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**GLOBAL KIER**  
TECHNOLOGIES



# Preliminary Power Generating Operation of the Supercritical Carbon Dioxide Power Cycle Experimental Test Loop with a Turbo-generator



Tuesday, March 27, 2018

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Daejeon, South Korea

[102] The 6th International Supercritical CO<sub>2</sub> Power Cycles Symposium  
March 27 - 29, 2018, Pittsburgh, Pennsylvania

# Outline



- 01 Introduction of KIER's S-CO<sub>2</sub> R&D
- 02 tens of kWe Test Loop & Axial Turbo-generator
- 03 500 °C kWe Test Loop & Radial Turbo-generator
- 04 Summary & Future Works

# Chapter 01

## Introduction of KIER's S-CO<sub>2</sub> R&D



The KIER, a global energy innovator, does its best in pursuing its mission to invent world-class energy technologies based on open innovation, life-cycle research quality assurance, participatory and open communication. Therefore the KIER will become the best energy technology R&D institute in the world, contributing to the creation of wealth and improvement of quality of life for the people.

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# S-CO<sub>2</sub> Power Cycle Systems in the world



2005      2009      2010      2011      2012      2013      2014      2015      2016      2017      2018      2019+

**SNL (250kWe)**  
477°C/105bar/15kWe

**'05~'10** Single Comp.    **'10** 1 TAC    **'11** RCBC/2 TAC/GFB    **'13** 480°C/105bar/15kWe    **'14** 100kWh    **'16** 500kWh

**Naval Nuclear Lab. (Bechtel) (100kWe)**  
282°C/141bar/40kWe

**'09~'11** 1 TC, 1 TG, GFB    **'14** 282°C/141bar/40kWe    **'16** 226°C/136bar/40kWe

**TIT (10kWe)**  
260°C/105bar/0.11kWe

**'07~'12** 1 TAC, Gas bearing, 260°C/105bar/0.11kWe

- \*RCBC : Recompression Closed Brayton Cycle
- \*TAC : Turbine-Alternator-Compressor
- \*TC : Turbo-Compressor
- \*TG : Turbo-Generator
- \*GFB : Gas Foil Bearing
- \*ACB : Angular Contact Ball Bearing
- \*TPB : Tilting-Pad Bearing
- \*DGS : Dry Gas Seal

**KIER (~ hundreds of kWe)**  
@sub-kWe, 170°C/101bar/0.67kWe (turbine)  
@tens of kWe, 205°C/100bar/11kWe (turbine)  
@kWe, 401°C/100bar/287kWe (turbine)

**'13~'14** 10kWe, TAC, GFB, 83°C/85bar/30,000rpm    **'14~'16** 1kWe TG (Radial), ACB, 170°C/101bar/670kWe (turbine)    **'15~'17** 60kWe TG (Axial), TPB, 205°C/100bar/11kWe (turbine)    **'16~'17** 5kWe TG (Radial), ACB, 401°C/100bar/287kWe (turbine)

**TODAY**

Hundreds of kWe  
/500°C/130bar  
→  
2 turbine  
1 compressor

**SwRI/GE (1MWe)**  
715°C/250bar/under commissioning

**'12~'17** 1 Turbo-expander, DGS, TPB, 1 Compressor, 700°C/250bar/under commissioning

**U.S. 10MWe Pilot Plant Project Leading Group**

1-2 phase: 500°C  
3 phase: 700°C

**Echogen (8MWe)**  
275°C/ ? bar/2.35MWe

**'14** 1 Turbo-pump (hermetic), 1 Power Turbine (DGS, TPB), TIT design 485°C/ Test 275°C/2.35MWe

# KIER S-CO<sub>2</sub> Project Overview



- 2013-2014 : 10 kWe test loop, 200°C (Preliminary Operation)
- 2014-2016 : sub-kWe test loop, 200°C (Power Generation)

- 2015-2019 : 5 years project
- Vision (long-term) : Substitution of the steam Rankine cycle system (particularly in fossil fuel application)
- Mid-term goal : WHR market**
- Budget : \$2M/yr
- Project Leader : Young-Jin Baik, Ph.D.  
Chief of Thermal Energy Systems Laboratory at KIER
- 2015-2017 : tens of kWe test loop, 200°C (Power Generation) Today Presentation
- 2016-2017 : kWe test loop, 500°C (Power Generation) Today Presentation
- 2016-2019 : **hundreds of kWe test loop, 500°C** (under construction)

# KIER S-CO<sub>2</sub> Test loops



	10 kWe-class (2013-2014)	Sub-kWe-class (2014-2016)	Tens of kWe-class (2015-2017)	kWe-class (2016-2017)	Hundreds of kWe-class (2015-2019)
Purpose	Feasibility 200°C/130bar	Power generation 200°C/130bar	Robust Turbo-generator 200°C/130bar	500°C operation <b>500°C/130bar</b>	500°C Full-cycle operation for WHR application 500°C/130bar
Status	Tested @ 30,000RPM Modified to the tens of kWe test loop	<b>670 We</b> power generation Modified to the kWe test loop	<b>11 kWe</b> power generation	<b>287 We</b> power generation	In progress
Cycle type	Simple Un-Recuperated Closed Brayton	Un-recuperated Transcritical	Un-recuperated Transcritical	Recuperated Transcritical	<b>Dual Brayton</b>
Turbomachinery	<b>1 Turbo-Alternator-Com pressor</b>	<b>1 Turbo-generator</b>	<b>1 Turbo-generator</b>	<b>1 Turbo-generator</b>	<b>2 Turbine 1 Compressor</b>
Compressor type	Centrifugal, Shrouded	Positive displacement Pump	Positive displacement Pump	Positive displacement Pump	<b>Centrifugal</b>
Turbine type	Radial, Shrouded	Radial w/ <b>Partial admission nozzle</b>	<b>Axial impulse w/ Partial admission nozzle</b>	Radial w/ Partial admission nozzle	TBD
Bearing	<b>Gas foil journal/thrust</b>	<b>Angular contact ball (Oil lubrication)</b>	<b>Tilting-pad (Oil lubrication)</b>	Angular contact ball (Oil lubrication)	TBD
Seal	Labyrinth	Labyrinth	Carbon Ring type <b>Mechanical Seal</b>	Labyrinth	<b>DGS</b>
Rotational speed (RPM)	70,000	200,000	45,000	120,000	TBD
Heater	LNG fired Thermal Oil Boiler	Immersion electric heater	LNG fired Thermal Oil Boiler	Immersion electric heater	LNG fired flue-gas Heater
Recuperator	none	none	none	PCHE	2 PCHE

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# Chapter **02**

## Tens of kWe Test Loop & Axial Turbo-generator



The KIER, a global energy innovator, does its best in pursuing its mission to invent world-class energy technologies based on open innovation, life-cycle research quality assurance, participatory and open communication. Therefore the KIER will become the best energy technology R&D institute in the world, contributing to the creation of wealth and improvement of quality of life for the people.

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# Technical Issues in S-CO<sub>2</sub> Turbomachinery



1. **Small Size** : 20 cm diameter @ 10 MWe class
2. **High Speed** : 30,000 RPM @ 10 MWe class

lab-scale  
(under MWe)

**Windage Loss**

**01 Fluid Friction Loss**

- High pressure, High density fluid  
**Windage loss at shaft & bearing**  
[SNL reports]
- Limitation of 1 casing TAC Structure**

**Gas Foil Bearing Failure [High Thrust]**

**02 Bearing Failure @ High T/P**

- Gas-foil bearing failure @ high temp.**  
[SNL reports, NASA report]
- High thrust occurs, but very weak**

KIER Strategy

Scale-up is OK!

**Non-hermetic**

Mechanical Seal  
or Labyrinth seal in lab-scale



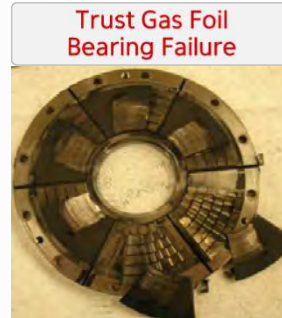
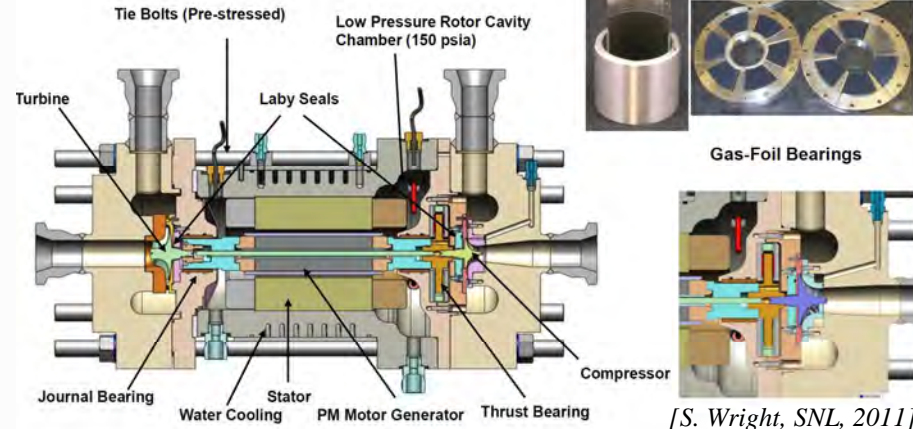
**Oil-lubricated Bearing**

Mature/Robust Commercial Bearing  
High thrust force capacity



**Reduce Rotational Speed !**  
**Partial Admission Nozzle**

KEY Design Strategy



ARL-MR-0749

[NASA] Limitation of Gas Foil Bearing @ high Temp.

A Novel Thermal Management Approach for Radial Foil Air Bearings

Kevin Radil  
NASA Glenn Research Center  
Cleveland, OH

[K. Radil et al., ARL-MR-0749, NASA, 2010]

[T. Conboy et al., Journal of Engineering for Gas Turbines and Power, Vol. 134, 2012]

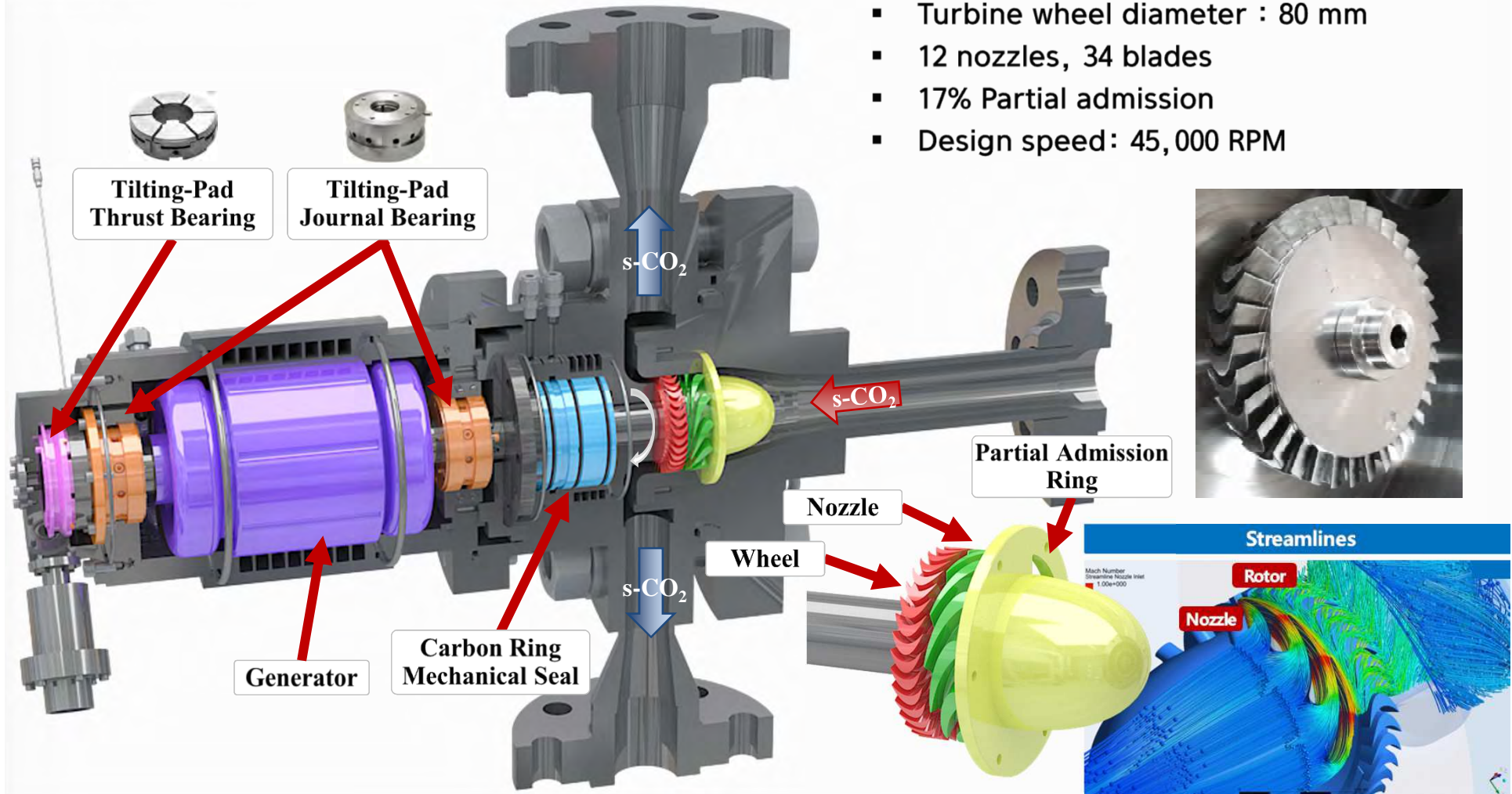
# KIER S-CO<sub>2</sub> Axial Impulse Turbo-Generator



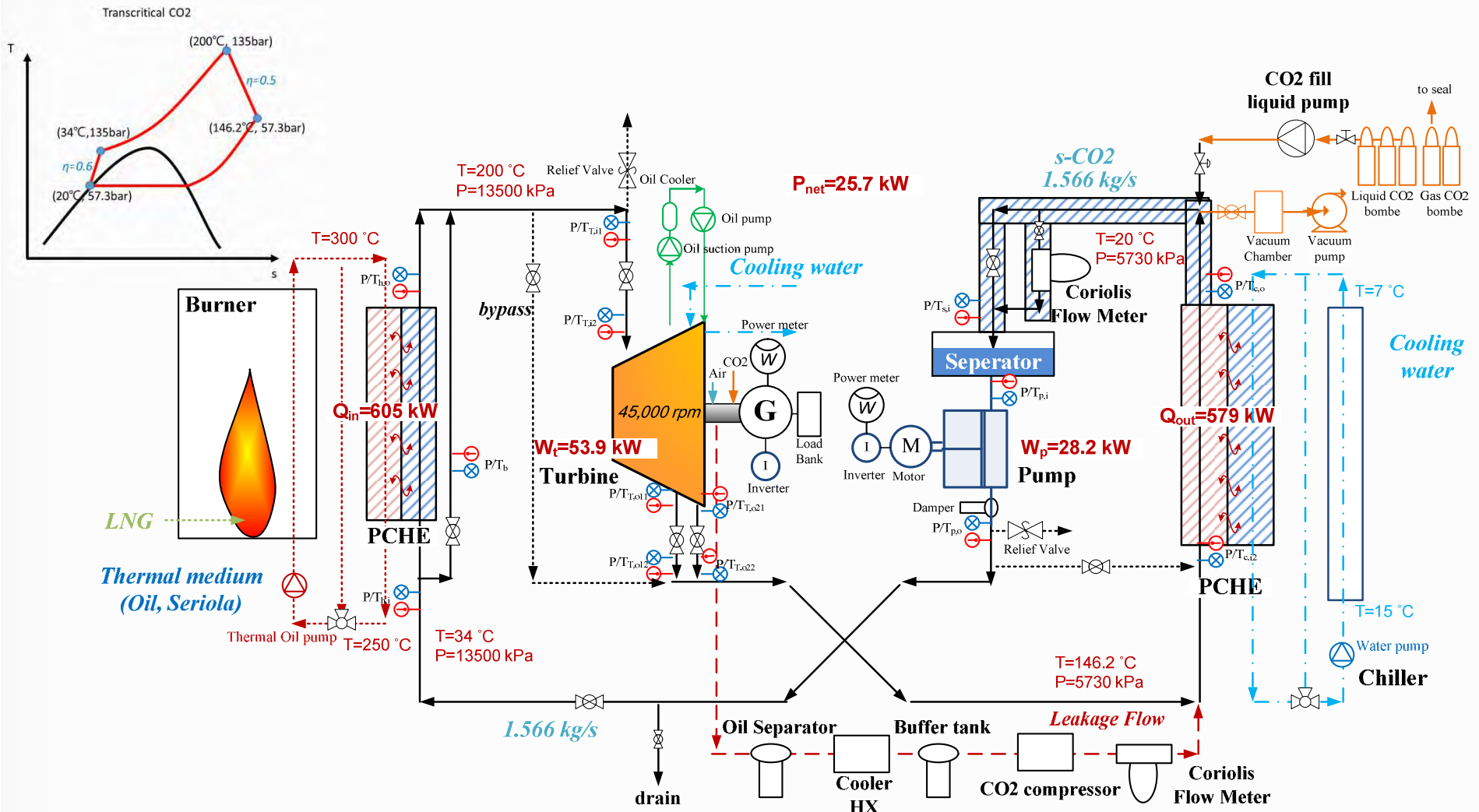
S-CO<sub>2</sub> Axial Impulse Turbo-Generator

Details are shown in 'GT2017-64349'

- Turbine wheel diameter : 80 mm
- 12 nozzles, 34 blades
- 17% Partial admission
- Design speed: 45,000 RPM

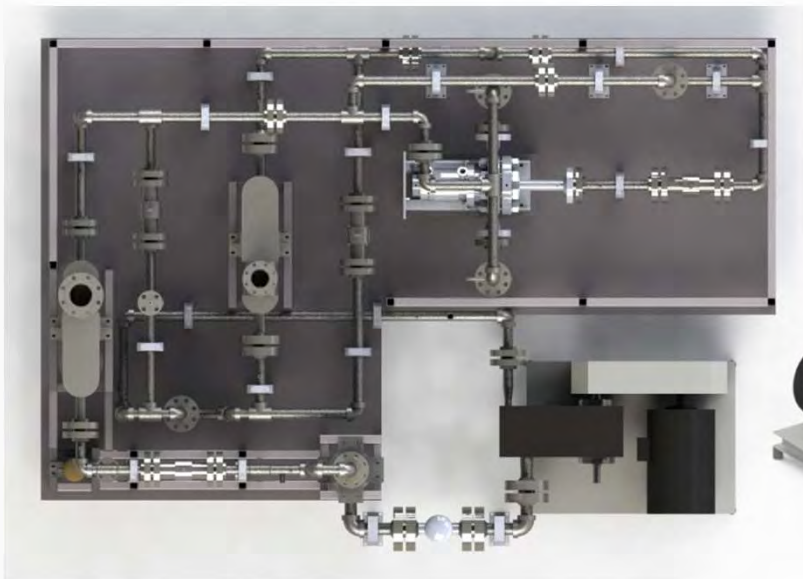


# Test loop (200°C off-design, Transcritical)

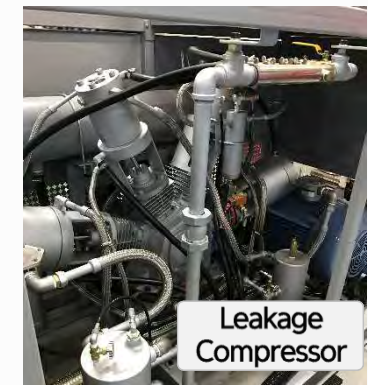
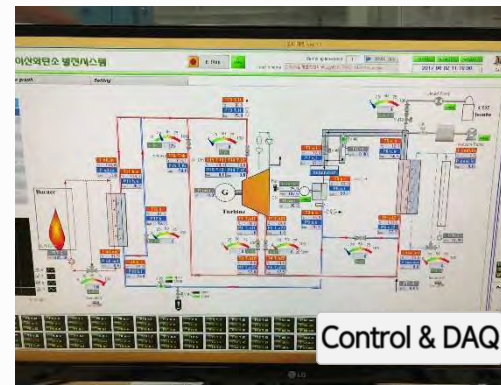
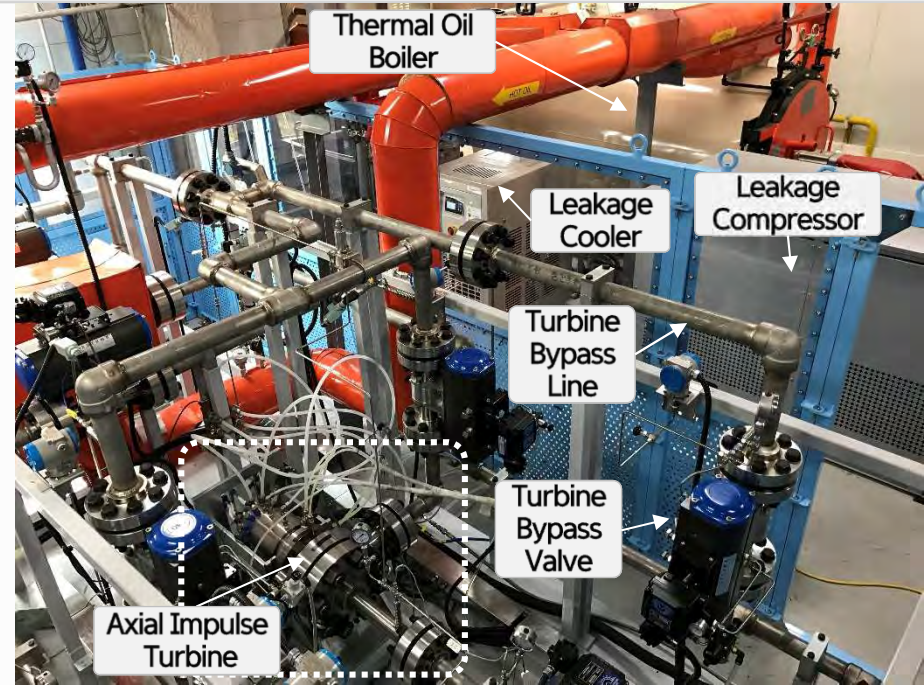
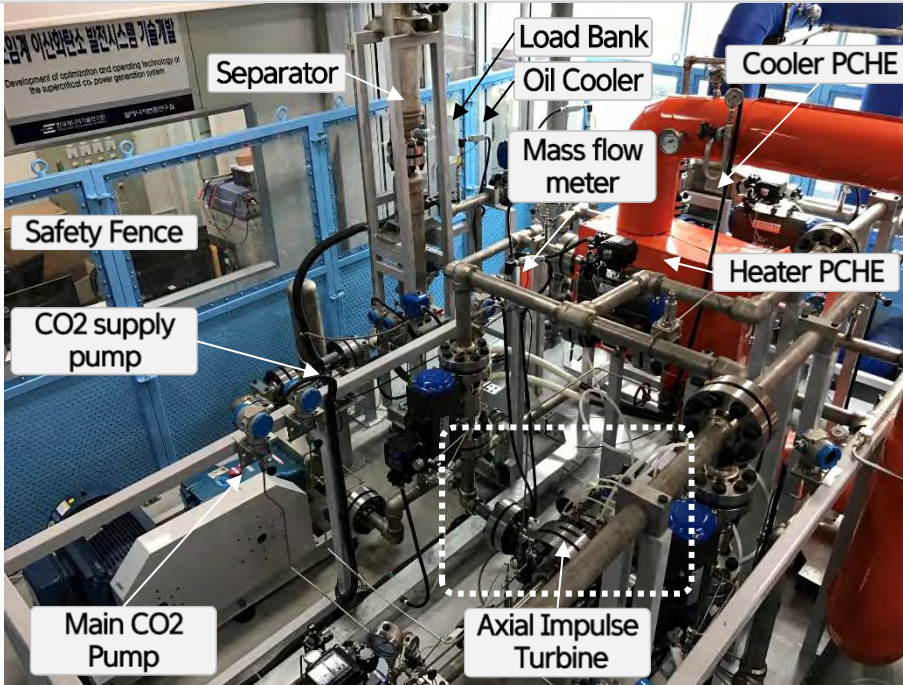


# Test loop (200°C off-design, Transcritical)

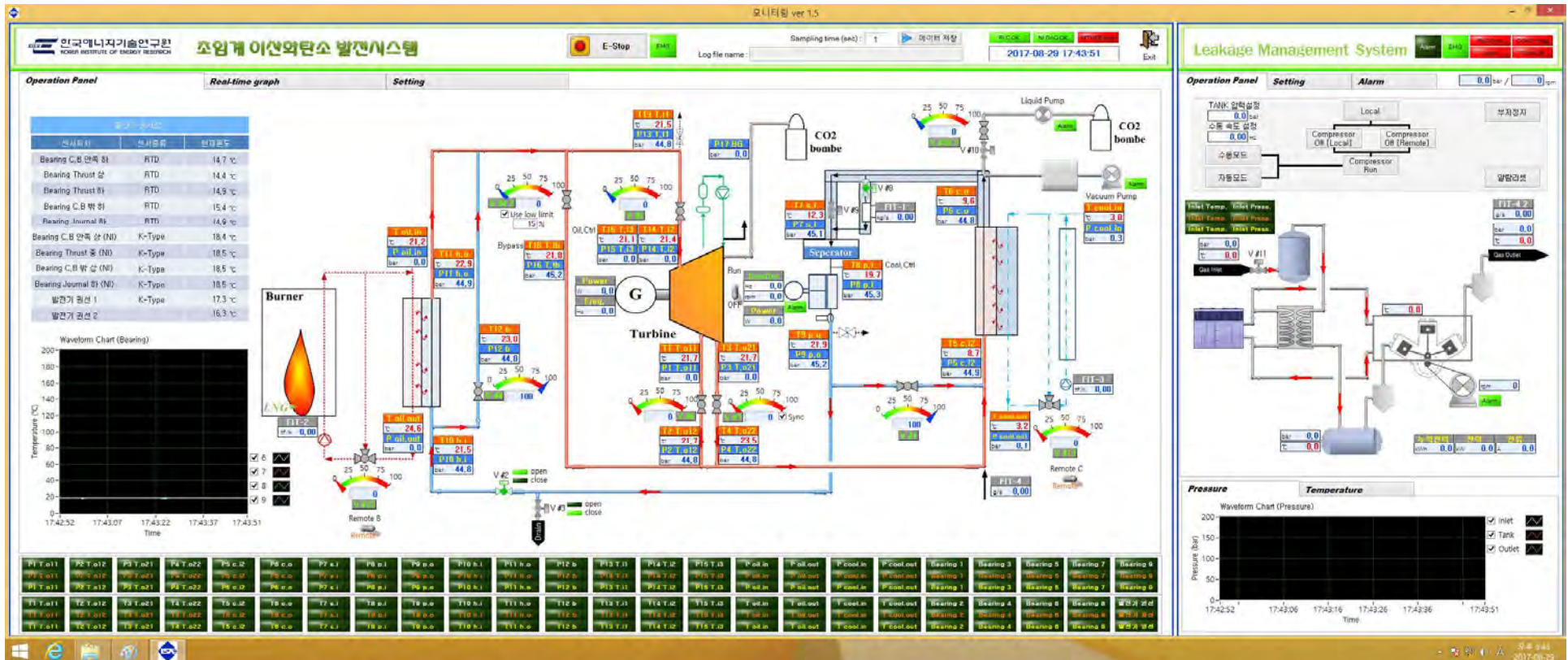
- **Pressure Vessel & Piping as ASME code guideline**
  - 2" SUS304, ANSI Flanges 1500, 2500 CLS
  - Metal Gaskets for CO<sub>2</sub>
  - Air-driven Control Valves
  - Cryogenic Valves & Regulators



# Test loop (200°C off-design, Transcritical)



# Tens of kWe Test Loop & Axial Turbo-generator DAS & Control



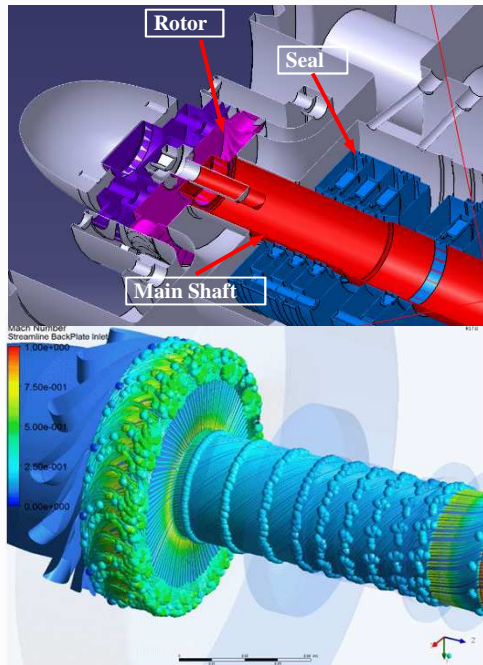
Bearing Temp. Monitoring

Main CO2 Loop Monitoring

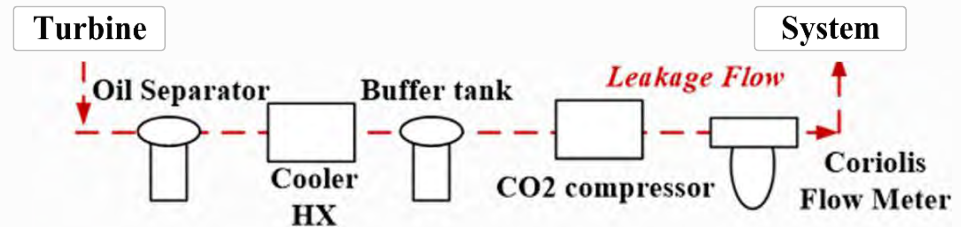
CO2 Leakage Monitoring

# Leakage Management

- Leakage Make-up System by using a 3-stages piston type compressor
  - Leakage : 2-3%(37g/s) of mass flow rate (1.57kg/s) through a floating carbon ring type mechanical seal
  - 3-stages compressor : 0.5 bar.g  $\rightarrow$  57~80 bar.g
  - Oil-separator, Cooler HX, Buffer tank, Chiller



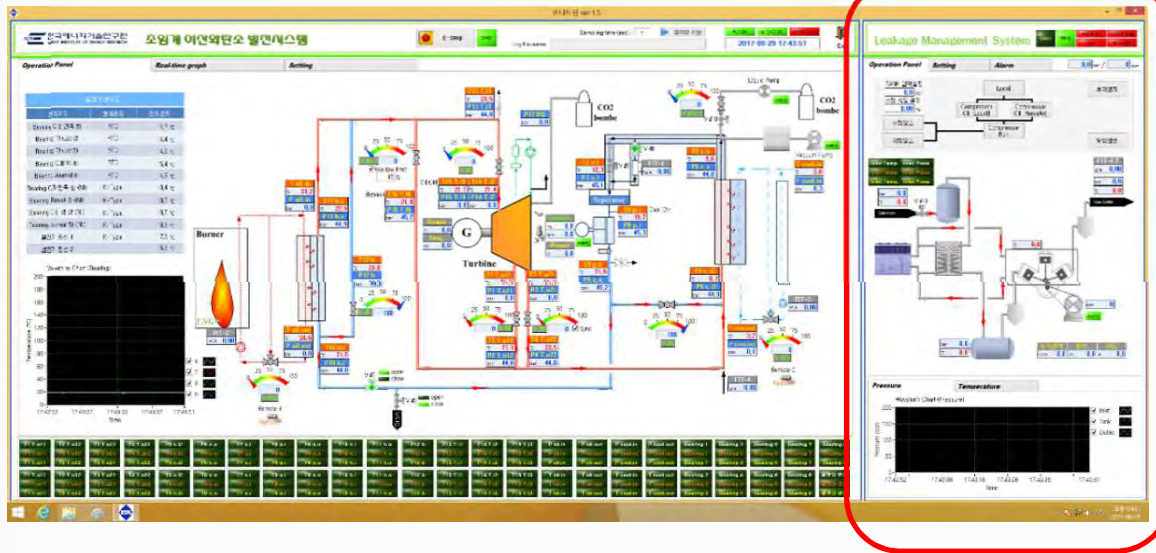
CFD analysis of leakage flow



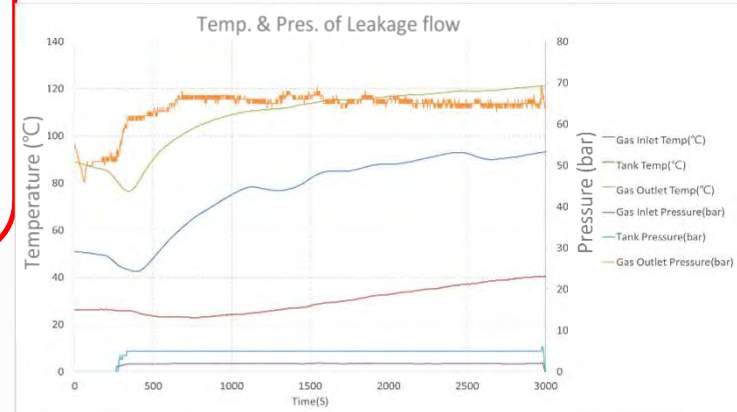
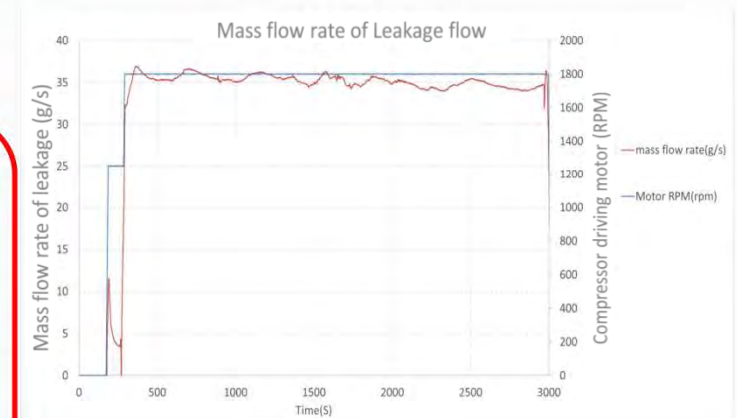
Leakage Make-up System

# Tens of kWe Test Loop & Axial Turbo-generator Leakage Management

- Leakage Make-up System by using a 3-stages piston type compressor
  - Continuous operation was successful



Leakage Make-up Controller

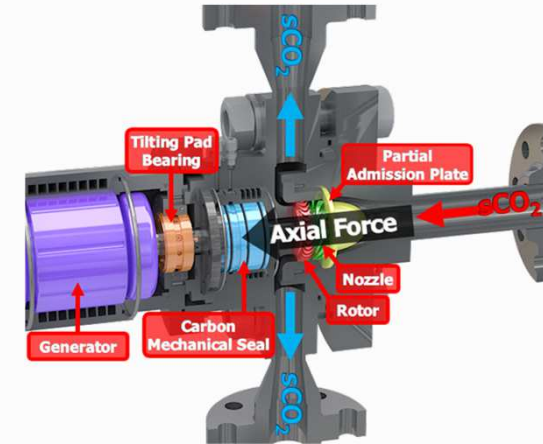


Operation Result

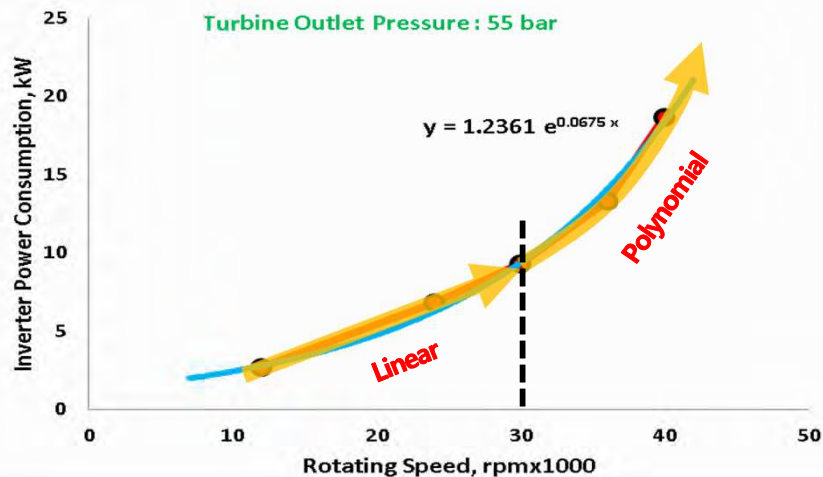


# Preliminary test

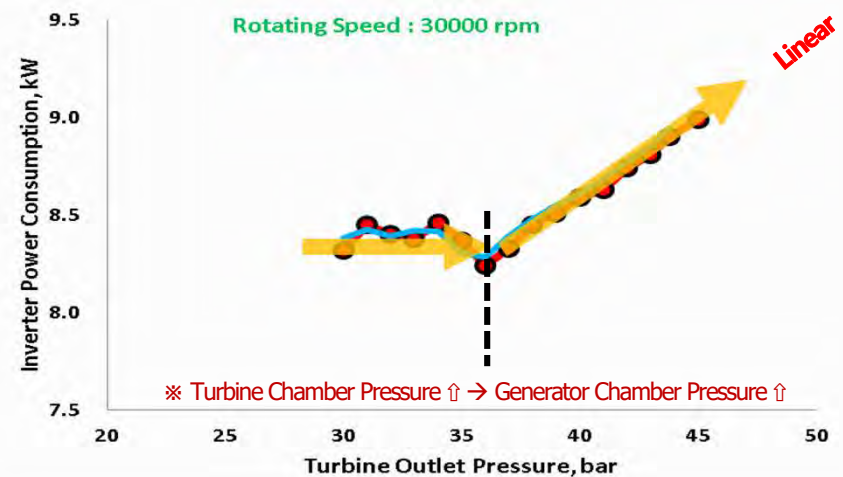
- Preliminary mechanical loss test as a RPM, Axial force
  - Turbo-generator was driven by an external electric inverter
  - Power consumption data was obtained
  - Mechanical loss of a tilting-pad bearing was estimated
  - High axial force was imposed on the bearing at the start
    - Inverter driving was necessary at the start



### "Rotating Speed" vs Power Consumption



### "Turbine Chamber Pressure" vs Power Consumption



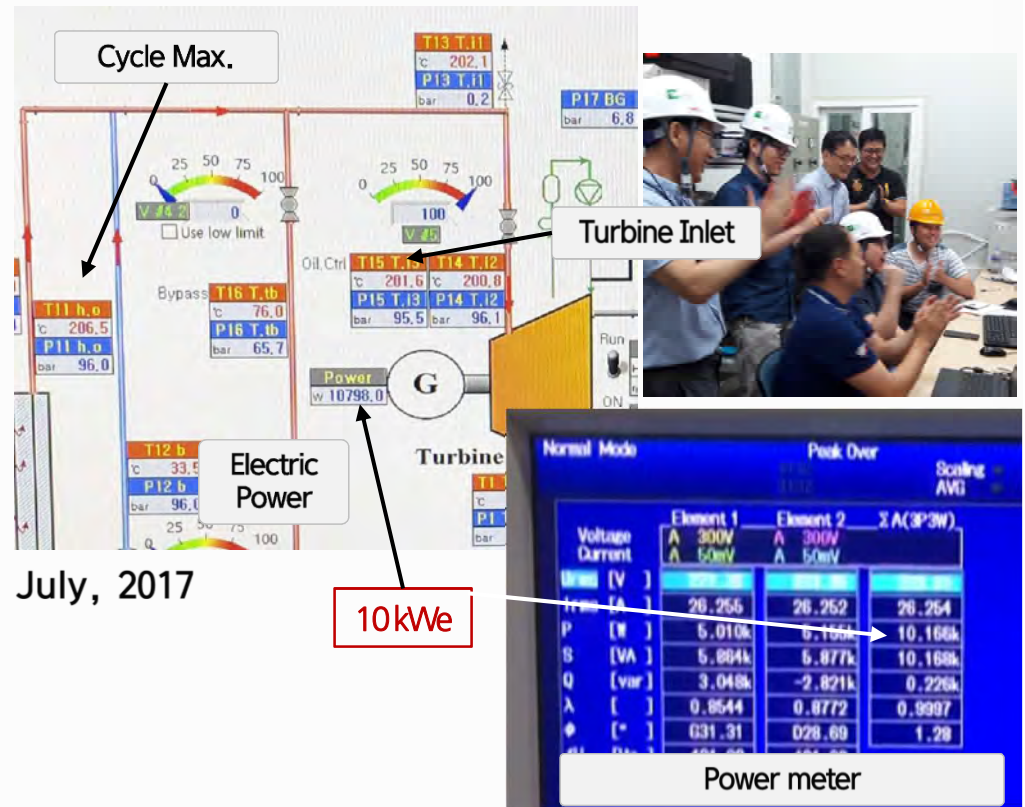
# Tens of kWe Test Loop & Axial Turbo-generator Preliminary Power Generation



- Electric Power generation with an Axial-type SCO2 Turbine

The objective of this project was not to demonstrate the efficiency benefits of SCO2 power cycles.

Rather, the objective of the project was to develop an SCO2 power generation system with an axial-type turbine resolving bearing failure problems reported by other research groups by applying turbomachinery technology applicable to a commercial plant.

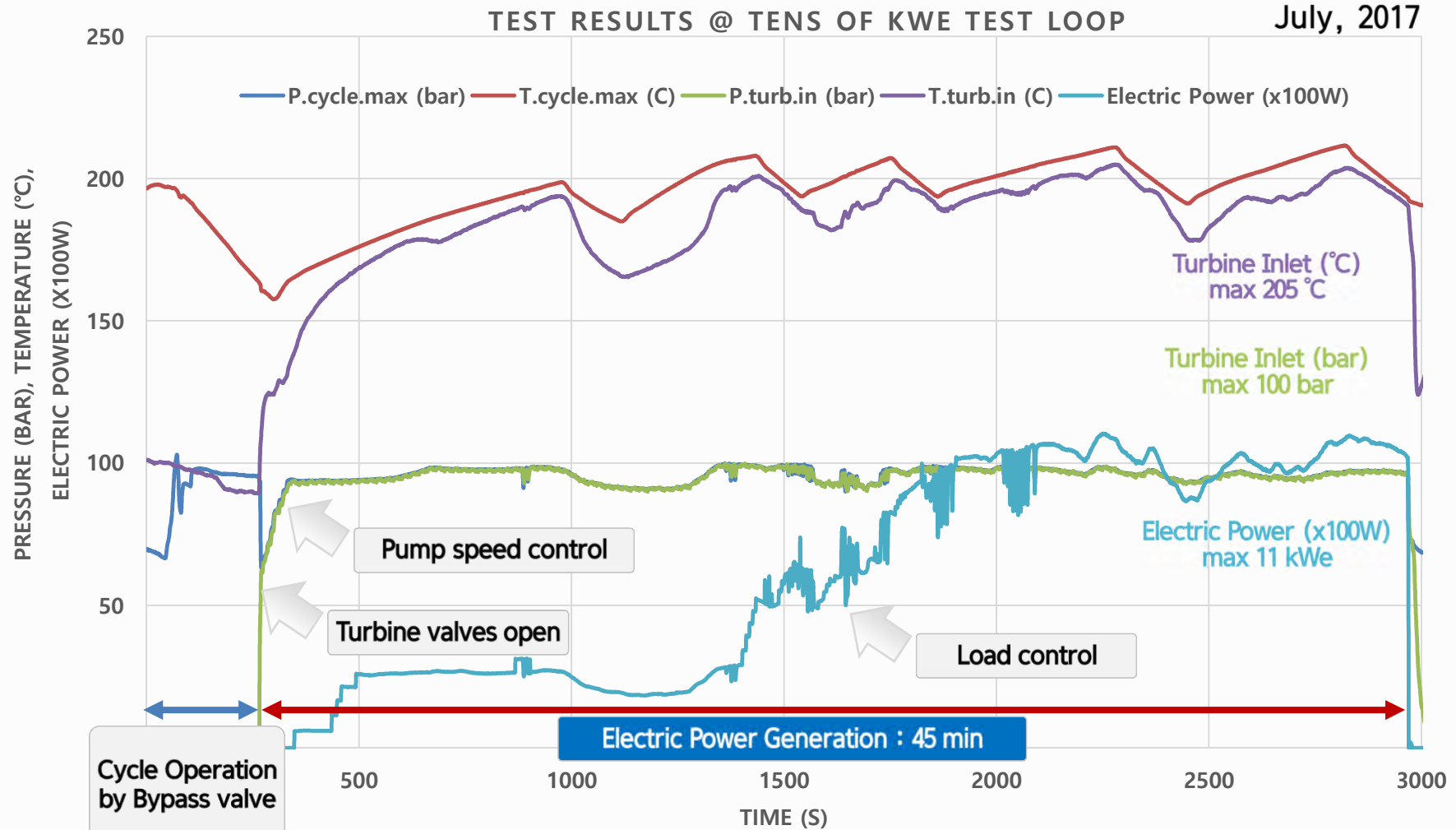


July, 2017

10kWe

Power meter

# Tens of kWe Test Loop & Axial Turbo-generator Preliminary Power Generation

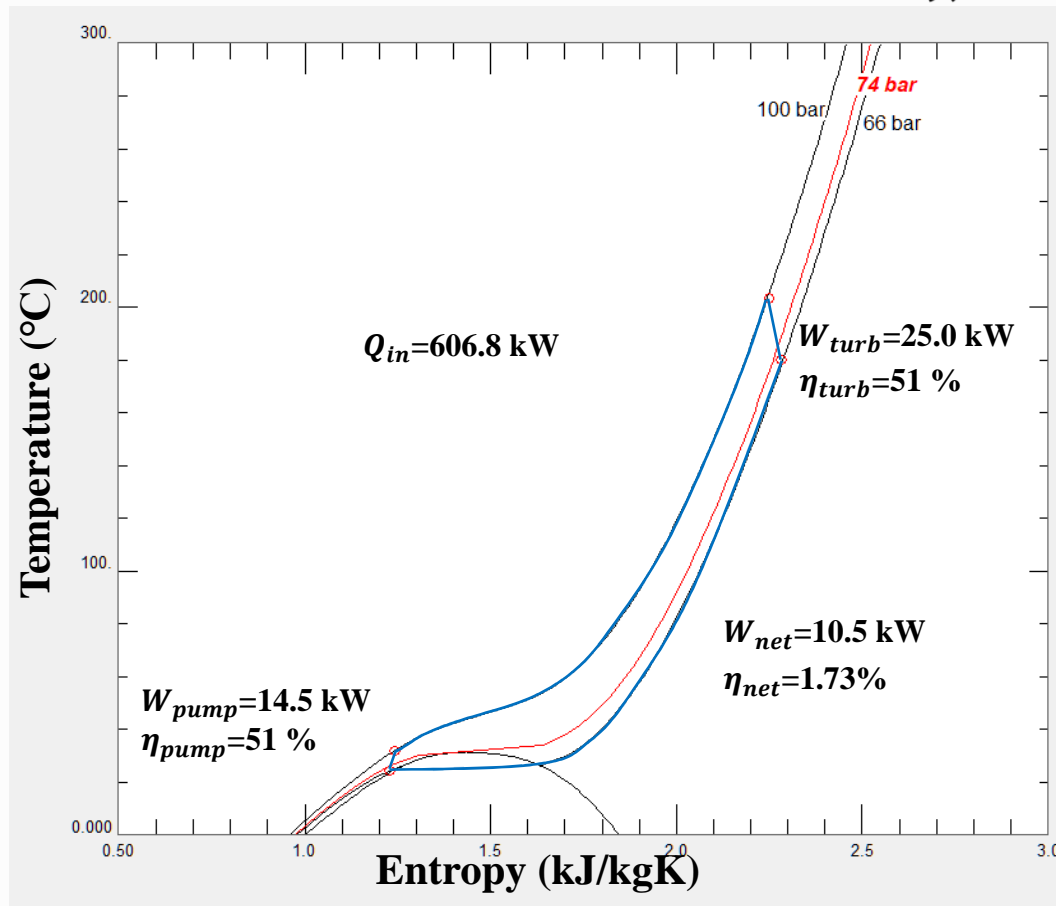


# Tens of kWe Test Loop & Axial Turbo-generator

## Preliminary Power Generation



July, 2017



Parameter	Value
Turbine Inlet Temperature (°C)	203.4
Turbine Inlet Pressure (bar)	98.1
Turbine Outlet Temperature (°C)	180.1
Turbine Outlet Pressure (bar)	67.7
Pump Inlet Temperature (°C)	24.3
Pump Inlet Pressure (bar)	65.9
Pump Outlet Temperature (°C)	31.7
Pump Outlet Pressure (bar)	99.1
Mass flow rate (kg/s)	1.69
Leakage mass flow rate (g/s)	34.46
Expansion ratio	1.45
Turbine Power (kW)*	25.0
Turbine efficiency (%)	51
Pump Power (kW)*	14.5
Pump efficiency (%)	51
Net Power (kW)*	10.5
Heat in (kW)*	606.8
Net efficiency (%)	1.73

\*Power was calculated by enthalpy difference

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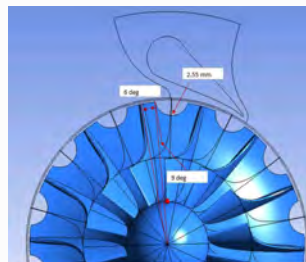
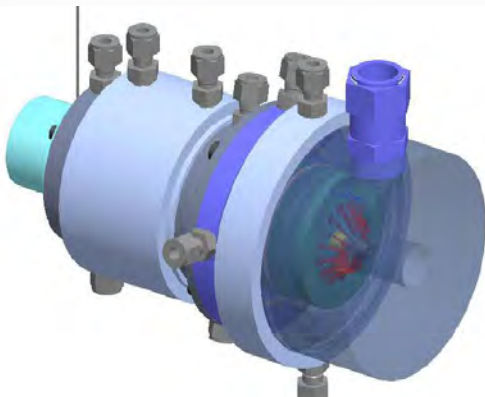
## 500°C kWe Test Loop & Radial Turbo-generator



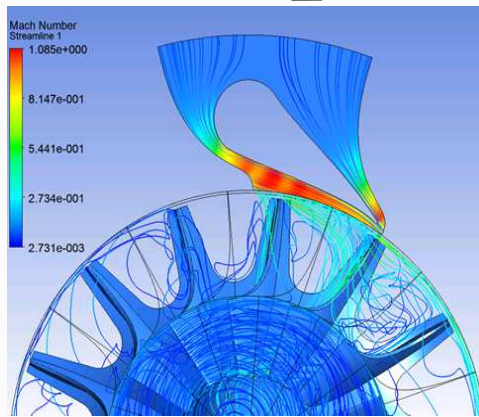
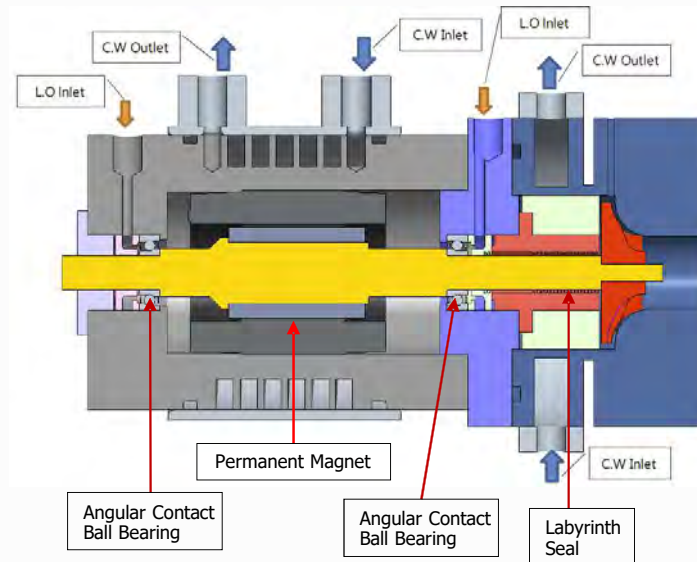
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# Turbo-generator for 500°C TIT

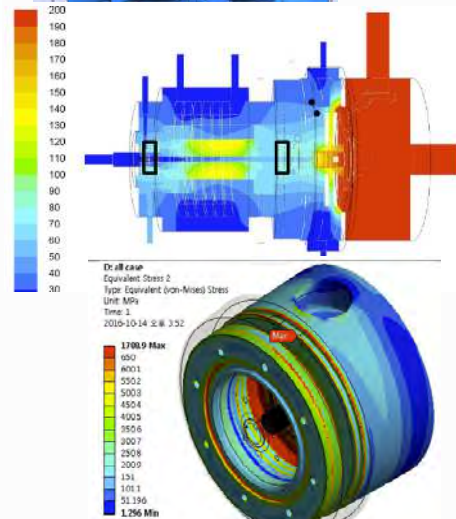
- Design power: 4.5 kW (w/o mechanical loss)
- Design rotational speed: 120,000 RPM
- Design axial force: 2 N
- 1/10 Partial admission nozzle



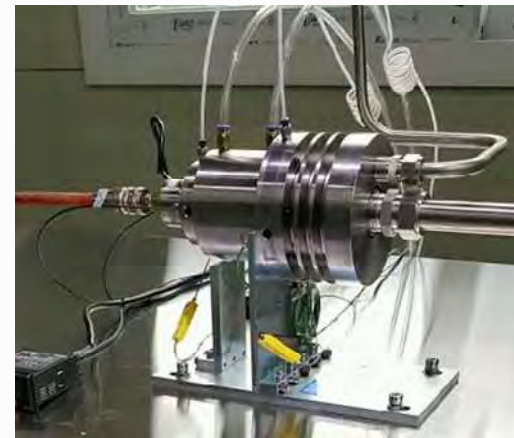
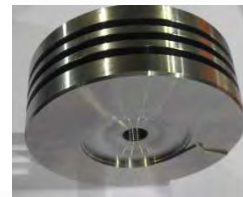
Scallop



Partial admission CFD

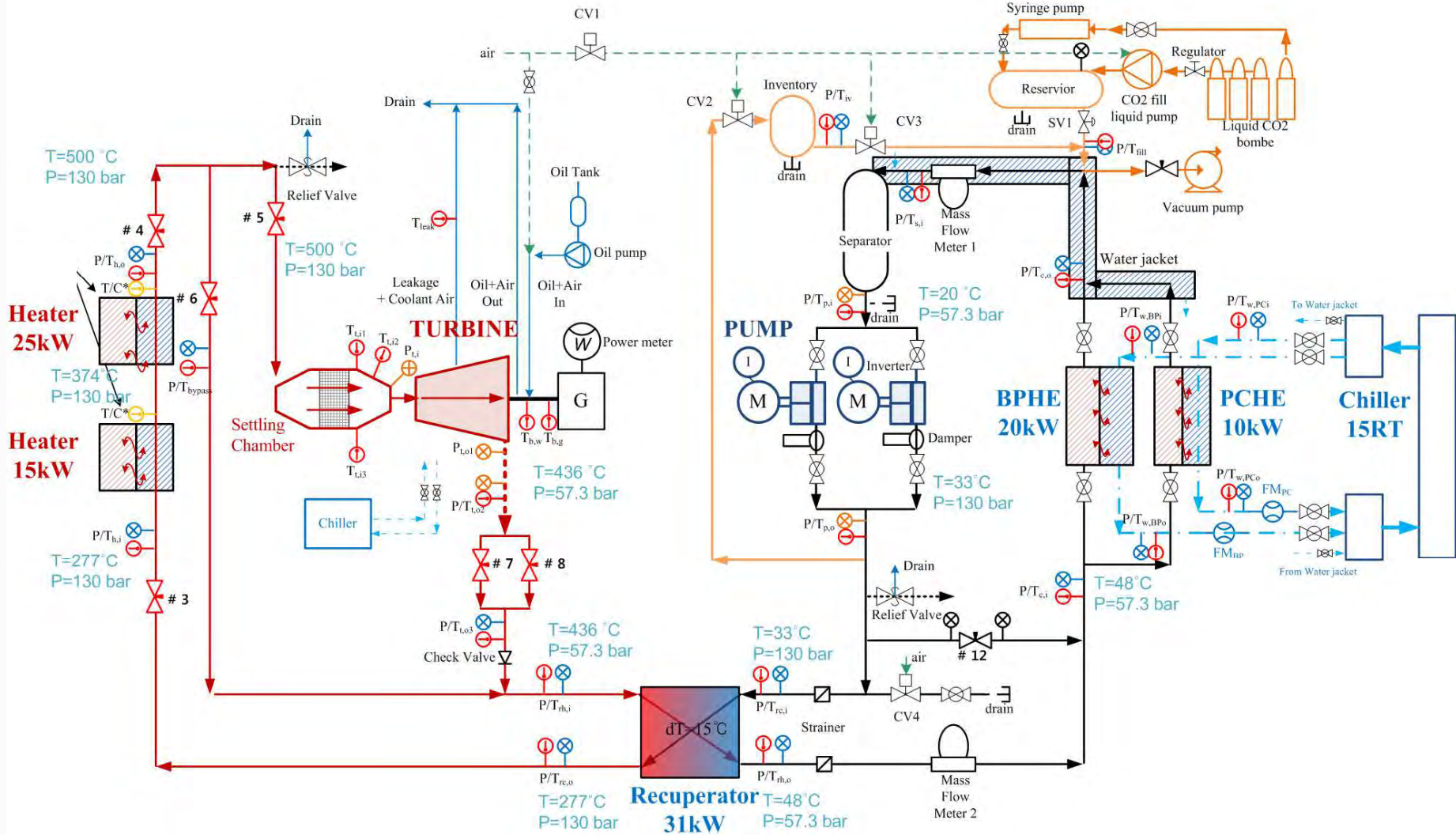


Thermal Analysis



# 500°C kWe Test Loop & Radial Turbo-generator

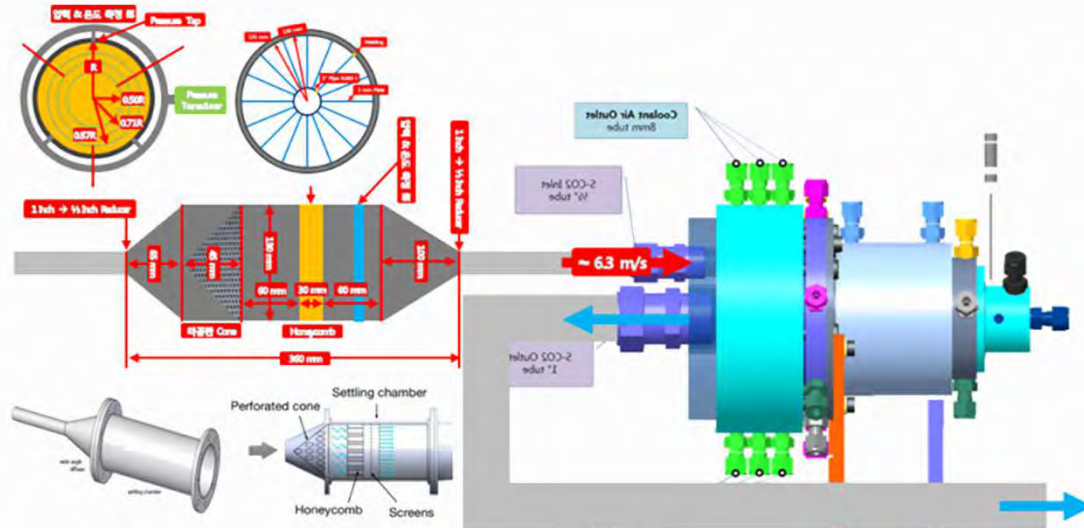
## Test loop (500°C, Transcritical)



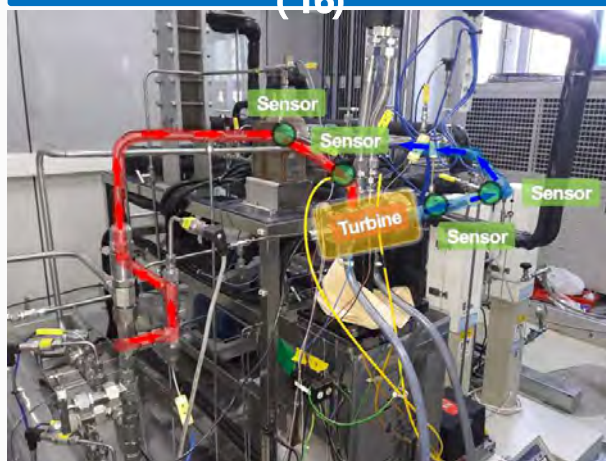
# 500°C kWe Test Loop & Radial Turbo-generator Test loop (500°C, Transcritical)



- Settling Chamber at the turbine inlet
  - Uniform Flow
  - Better instrumentation



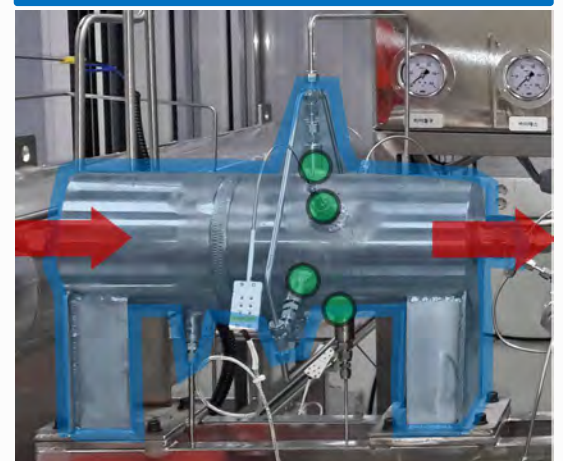
Previous Cycle Loop Test Facility ('16)



Cycle Loop Test Facility ('17)



Manufactured Settling Chamber

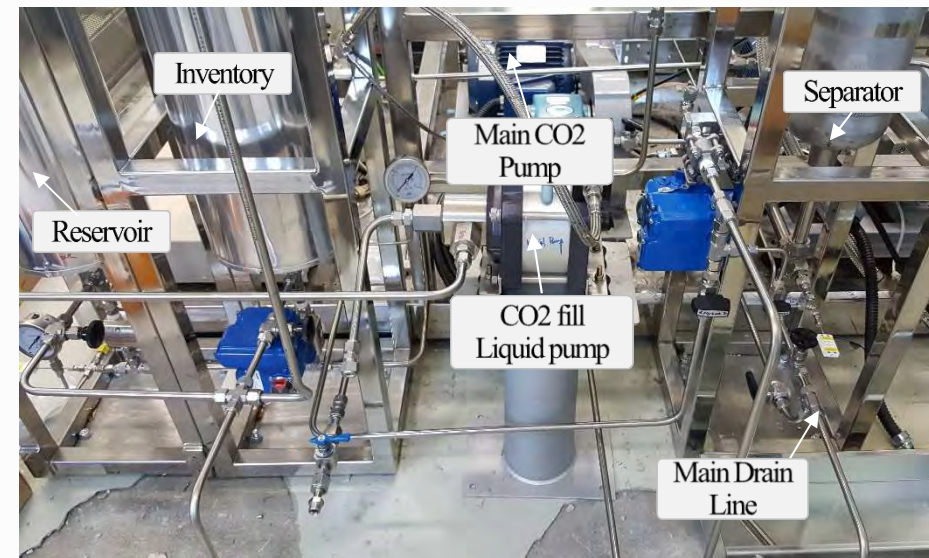
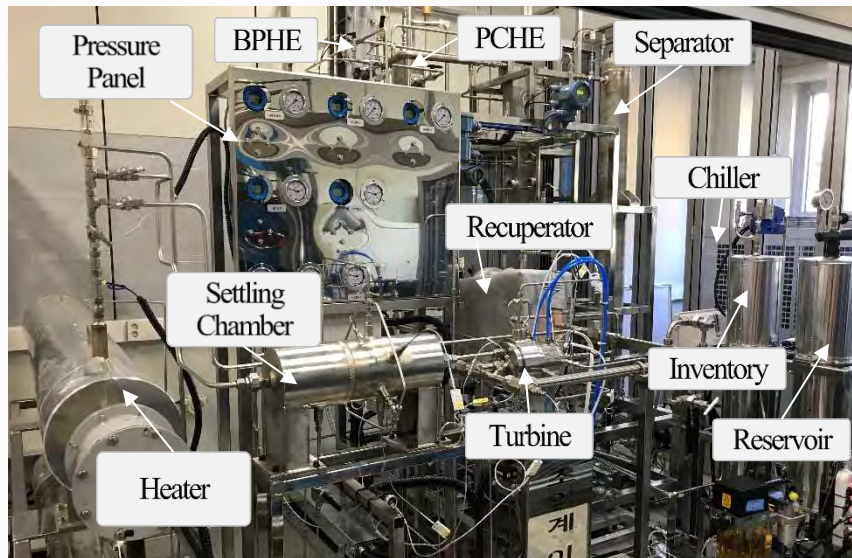




# Test loop (500°C, Transcritical)



## Test loop components



## Test Procedure



# Test loop (500°C, Transcritical)



- Final test loop with an insulation & safety fence



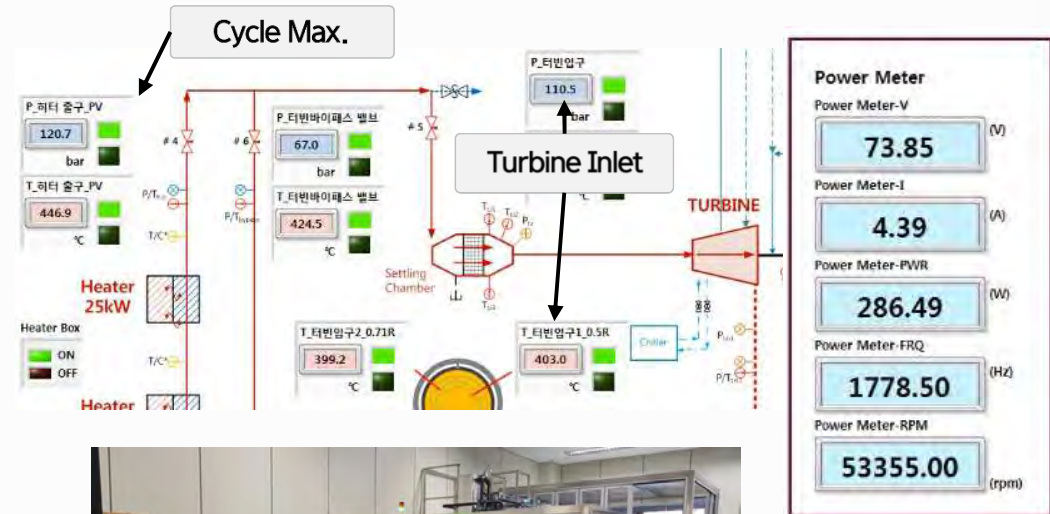
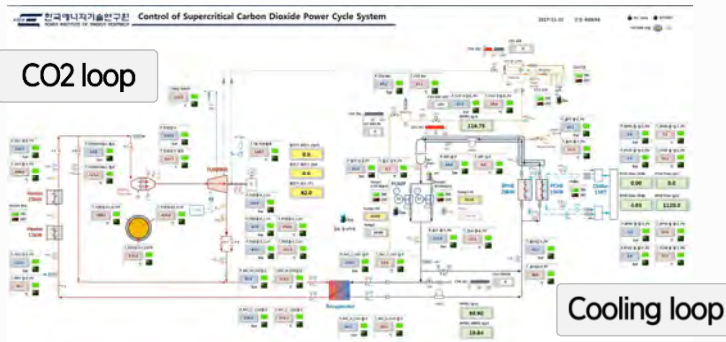
# 500°C kWe Test Loop & Radial Turbo-generator Preliminary Power Generation

**Cycle Max.**  
127 bar  
& 477 °C

**Turbine Inlet**  
112 bar  
& 401 °C

**Turbine Power**  
287 We

November, 2017



Bearing Temp.

Cycle Monitoring

Heater & Valves

Pump Control

Power meter

Control & Monitoring System

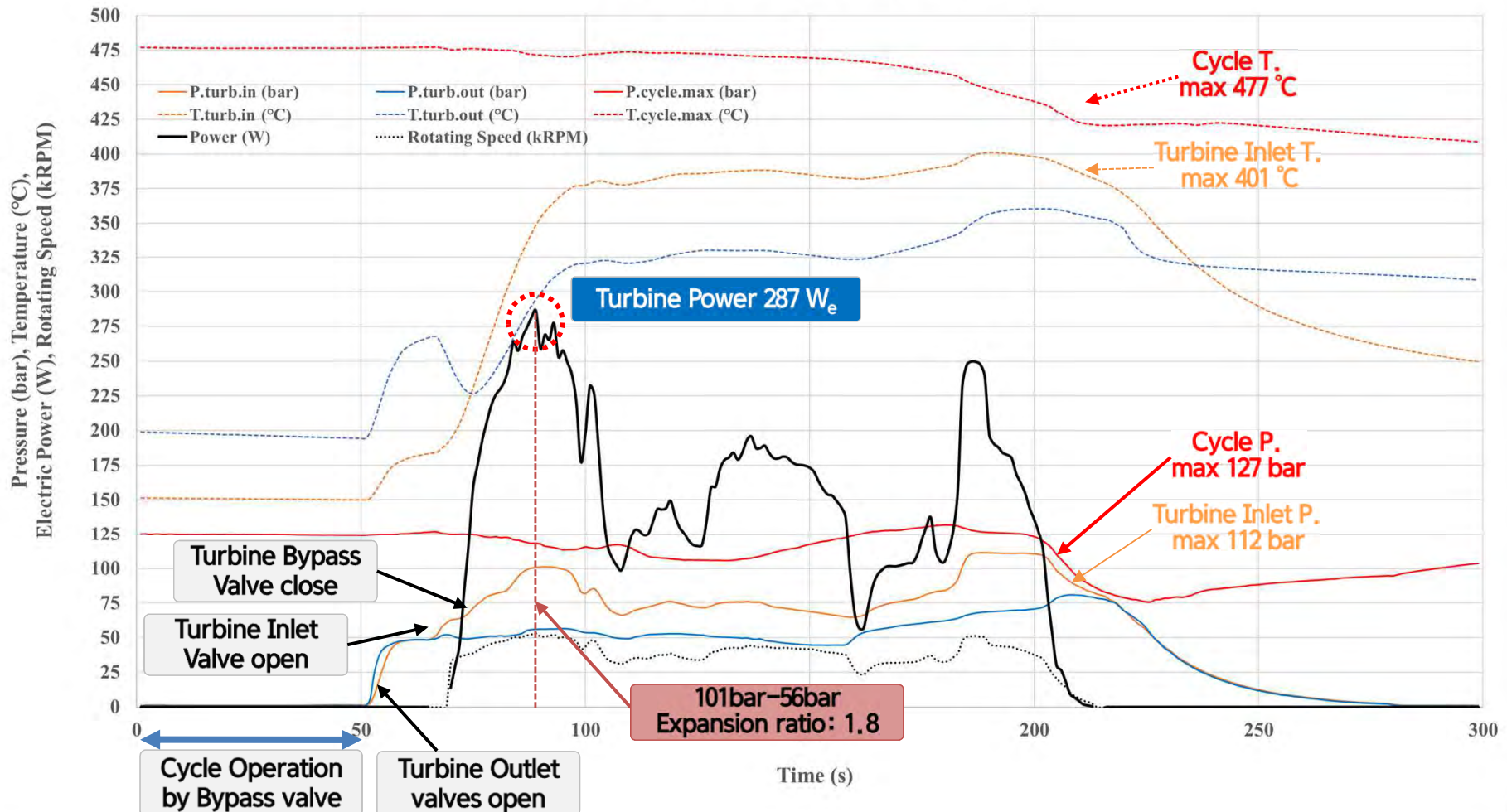


# Preliminary Power Generation

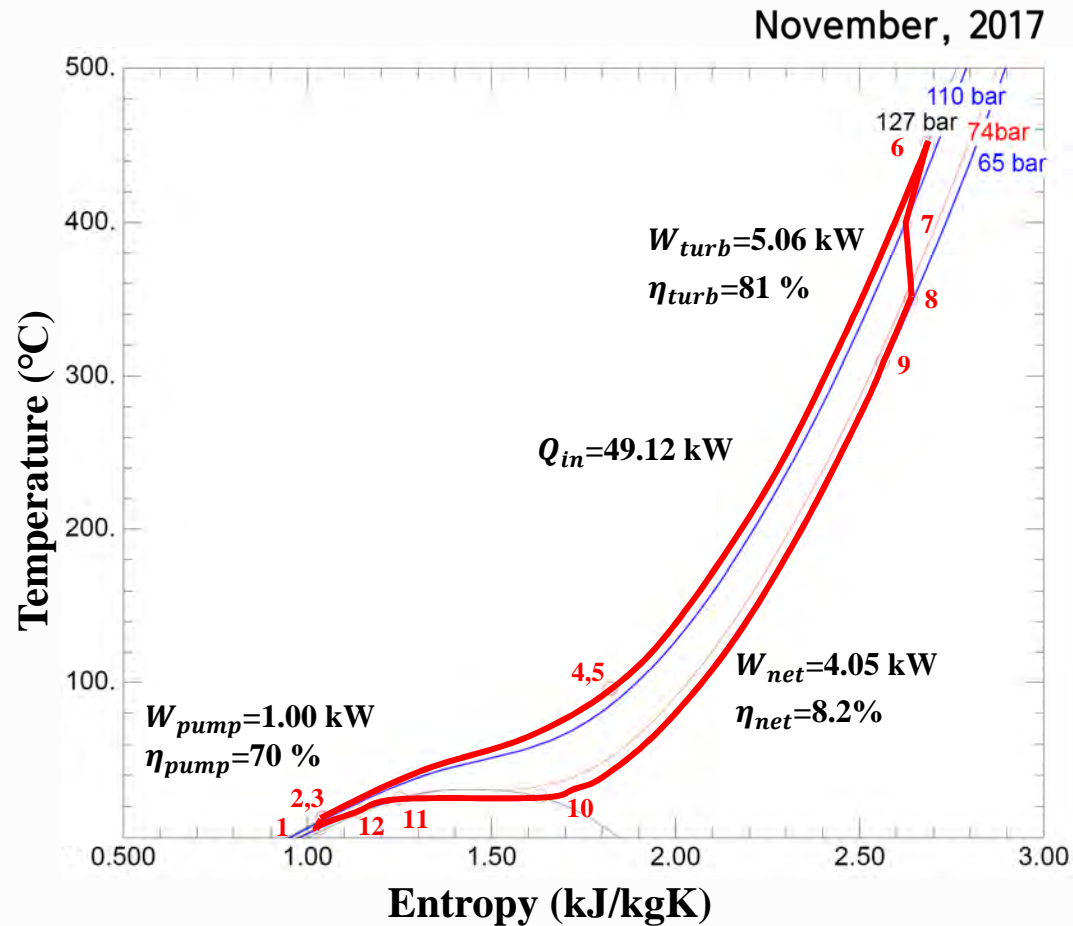


TEST RESULTS @ KWE TEST LOOP

November, 2017



## Preliminary Power Generation



	Temp. (°C)	Pressure (bar)
1. Pump.Inlet	6.2	64.9
2. Pump.Outlet	12.4	128.2
3. Recup.Cold.Inlet	12.7	126.9
4. Recup.Cold.Outlet	96.3	126.8
5. Heater.Inlet	95.8	124.5
6. Heater.Outlet	451.2	124.5
7. Turbine.Inlet	398.75	110.5
8. Turbine.Outlet	349.9	67.3
9. Recup.Hot.In	309	67.2
10. Recup.Hot.Out	27	67
11. Cooler.Inlet	25.4	65.5
12. Cooler.Outlet	17.1	65.3
Mass flow rate	110.19 g/s	
Turbine Leakage	7.66 g/s (7%)	
Pump Work	1.00 kW	
Pump Efficiency	70 %	
Turbine Work	5.06 kW	
Turbine Efficiency	81 %	
Expansion Ratio	1.64	
Heat In	49.12 kW	
Heat Out	13.61 kW	
Recuperator	38.53 kW (Hot CO <sub>2</sub> )	
Recuperator Approach Temp.	15 °C	
Recuperator CO <sub>2</sub> pressure drop	0.1 bar (Cold CO <sub>2</sub> ) 0.2 bar (Hot CO <sub>2</sub> )	
Net Power	4.05 kW	
Net Efficiency	8.2 %	

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## Summary & Future Works



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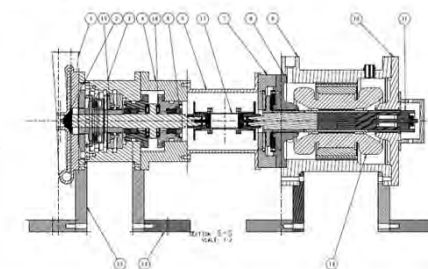
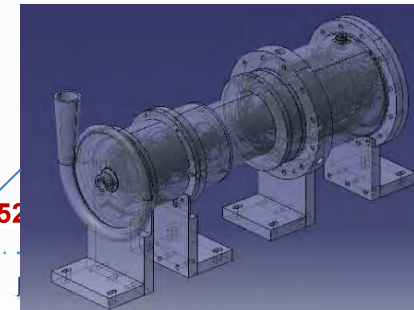
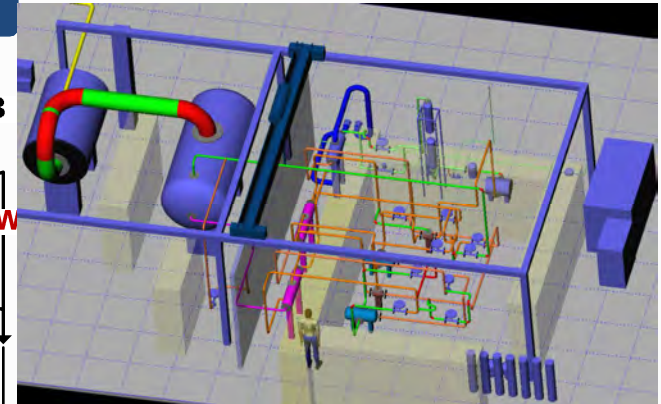
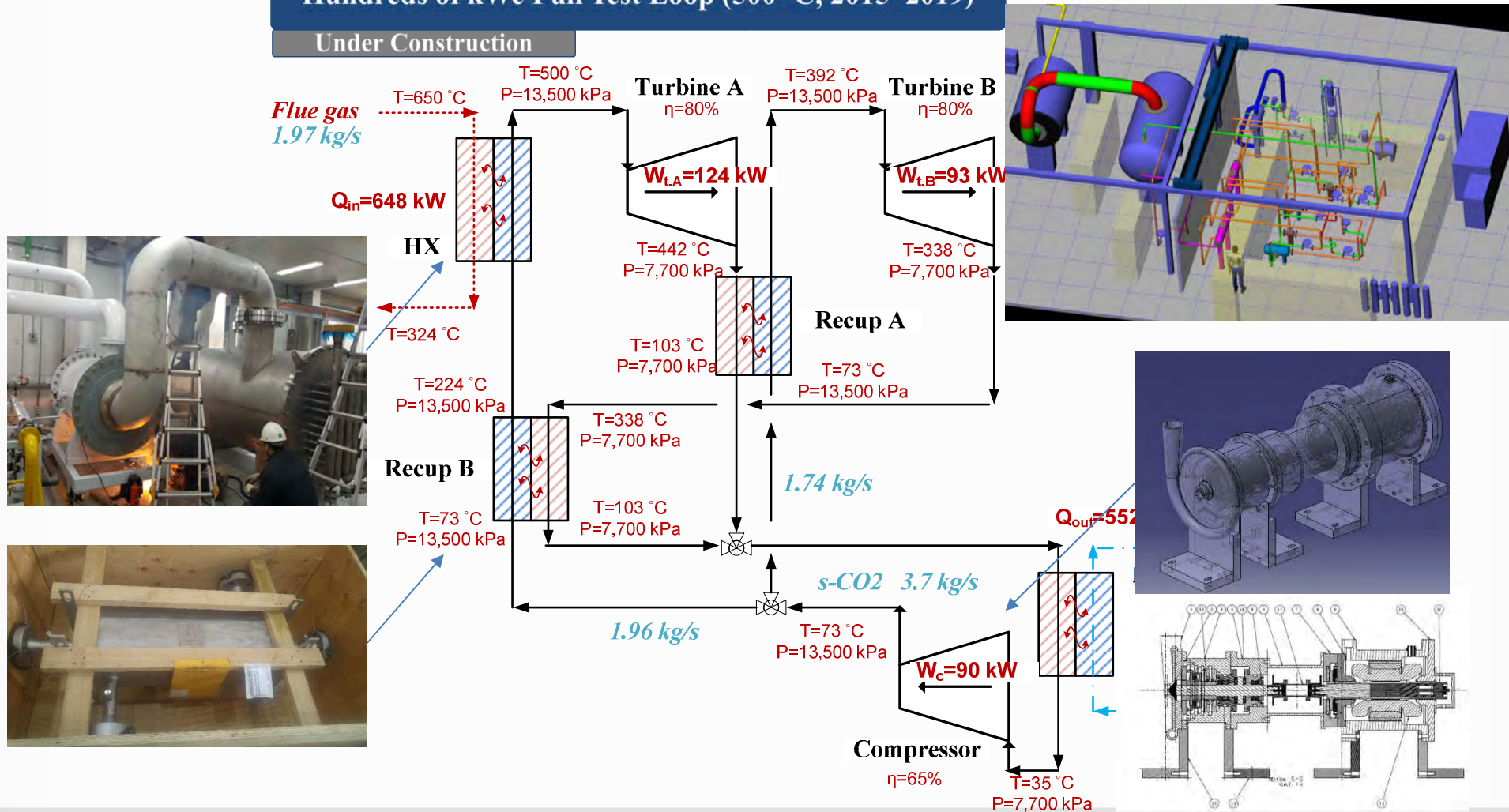
- **KIER's S-CO<sub>2</sub> Power Cycle Test Loops :**
  - 2013-2014 : 10 kWe test loop, 200°C
  - 2014-2016 : sub-kWe test loop, 200°C (Power Generation)
  - 2015-2017 : tens of kWe test loop, 200°C (Power Generation)  
Axial impulse turbo-generator
  - 2016-2017 : kWe test loop, 500°C (Power Generation)  
Radial turbo-generator
  - 2016-2019 : hundred of kWe test loop, 500°C (under construction)



# Future Works (In-direct)

## Hundreds of kWe Full Test Loop (500 °C, 2015~2019)

Under Construction







# Future Works (Direct)

- **Development of a combustor for direct SCO<sub>2</sub> (Allam cycle)**
  - KIER started small project in 2018
  - Development of a basic combustion test loop

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Thank you for your attention



The KIER, a global energy innovator, does its best in pursuing its mission to invent world-class energy technologies based on open innovation, life-cycle research quality assurance, participatory and open communication. Therefore the KIER will become the best energy technology R&D institute in the world, contributing to the creation of wealth and improvement of quality of life for the people.