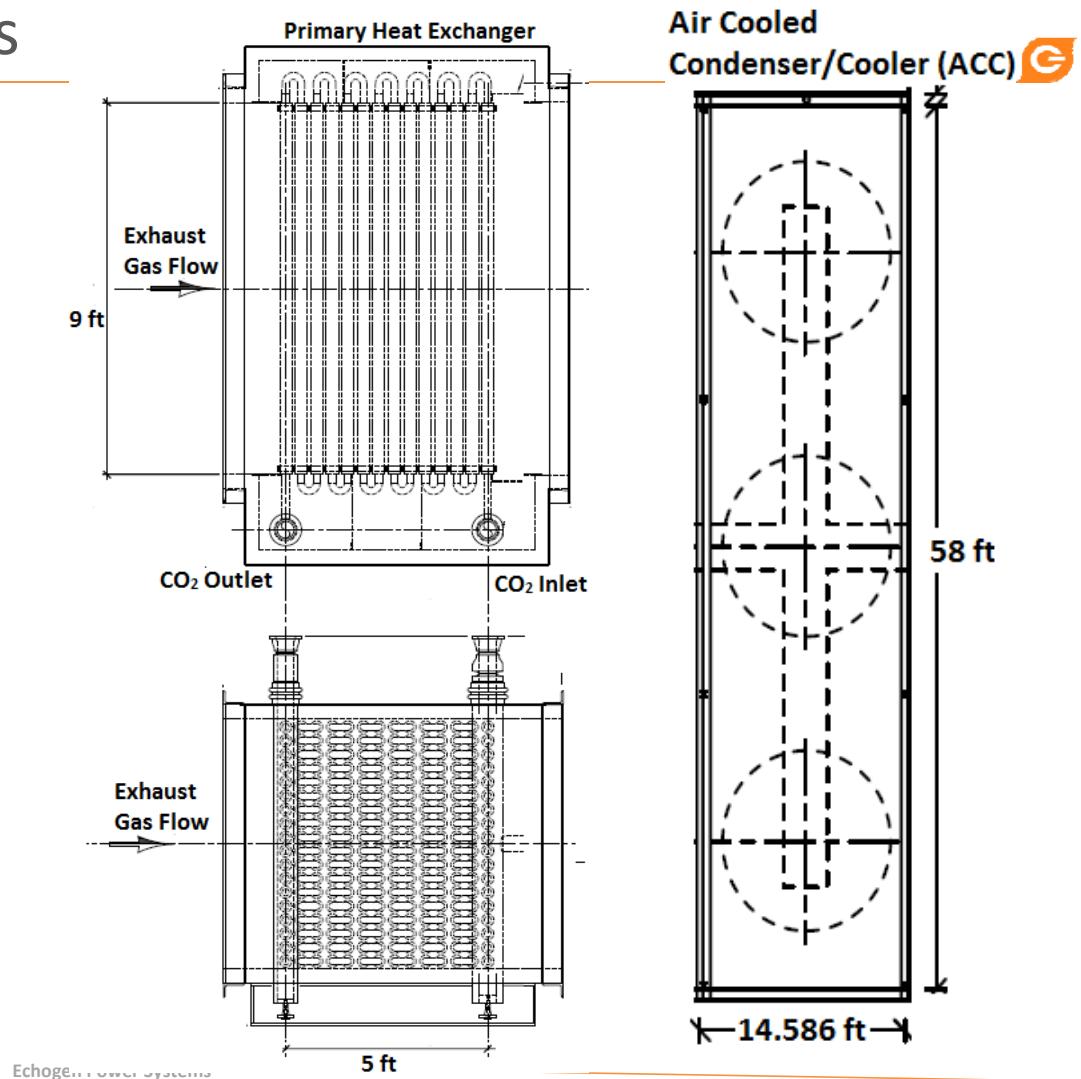


>15MW  
>300m<sup>2</sup> heat transfer area  
~13000kg  
Core ~ 1.5 x 1.5 x 0.5 m

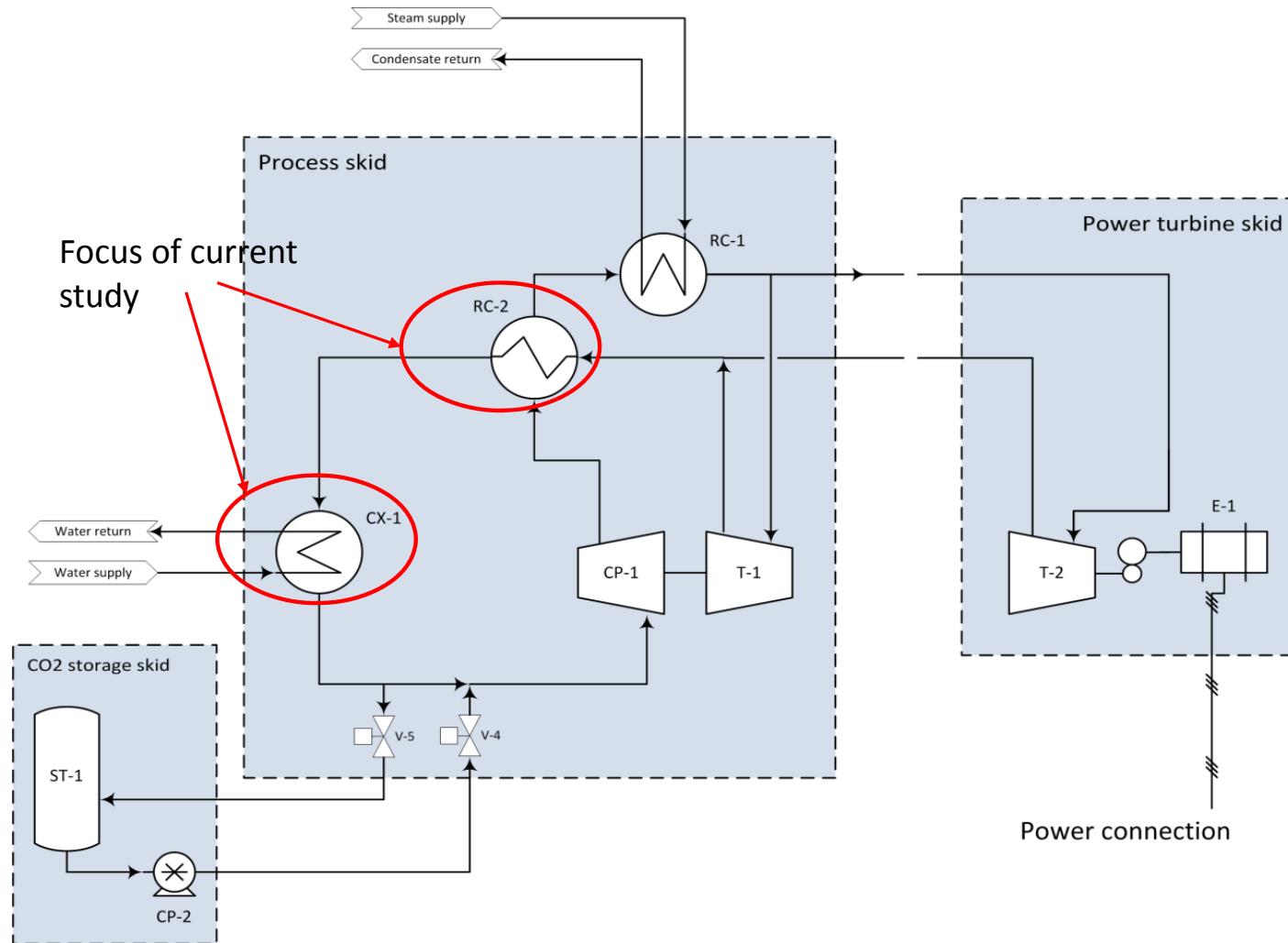
Comparable S&T:  
>850m<sup>2</sup>  
~50000kg  
Shell ~ 1.2m diameter x 12m length

# Finned-Tube Heat Exchangers

- Primary heat exchangers (PHX): In gas turbine exhaust heat recovery of sCO<sub>2</sub> power cycles, finned-tube HX are used for exhaust gas-to-CO<sub>2</sub> (heat source) heat transfer.
- Air cooled condenser/cooler (ACC): In sCO<sub>2</sub> power cycle finned-tube HX are used for CO<sub>2</sub>-to-air (heat sink) heat transfer which also allow for complete water-free operation.
- Consists of series of tube banks with circular louvered fins brazed on surface of each tube.
- As no test data was available from testing of these heat exchangers in sCO<sub>2</sub> applications, they were modeled based on manufacturer-supplied design point data.



# Test System: 7.3MWe net power sCO<sub>2</sub> cycle

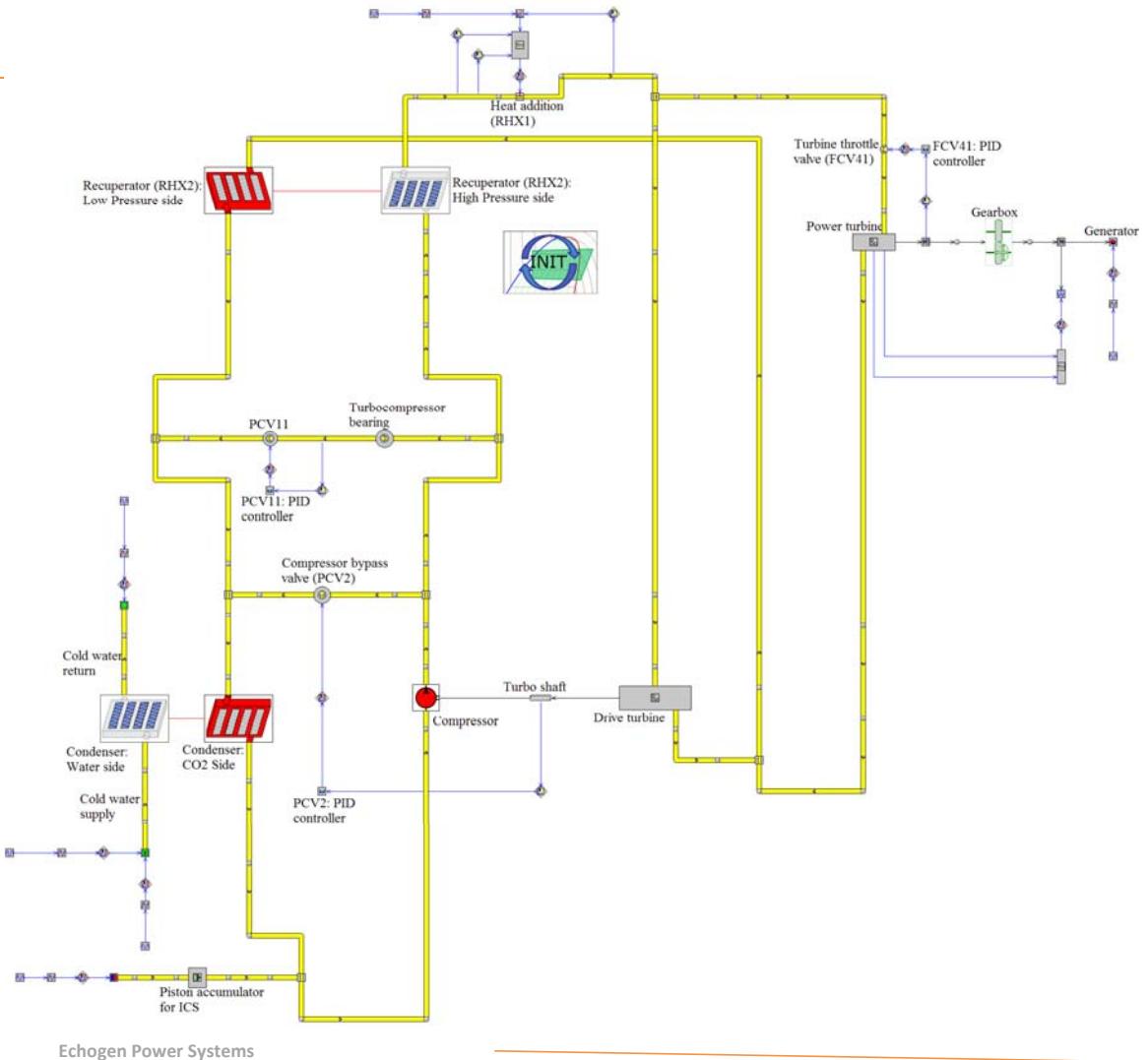


EPS100 – commercial sCO<sub>2</sub> power cycle

- 350 hours of testing
- 3.1MWe max output power
- Numerous transient events (planned & otherwise)

# Modeling Platform

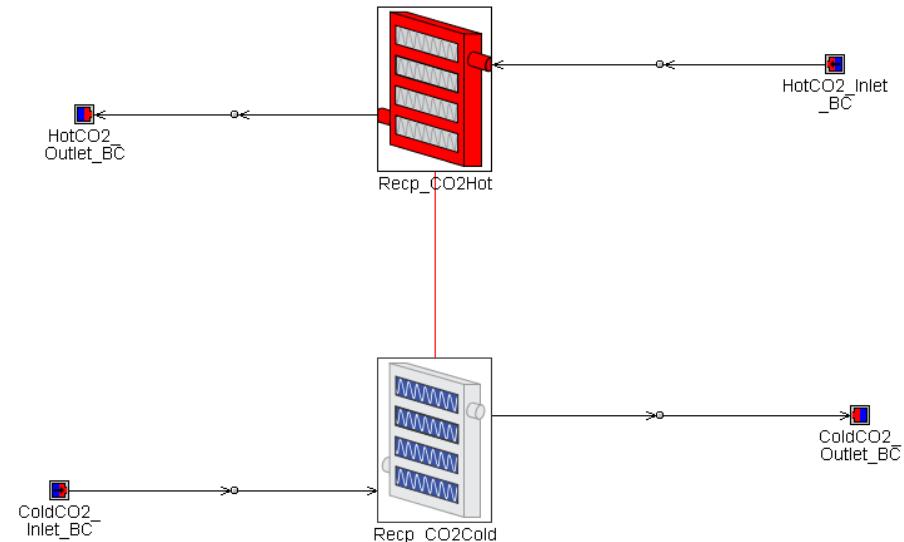
- GT-SUITE 1D engineering system simulation software was used
- Component templates can be GT supplied or user defined templates or Fortran models
- Individual component models were first developed and validated against test data
- Described in ASME GT2017-63279



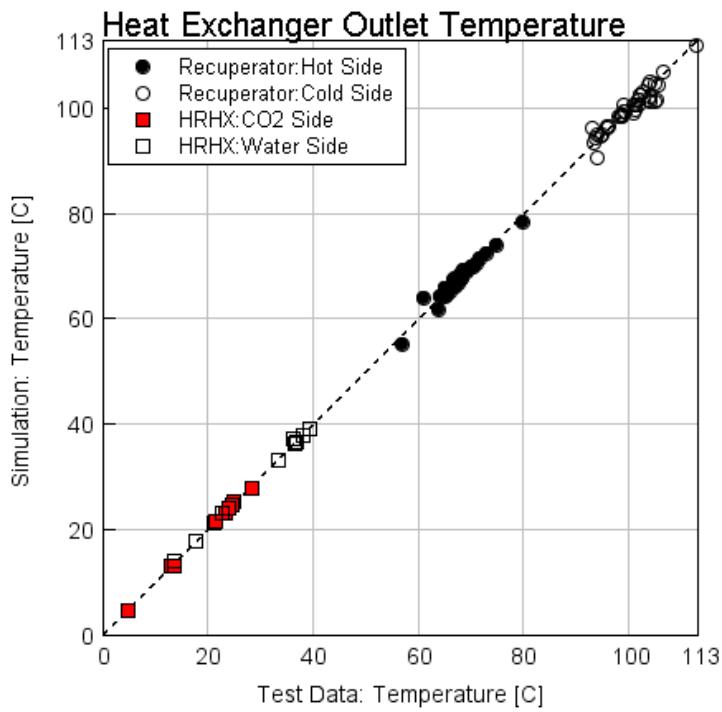
# PCHE Model: RHX and HRHX



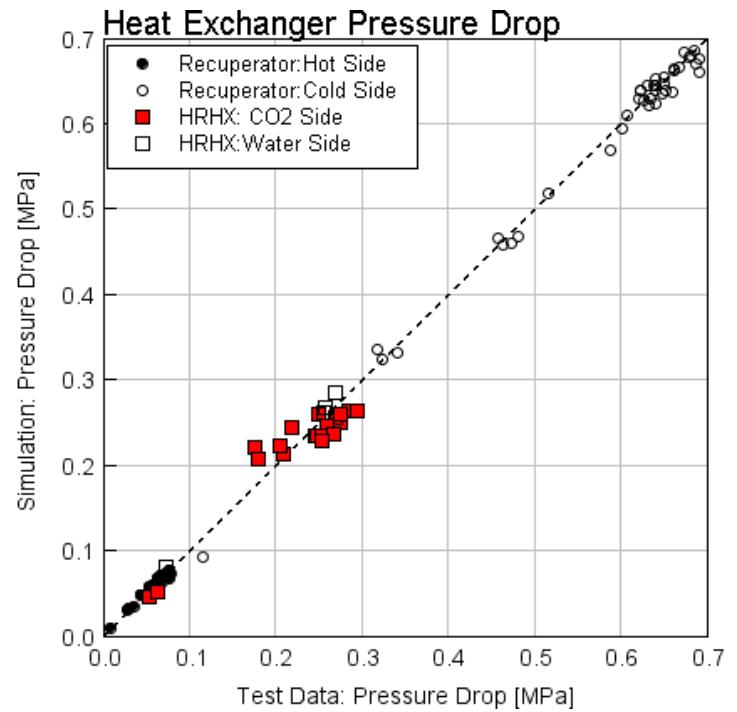
- Counterflow model based on Plate & Frame heat exchanger template
- Inlet temperatures, pressures and flows are imposed boundary conditions
- Heat transferred (hot and cold sides independently) and hot/cold side pressure drops are outputs
- HTC and dP models include calibration to selected steady-state data points
- RHX was modeled as single-phase heat transfer where as the HRHX is modeled as two-phase heat transfer



# HRHX and RHX Model Validation



Recuperator and HRHX outlet temperatures from validation simulation

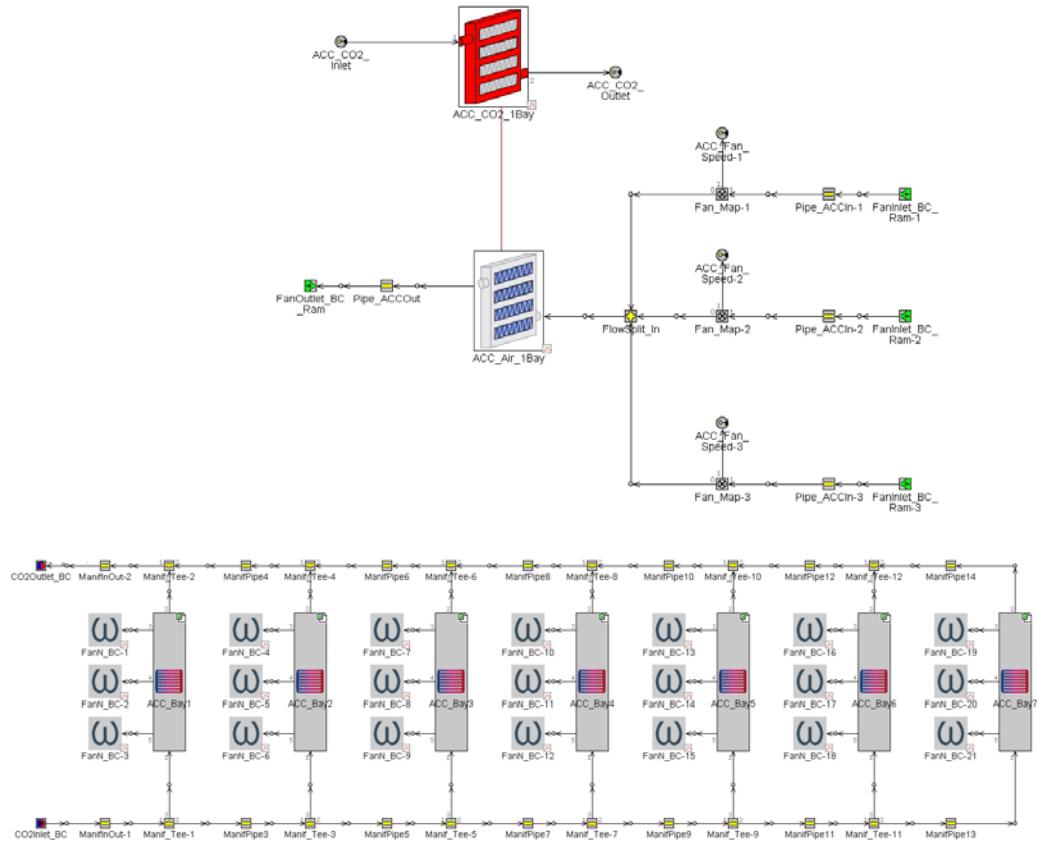


Recuperator and HRHX pressure drops from validation simulation

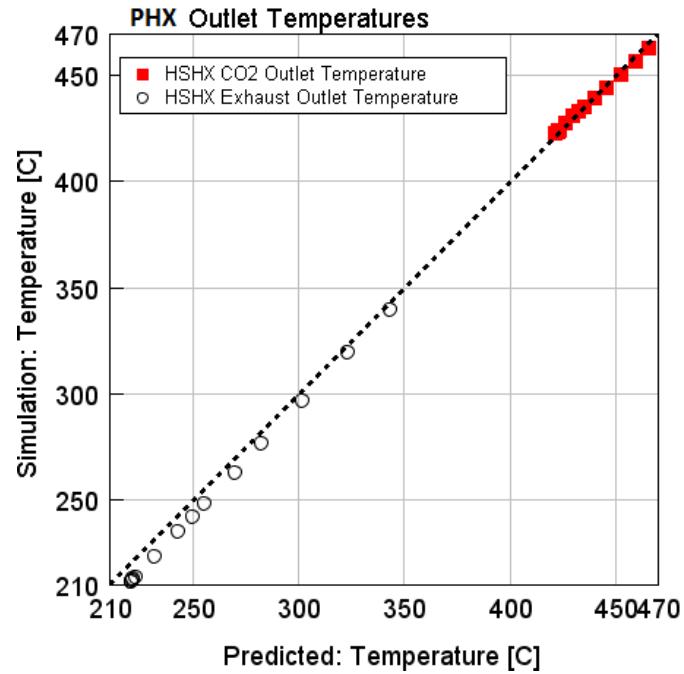
# Finned-tube Model: PHX and ACC



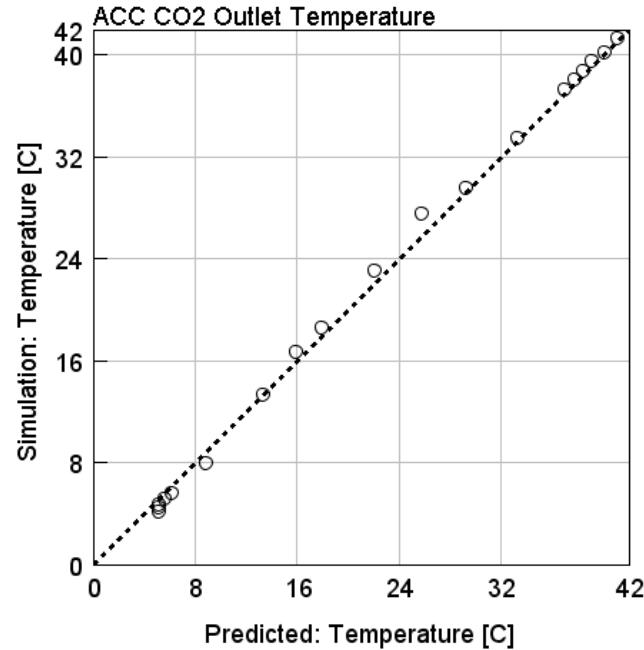
- Crossflow model based on built-in fin-tube heat exchanger template
- Manufacturer-provided design point data was used for calibration
- PHX is a single tube-bundle heat exchanger unit and modeled as single-phase heat transfer with CO<sub>2</sub> on tube side and gas turbine exhaust on fin side.
- ACC has 7-bays with each bay consisting of single tube-bundle heat exchanger unit and 3-fans per bay.
- ACC is modeled as two-phase heat transfer with CO<sub>2</sub> on tube side and air on fin side.



# PHX and ACC Model Validation

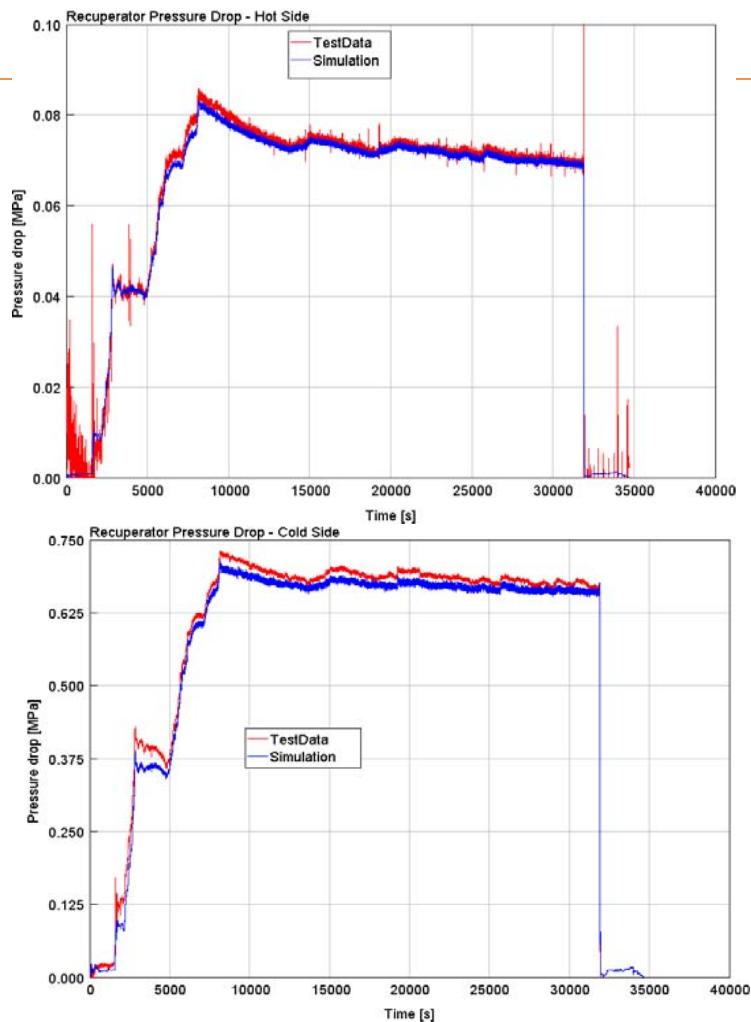
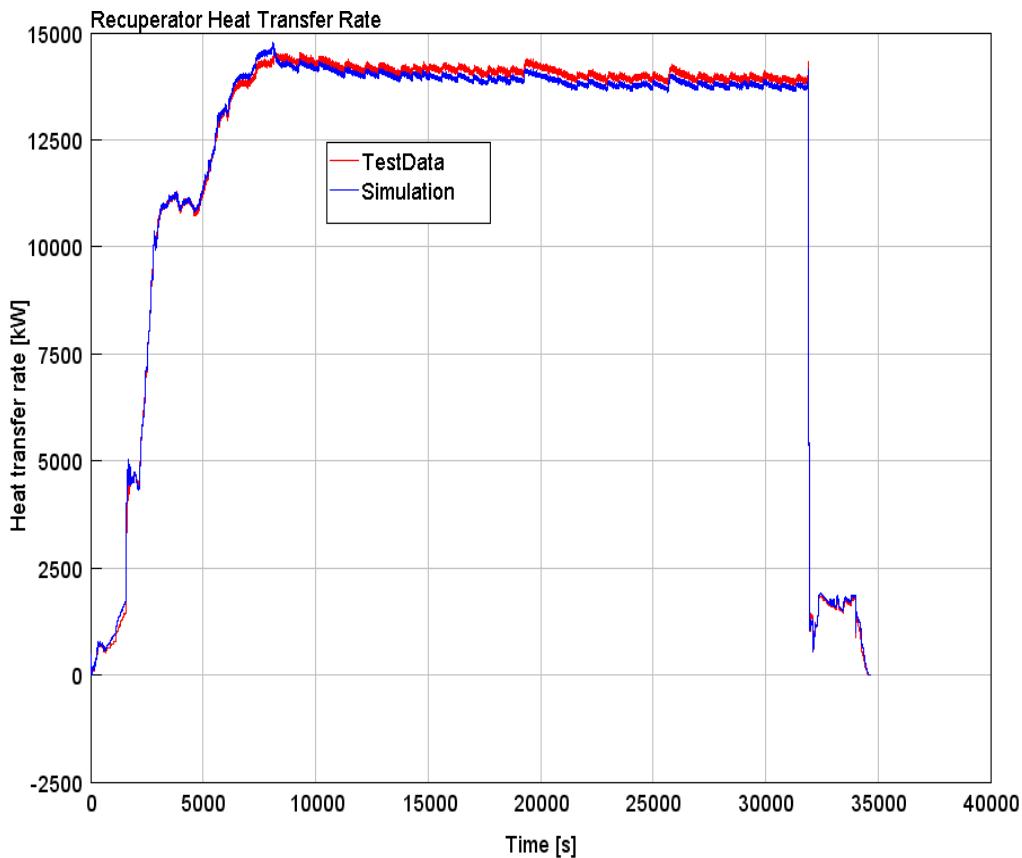


PHX  $\text{CO}_2$  and exhaust outlet temperatures from validation

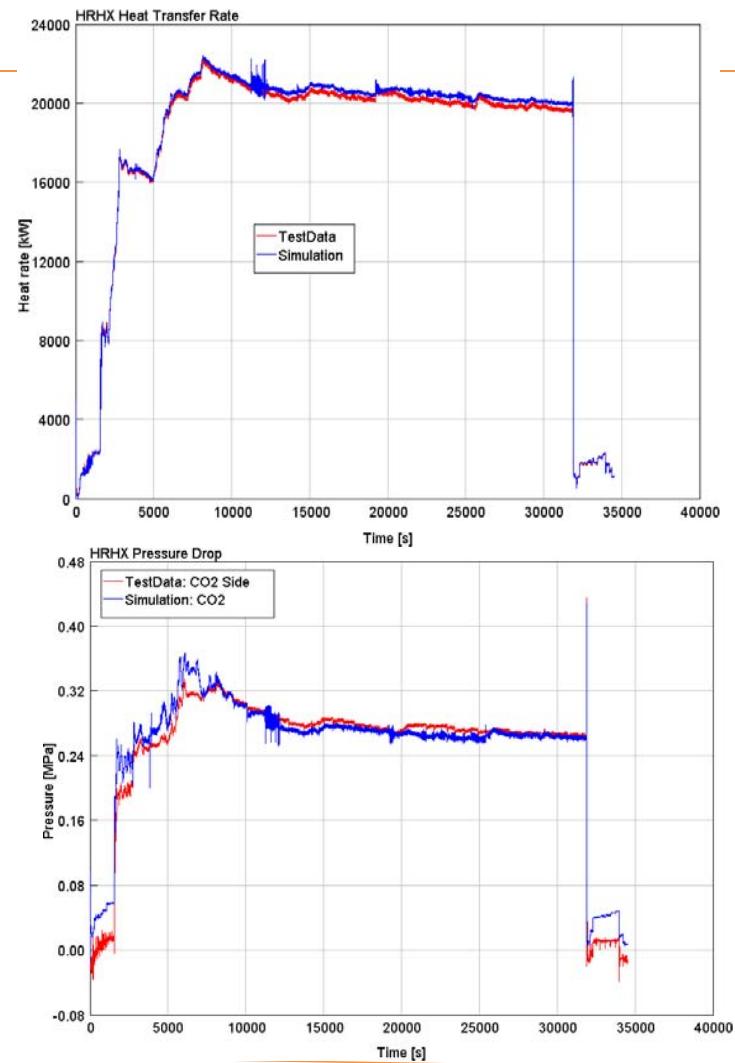
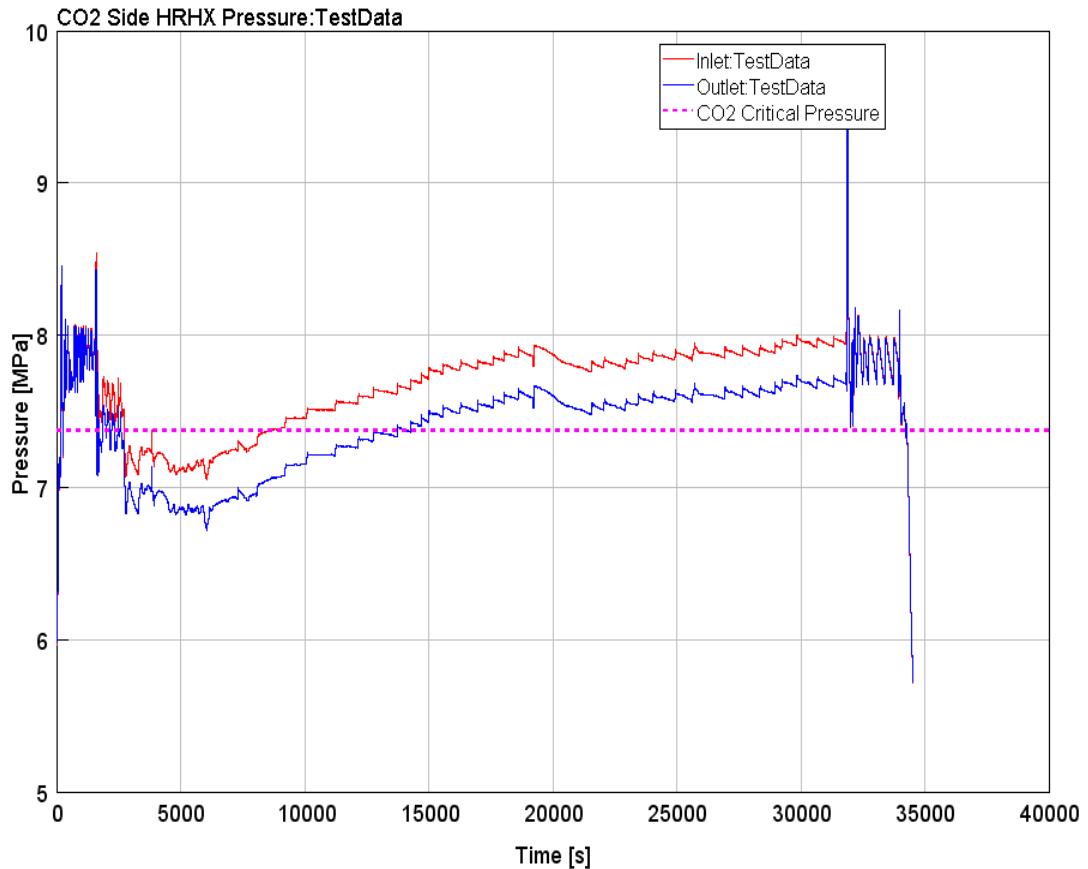


ACC  $\text{CO}_2$  outlet temperatures from validation  
(seven bays simulation)

# Results: Recuperator (PCHE)

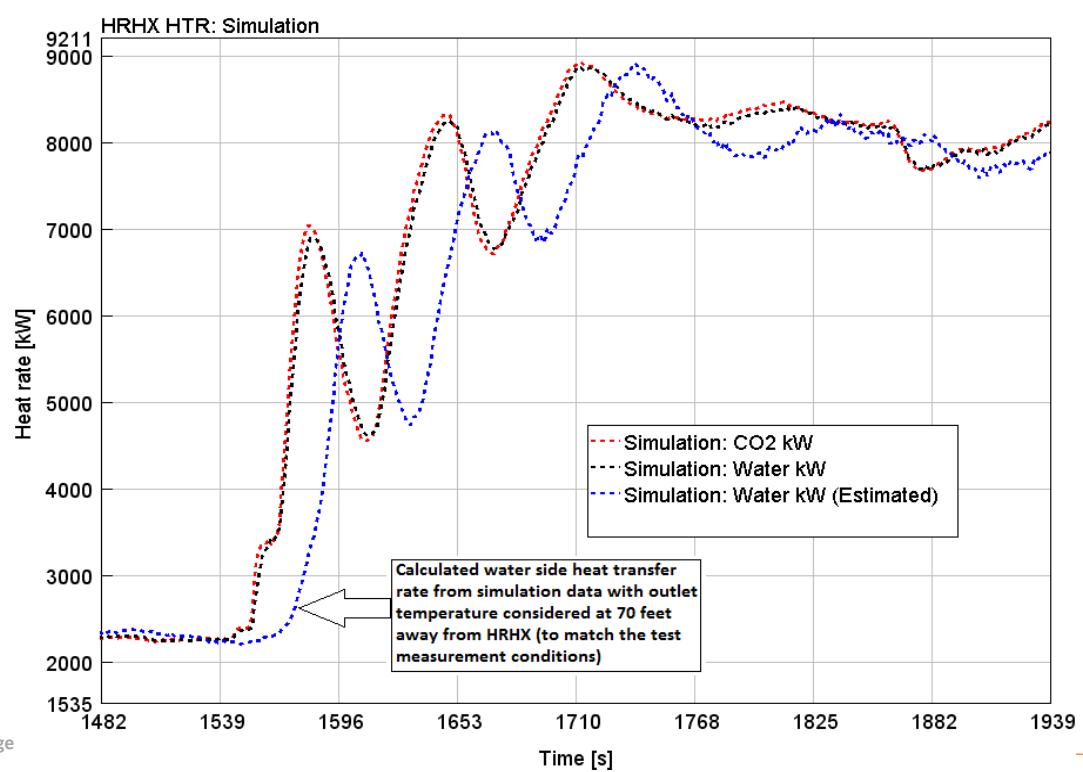
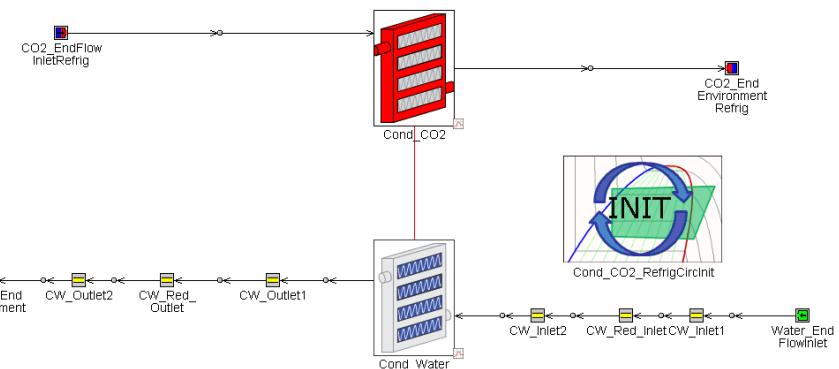
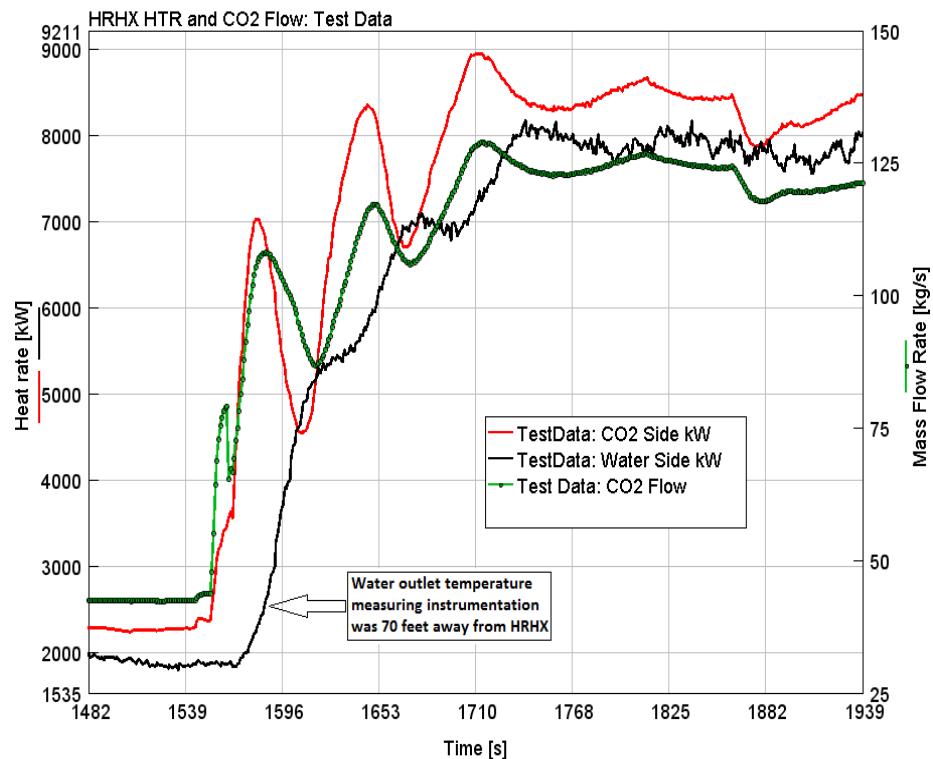


# Results: Heat rejection HX (PCHE)

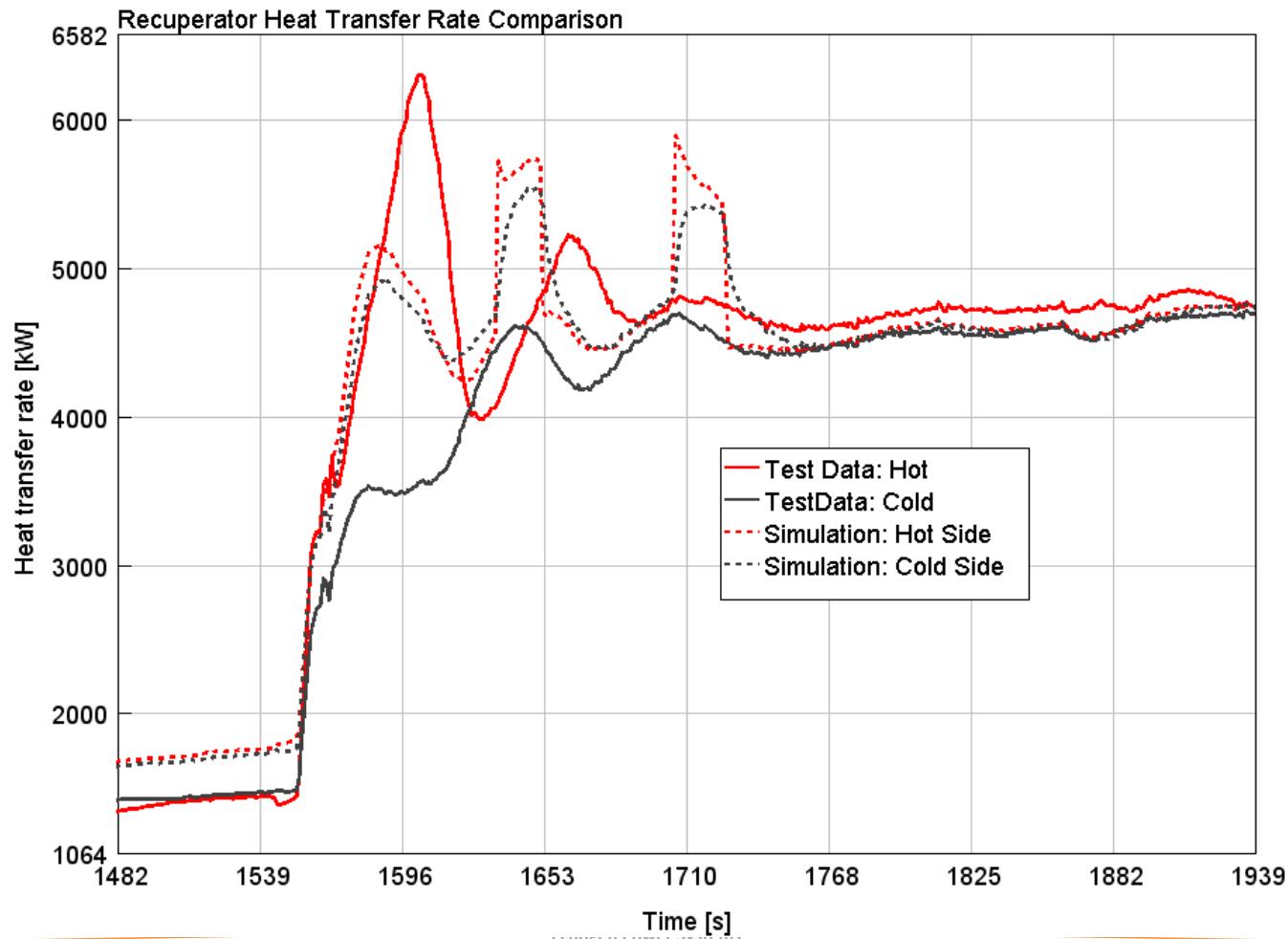


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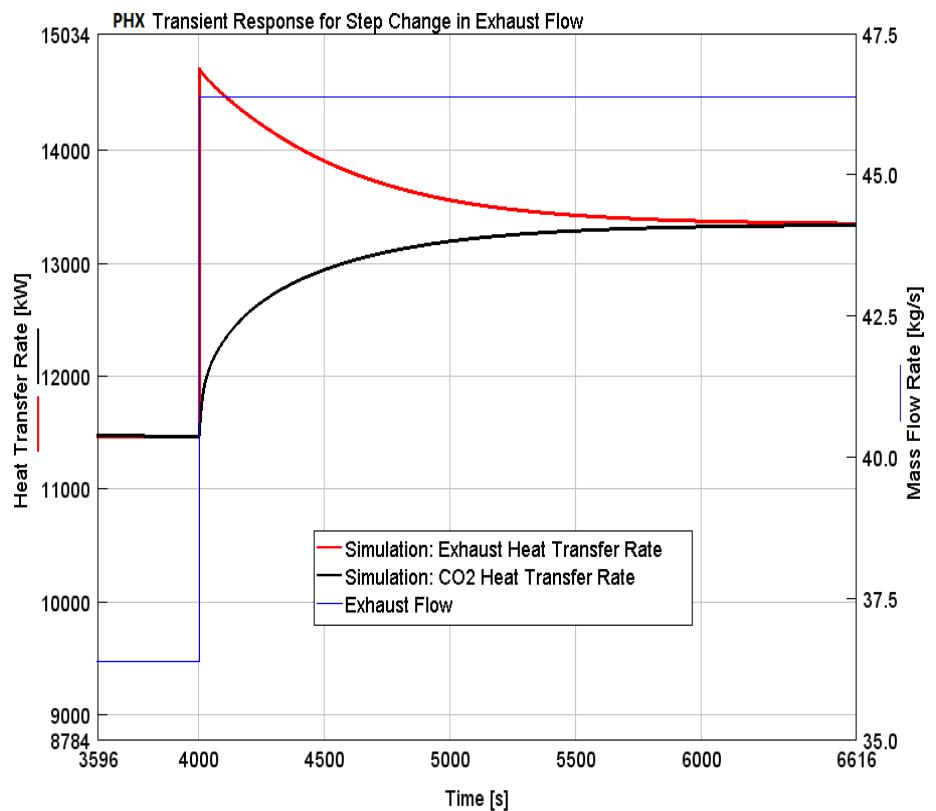
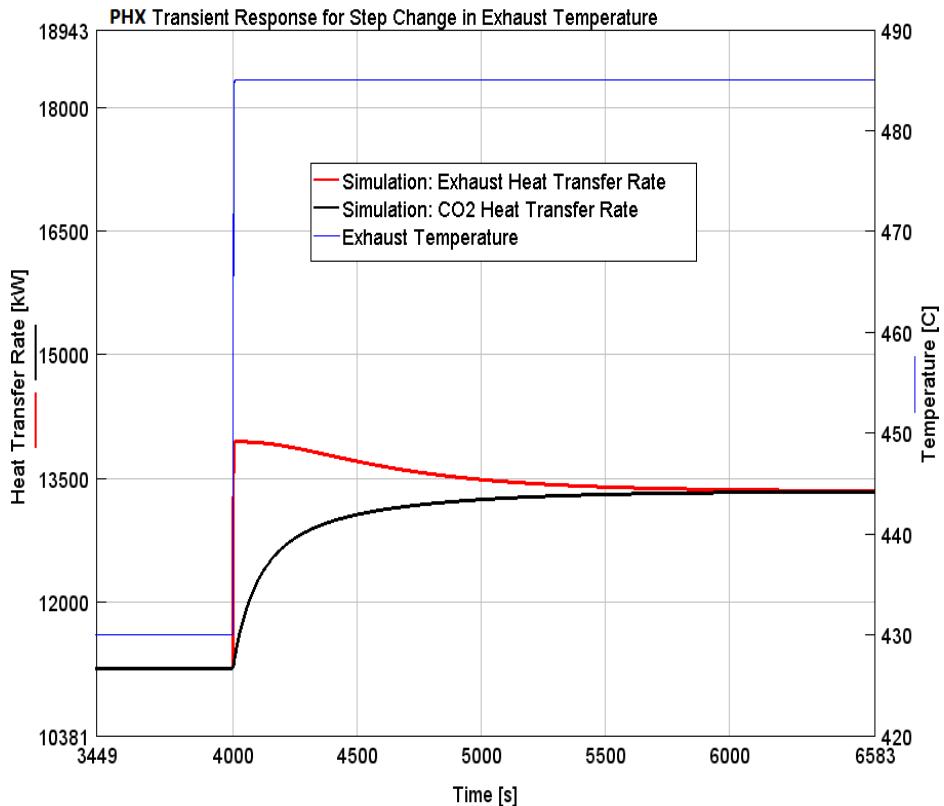
# Results: HRHX transient response



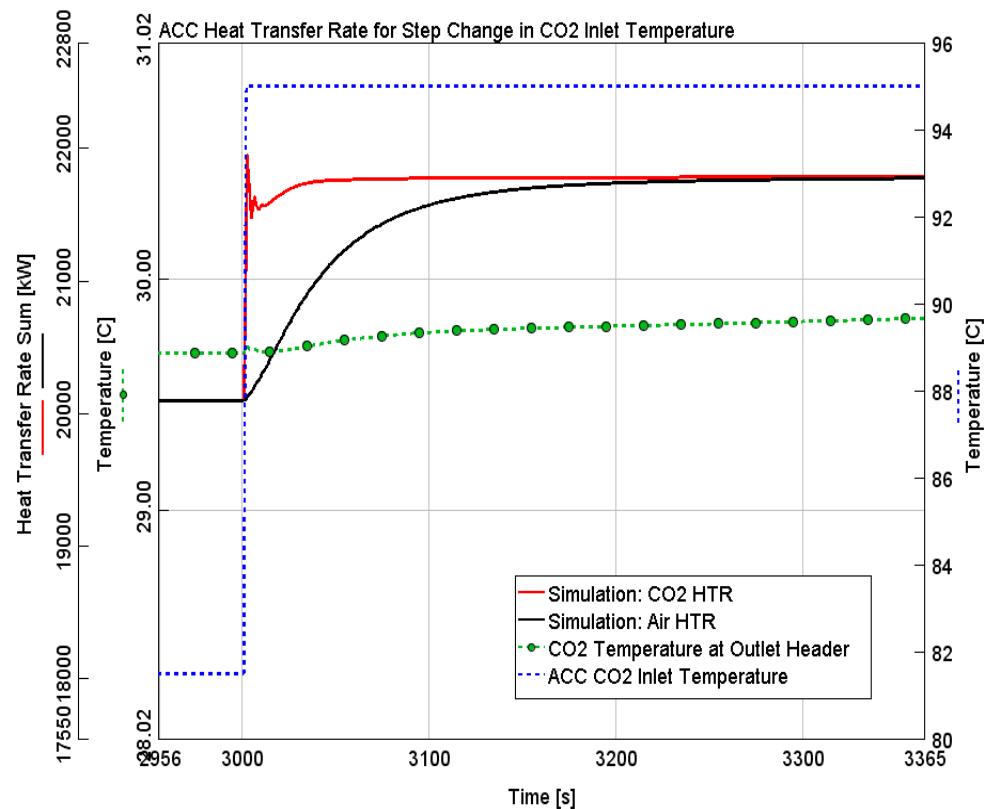
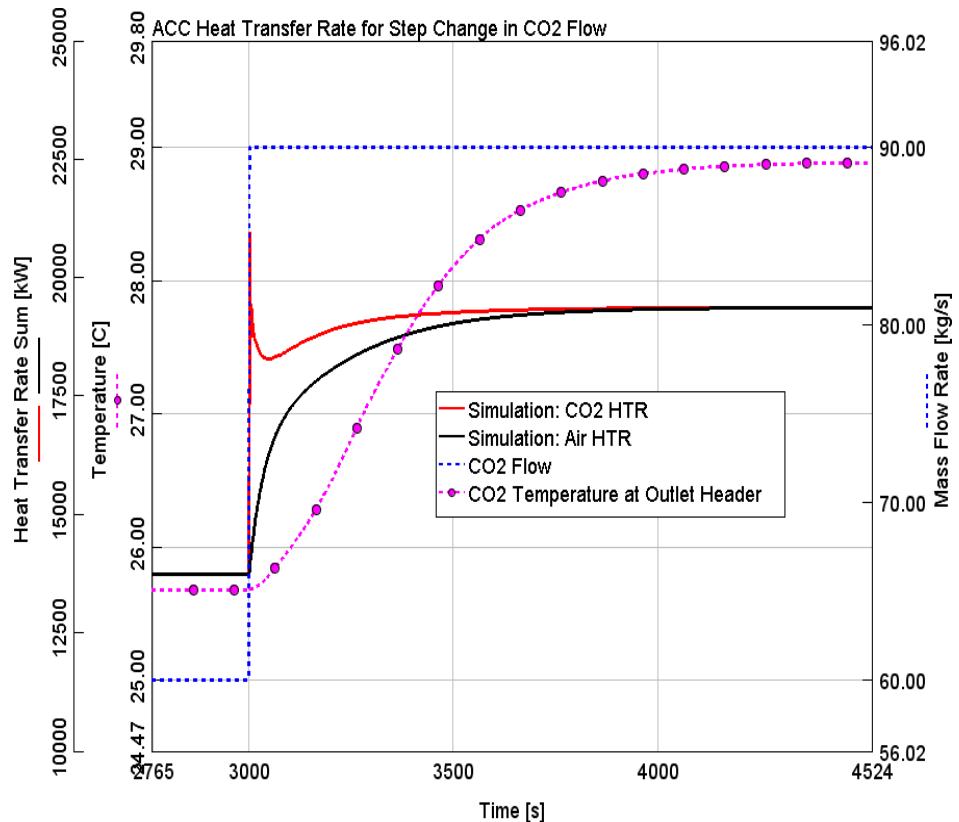
# Results: PCHE transient response - RHX



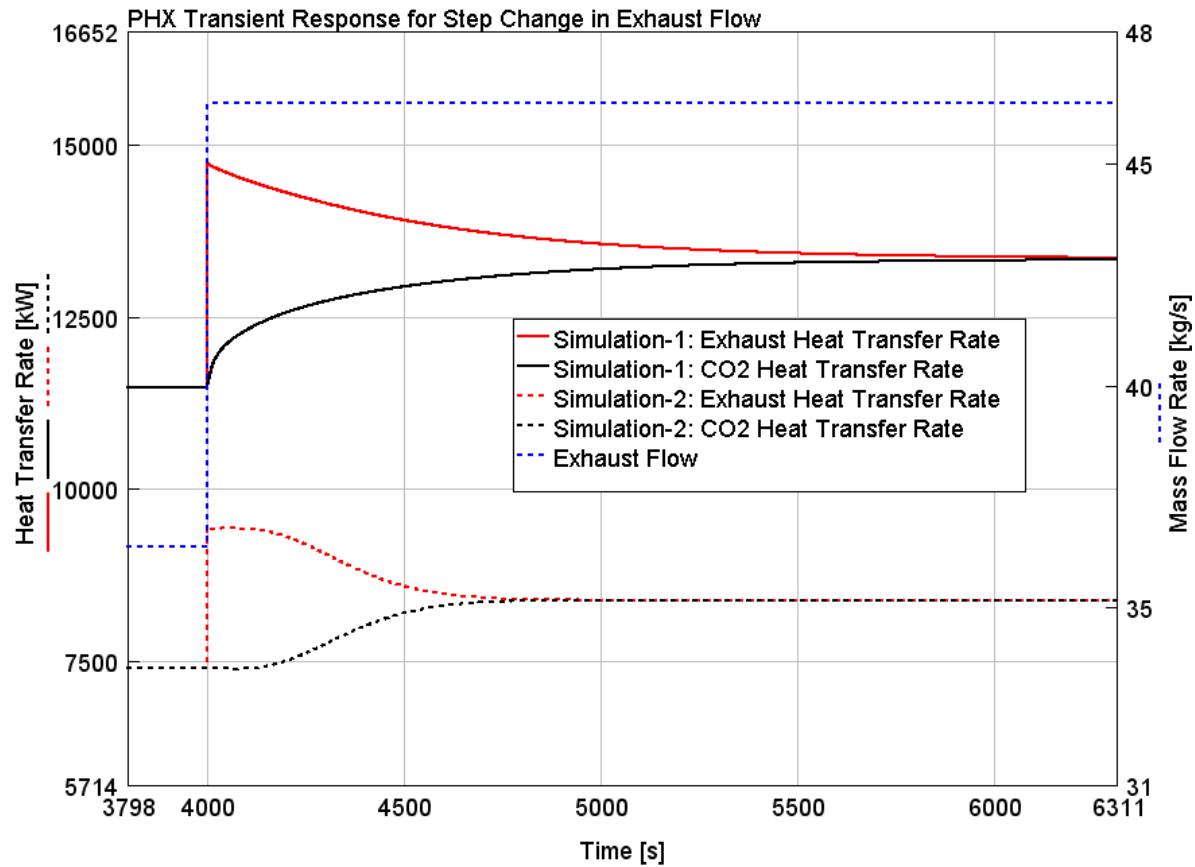
# Results: Fin-tube transient response – PHX



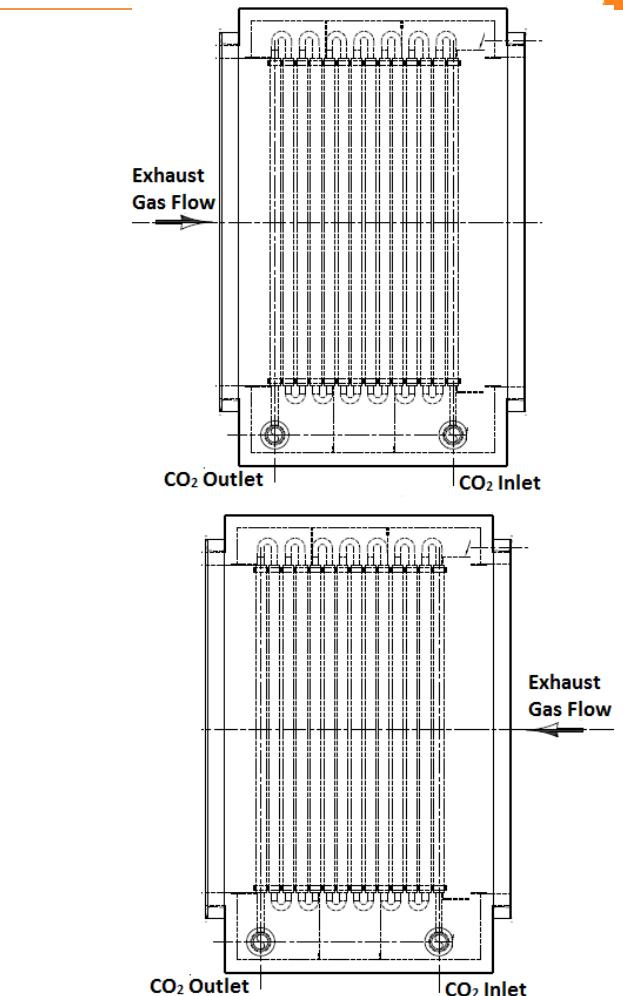
# Results: Fin-tube transient response – ACC



# Results: PHX transient response – Exhaust flow direction



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# Summary

- GT-SUITE 1D system simulation code was used to study the transient and steady state response of PCHE and Fin-tube HX
- The quasi-steady-state as well as transient heat transfer behavior of the heat exchangers, and the single-phase and two-phase pressure drop behavior, can be modeled with reasonable accuracy. Good agreement between transient simulation results and test data was observed.
- Due to highly compact nature of PCHEs, the transient response time for a fluid is very short for any changes in inlet conditions on other fluid side, while the finned-tube heat exchangers have a transient response time approximately two orders of magnitude longer.
- In an sCO<sub>2</sub> power cycle field installation, the finned-tube heat exchangers will define the system time constants, which will also define the plant control system requirements.

$$mc \frac{dT}{dt} = UA (\Delta T_m - \Delta T_{m,ss})$$

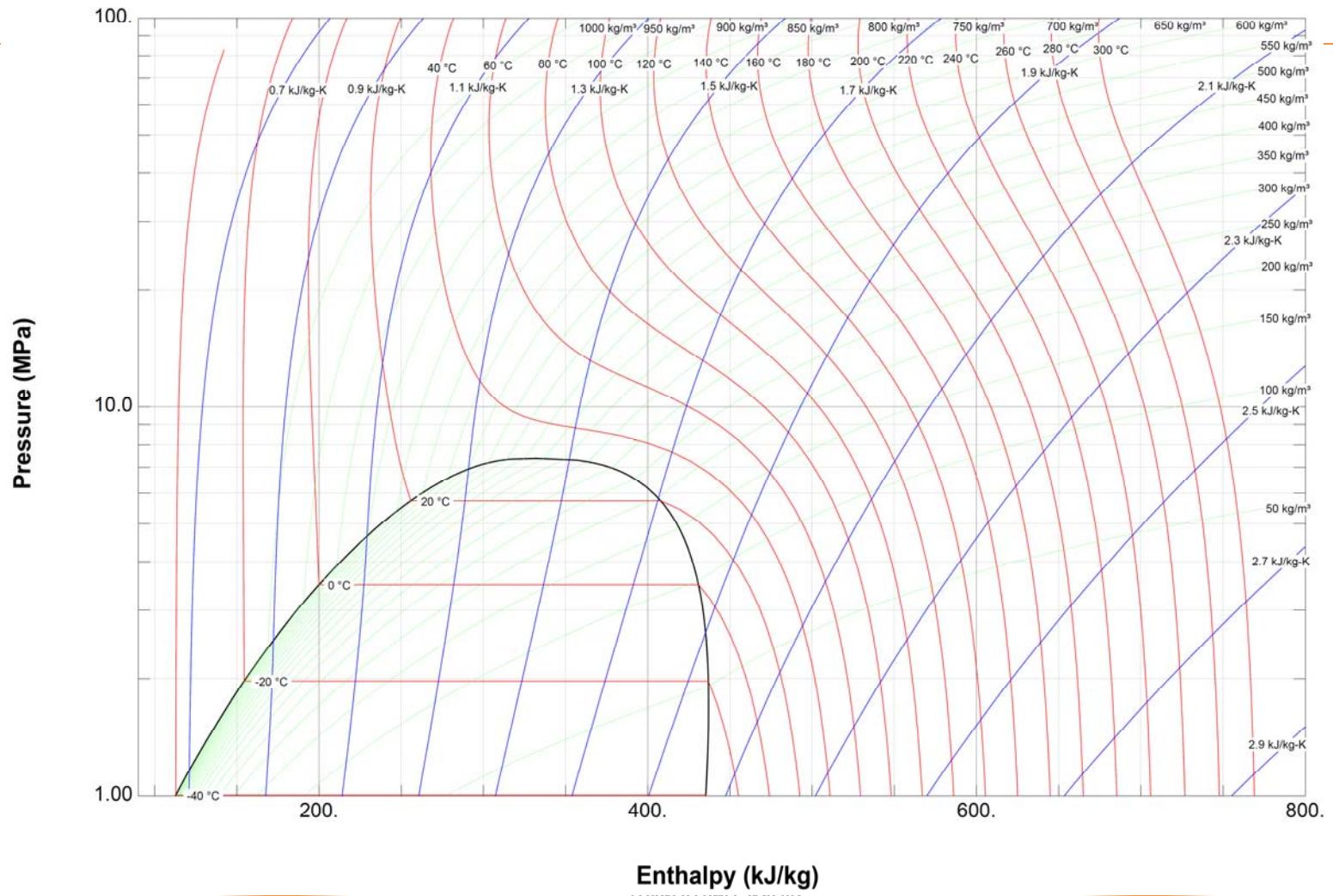
m = mass of HX

c = specific heat of HX material

Characteristic time ( $\tau$ ) =  $\frac{mc}{UA}$

	HRHX (PCHE)	Recuperator (PCHE)	PHX (Finned-tube HE)	ACC (Finned-tube HE)
Mass (kg)	14890	3470	54431	46274 (One Bay)
UA (kW/K)	4811	201	196	402
$\tau$ (seconds)	1.5	9	139	60

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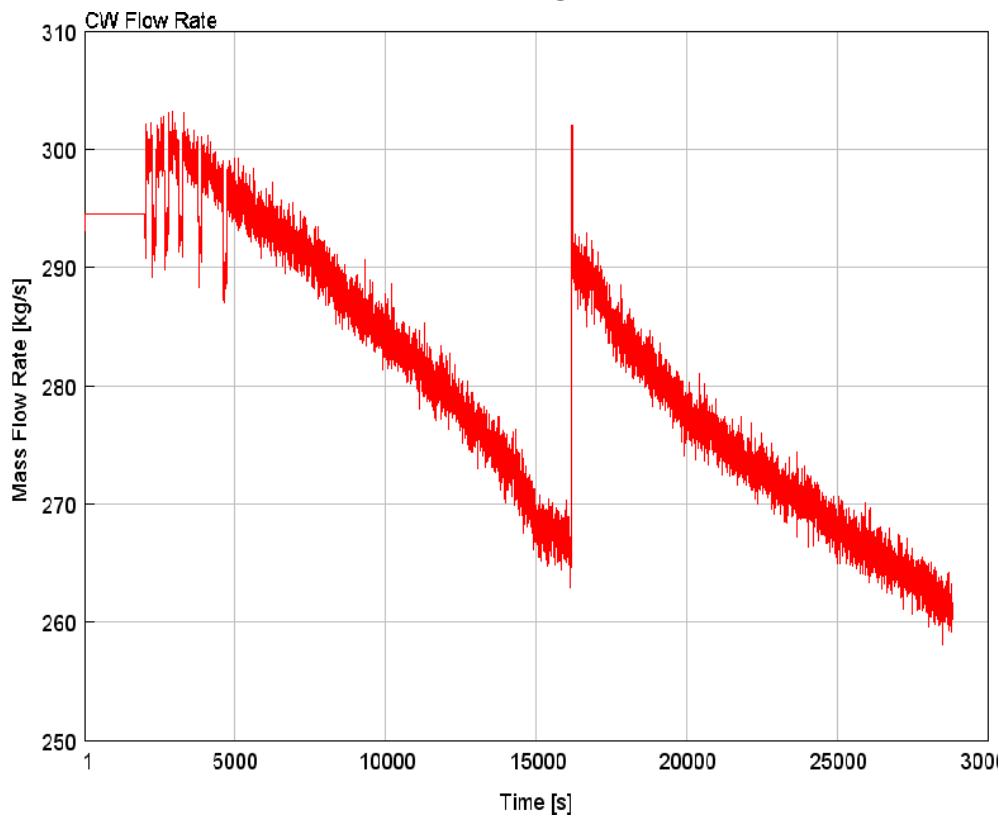


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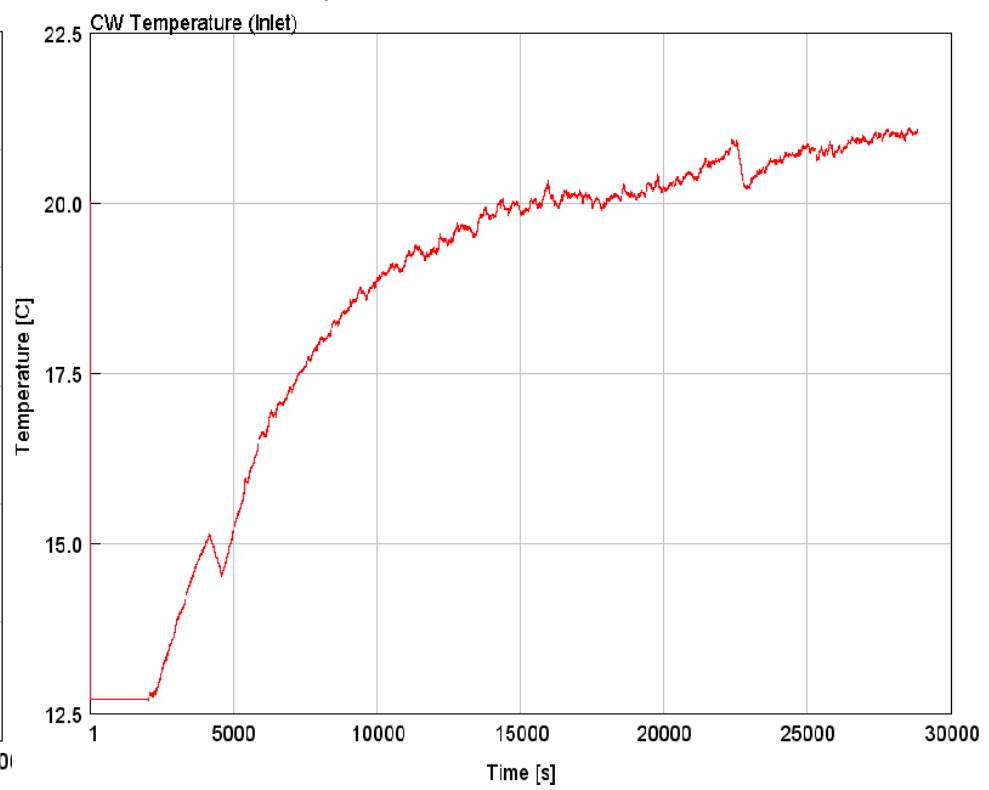
# System Model: Boundary Conditions



Cold water flowrate (kg/s)



Cold water temperature (°C)



# Boundary conditions

