

# The influence of radiative transfer on the turbulent flow inside solar absorbers operating with supercritical CO<sub>2</sub>

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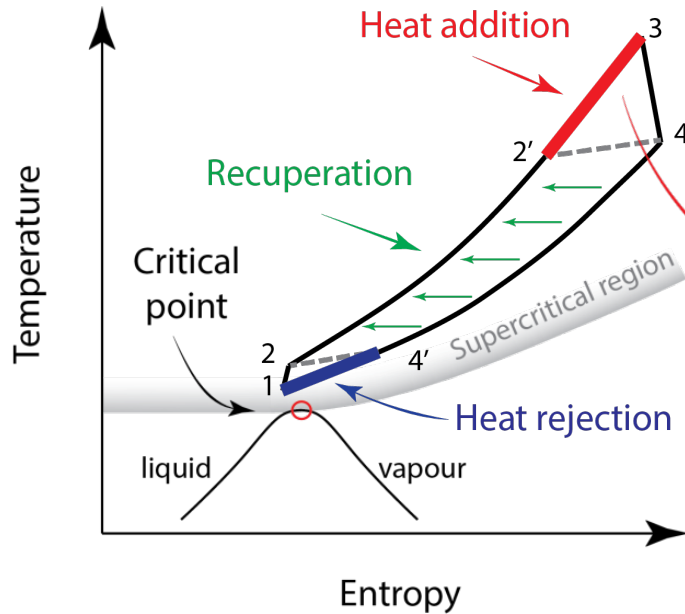
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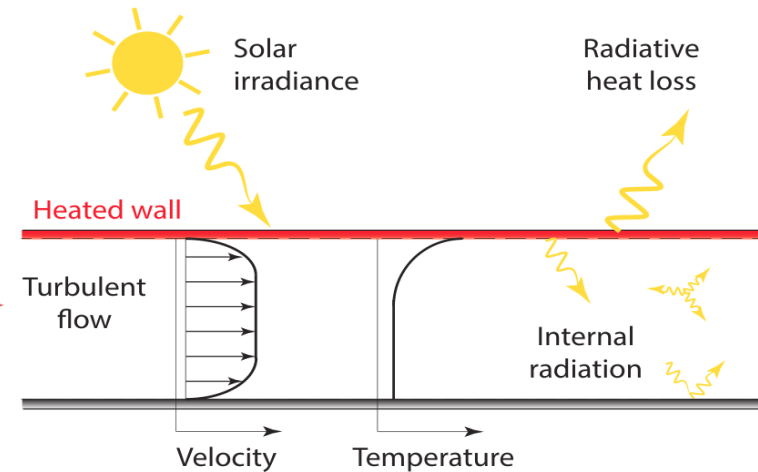
# Contents

- Surface vs. volumetric receivers
- Optical properties of scCO<sub>2</sub> and nanoparticles
- Numerical methods
- Investigated cases
- Results
- Conclusions and recommendations

# Solar receivers for scCO<sub>2</sub> cycle



## Surface receiver



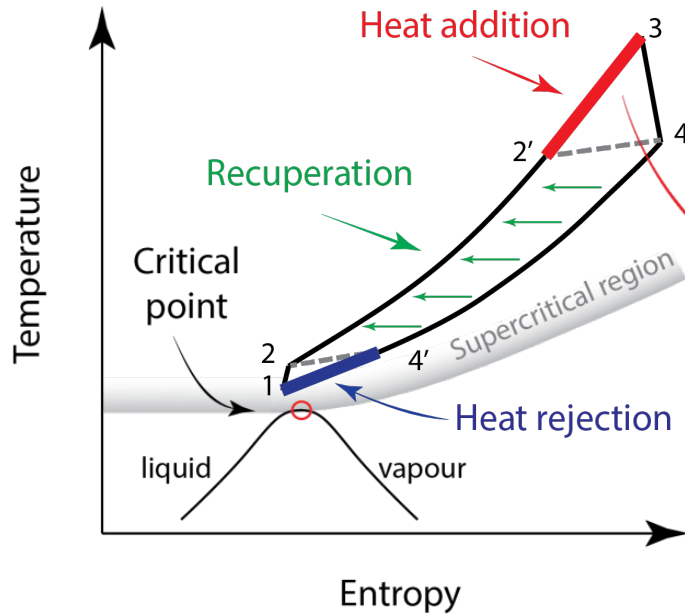
Selective coating at the surface

- High absorptance for solar band
- Low emittance for infrared band

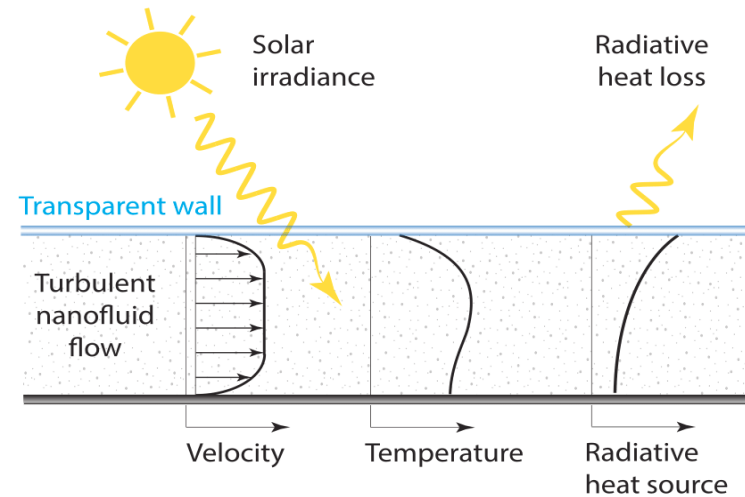
High wall temperatures

- Radiation inside the receiver
- Relatively large emissive losses
- Low efficiency

# Solar receivers for scCO<sub>2</sub> cycle



## Volume receiver



Radiation absorbed by nanoparticles (1 – 100 nm)  
Optimize absorption with volume fraction & material

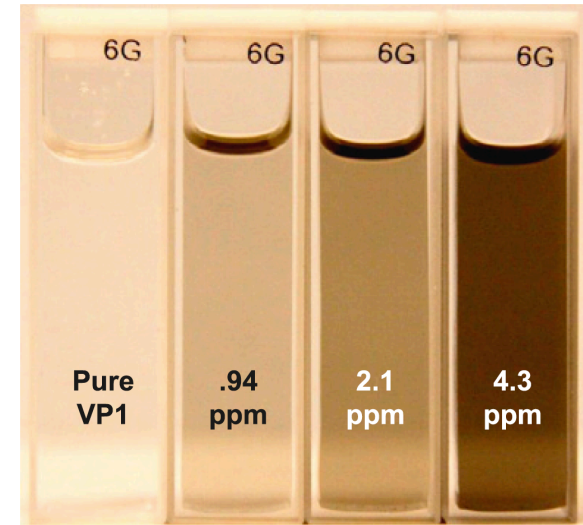
Nanoparticles:

- no significant sedimentation velocity
- no significant erosion to the equipment
- do not readily deposit on walls

# Nanofluids for solar thermal receivers

## Nanofluids solar thermal receivers:

- Hordy et al., Solar Energy, 2014
  - Metal- and carbon based nanoparticles in water and VP1
  - Thermal stability tests up to 220 °C (VP1 boiling temp. 257 °C)
- Metallic nanoparticles:
  - Plasmonic effect



Carbon coated cobalt nanoparticle in VP1  
Lenert et al., Solar Energy, 2012

## scCO<sub>2</sub> used for nanoparticle synthesis (RESS) and mixing:

- G.P. Sanganwar, R.B. Gupta, Powder Technology, 196(1) (2009) 36-49.
- M. Carrott, C. Wai, Analytical chemistry, 70(11) (1998) 2421-2425.

# Research questions

- 1. How does radiation within surface receiver affect performance?**
- 2. What is the theoretical potential of volumetric receivers?**
  - Comparison surface and volumetric receivers with  $\text{scCO}_2$

# Optical properties

- scCO<sub>2</sub>
- Nanoparticles
- Nanofluid (scCO<sub>2</sub> + nanoparticles)

# Optical properties of scCO<sub>2</sub>

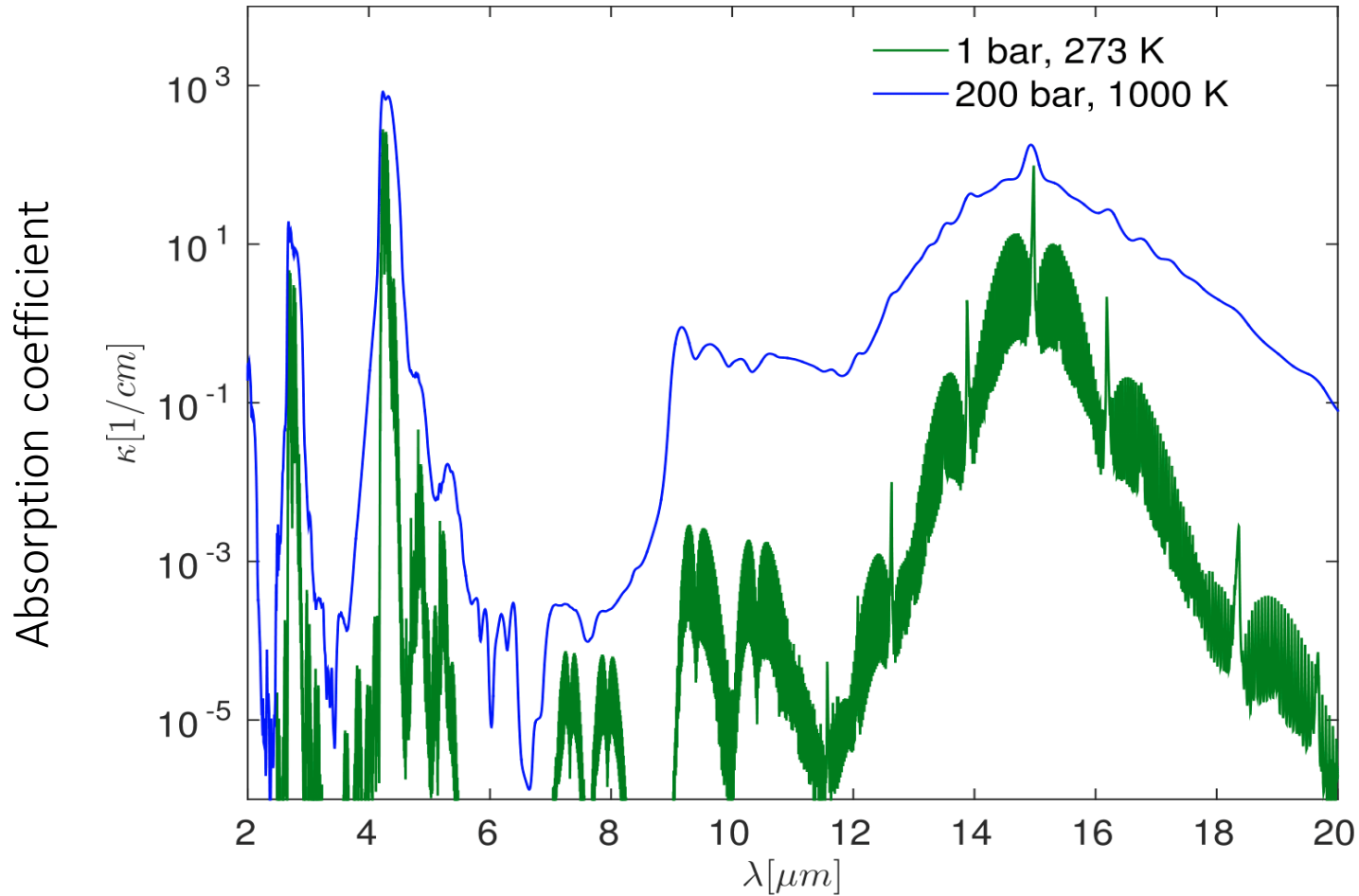
- HITRAN database → spectral lines
- Lorentz broadening of spectral lines
- Sub-Lorentzian behaviour
  - Deviation of Lorentz shape (Perrin & Hartmann, 1989)
  - Extrapolation of empirical relations
- Open-source software to obtain k-spectrum

## Refs:

- Rothman et al., 2013. J. of Quantitative Spectroscopy and Radiative Transfer, 130, 4–50.
- Rosenmann, et al., 1988. Applied Optics, 27(18), 3902–3907.



# Optical properties of s-CO<sub>2</sub>



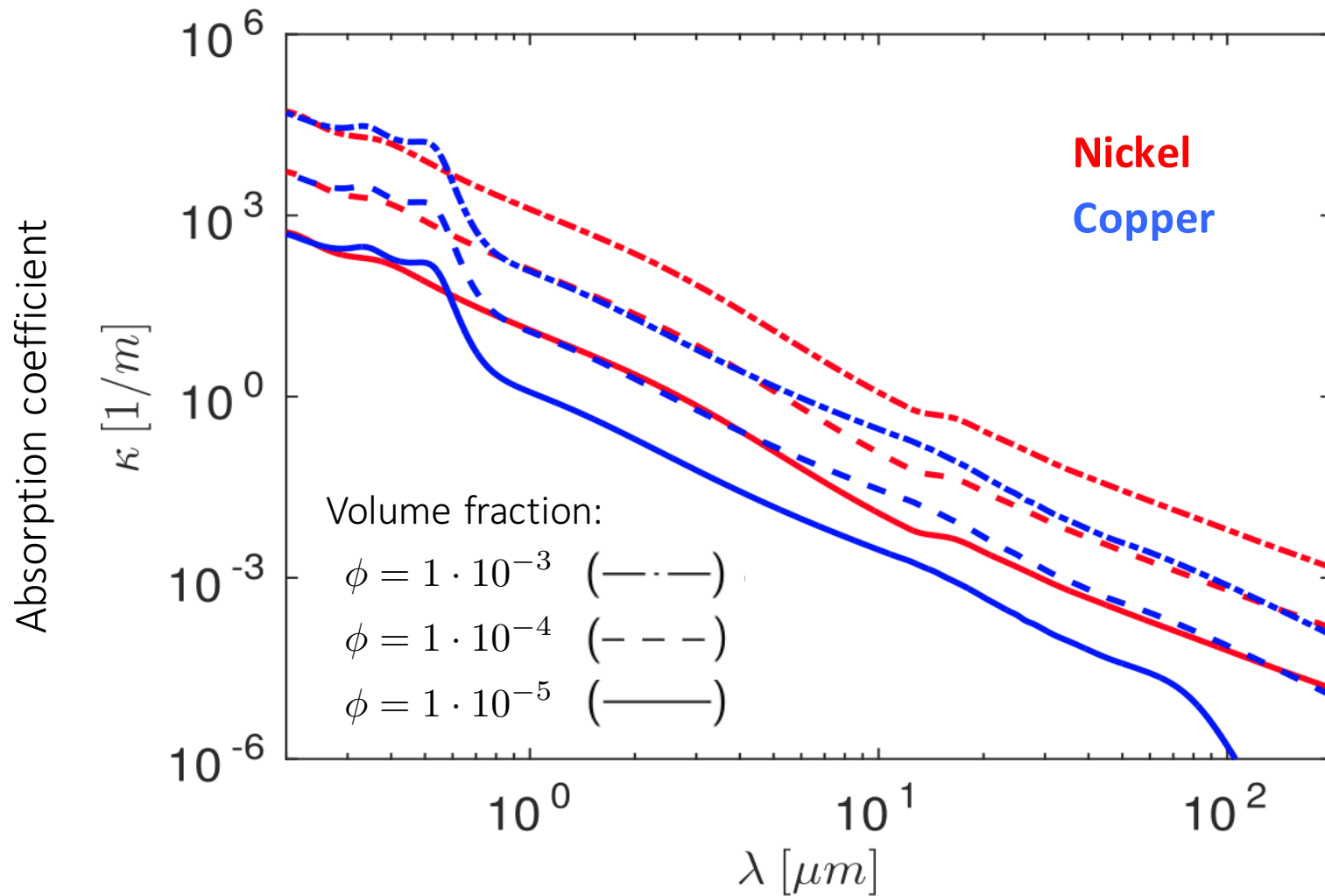
# Optical properties of particles

- Two materials chosen with high melting temperatures
  - Nickel and Copper
- Rayleigh scattering theory (Modest, 2012)
  - Very small particles compared to wavelength
  - Transparent medium with particles
  - Low volume fraction

$$\kappa_{\lambda} = -\Im \left( \frac{m^2 - 1}{m^2 + 2} \right) \frac{6\pi\phi}{\lambda}.$$

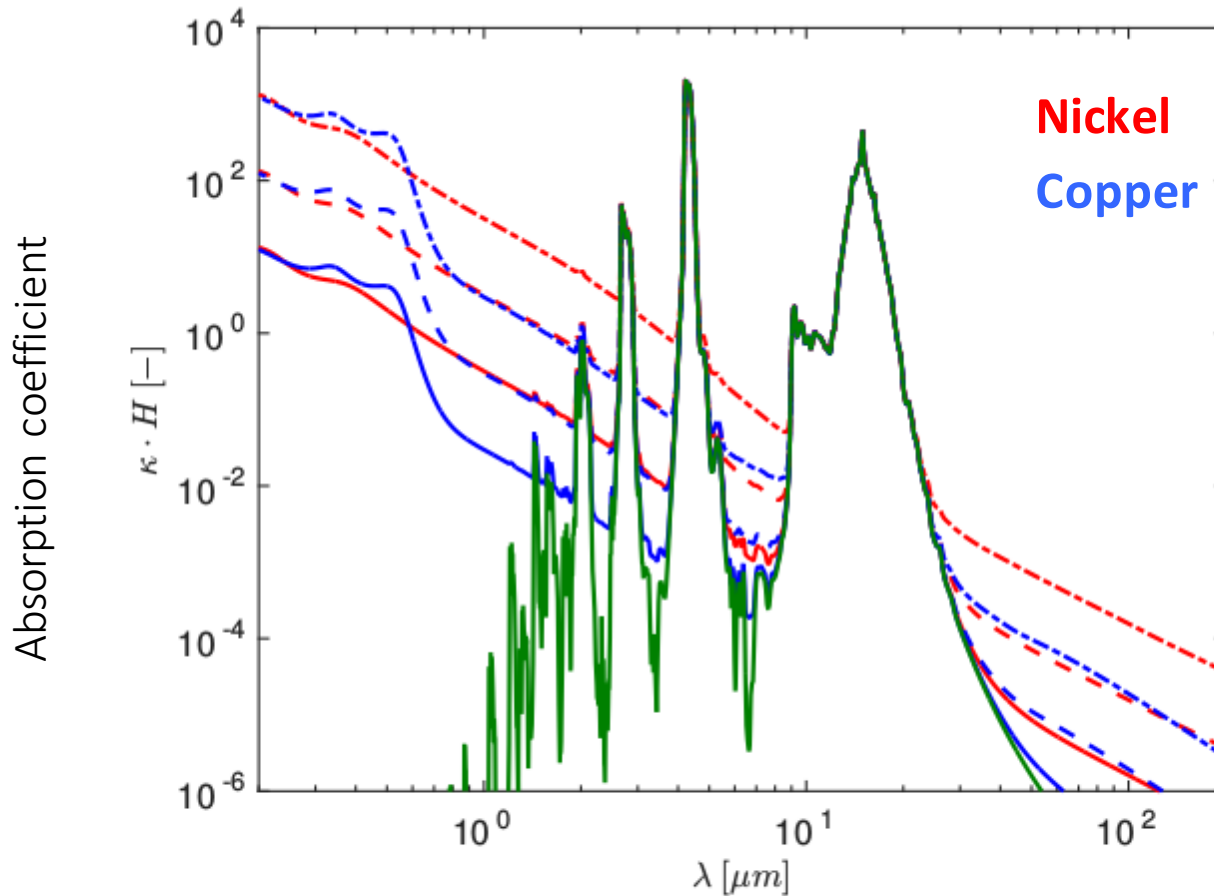
- Complex index of refraction from literature
  - Ordal et al. (1985), Rakic et al (1998)

# Optical properties of particles



# Optical properties of particles

- Linear combination of absorption spectra:  $\kappa_{nf} = \kappa_p + \kappa_f$



Volume fraction:

$$\phi = 1 \cdot 10^{-3} \quad (\text{---}\cdot\text{---})$$

$$\phi = 1 \cdot 10^{-4} \quad (\text{---})$$

$$\phi = 1 \cdot 10^{-5} \quad (\text{---})$$

# Numerical method

Navier-Stokes solver

Radiative transfer solver

# Numerical method

- Favre averaged Navier-Stokes equations (enthalpy based)

$$\begin{aligned}\frac{\partial \bar{\rho} \tilde{u}_i}{\partial x_j} &= 0, \\ \frac{\partial \bar{\rho} \tilde{u}_i \tilde{u}_j}{\partial x_j} &= \frac{\partial \bar{p}_i}{\partial x_i} + \frac{1}{Re_0} \frac{\partial \bar{\tau}_{ij}}{\partial x_j}, \\ \frac{\partial \bar{\rho} \tilde{h} \tilde{u}_j}{\partial x_j} &= \frac{\partial}{\partial x_j} \left[ \left( \frac{\bar{\alpha}}{Re_0 Pr_0} + \frac{\mu_t}{Pr_t} \right) \frac{\partial \tilde{h}}{\partial x_j} \right] - R \frac{\partial q_r}{\partial x_j},\end{aligned}$$

- Radiative heat source:

$$\frac{\partial q_r}{\partial x_j} = \int_0^\infty \kappa_\lambda \left( 4\pi I_{b\lambda} - \int_{4\pi} I_\lambda d\Omega \right) d\lambda,$$

- Radiative transfer equation:

$$\int_0^\infty \frac{dI_\lambda}{ds} d\lambda = \int_0^\infty \kappa_\lambda (I_{b\lambda} - I_\lambda) d\lambda$$

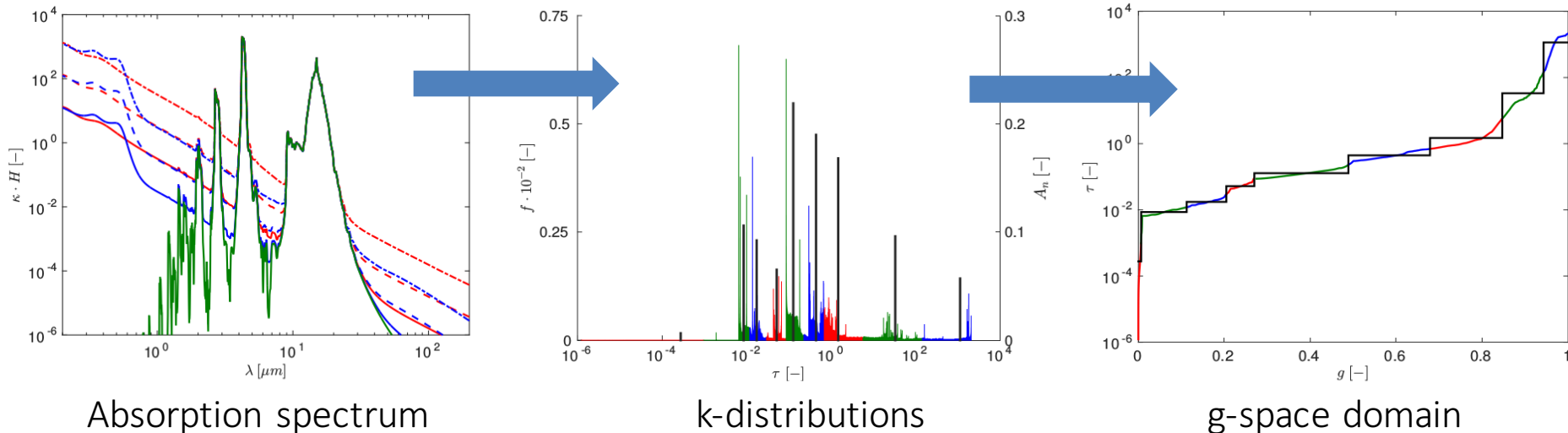
# Governing equations

Inhouse RANS and radiative transfer solver:

- Incompressible 2D RANS solver
  - Variable transport properties (NIST, Span & Wagner, 2004)
  - Finite volume method
  - SIMPLE solver
- k- $\epsilon$  turbulence model
  - Myong & Kasagi, 1990
- Radiation model
  - Finite volume method
  - Weighted sum of gray gases

# Radiation modeling

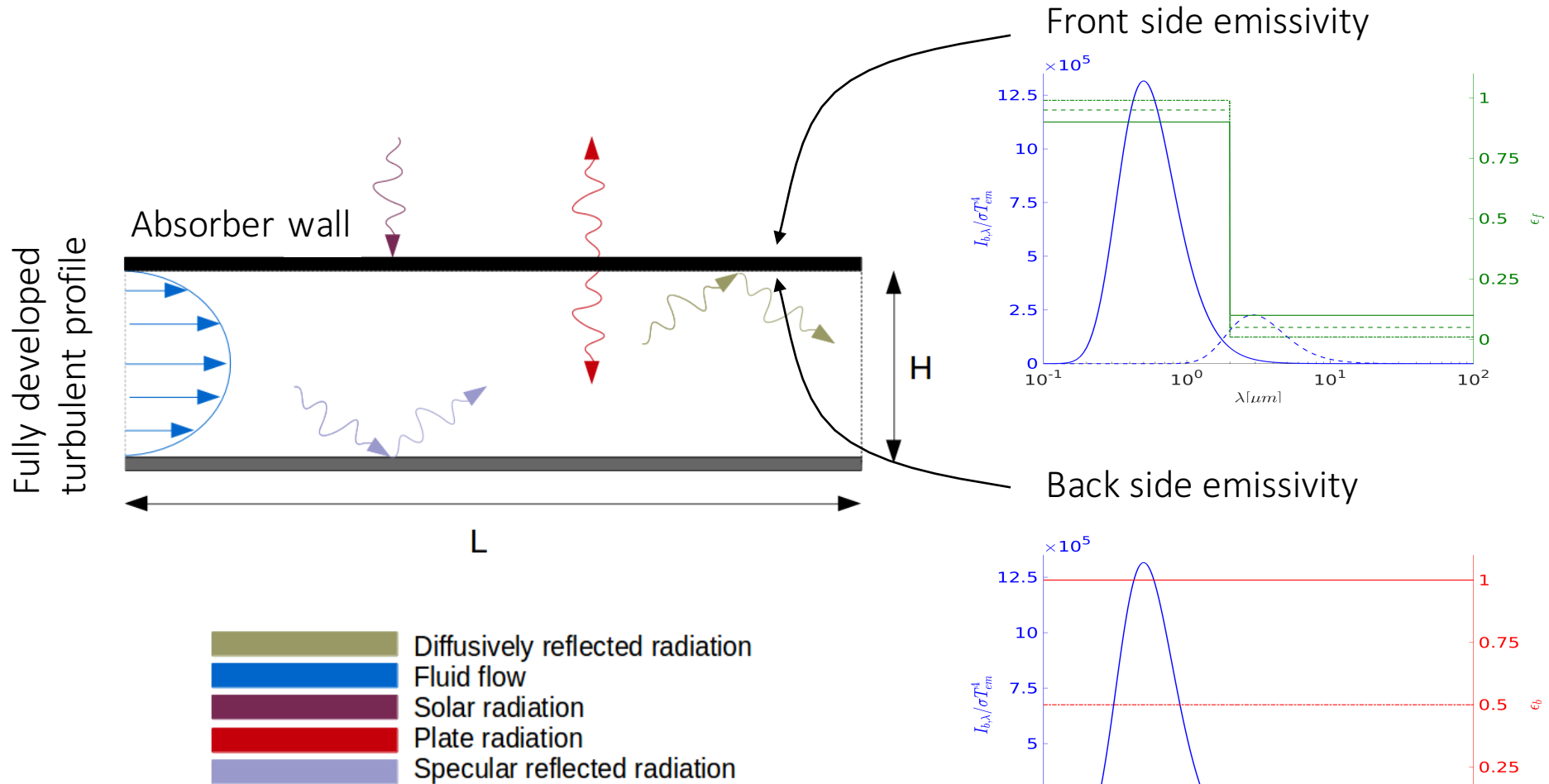
- Full k-distribution model (Modest, 2002)
  - Method for reducing number of equations for radiation
  - Convert absorption spectra from wavelength domain to g-space domain using k-distributions
  - Wavelengths with the same absorption coefficient are matched
  - Distributions are more smooth, therefore more suitable to fit
- Weighted Sum of Grey Gasses (WSGG) model
  - Approximation of k-distribution model (Modest, 2002)



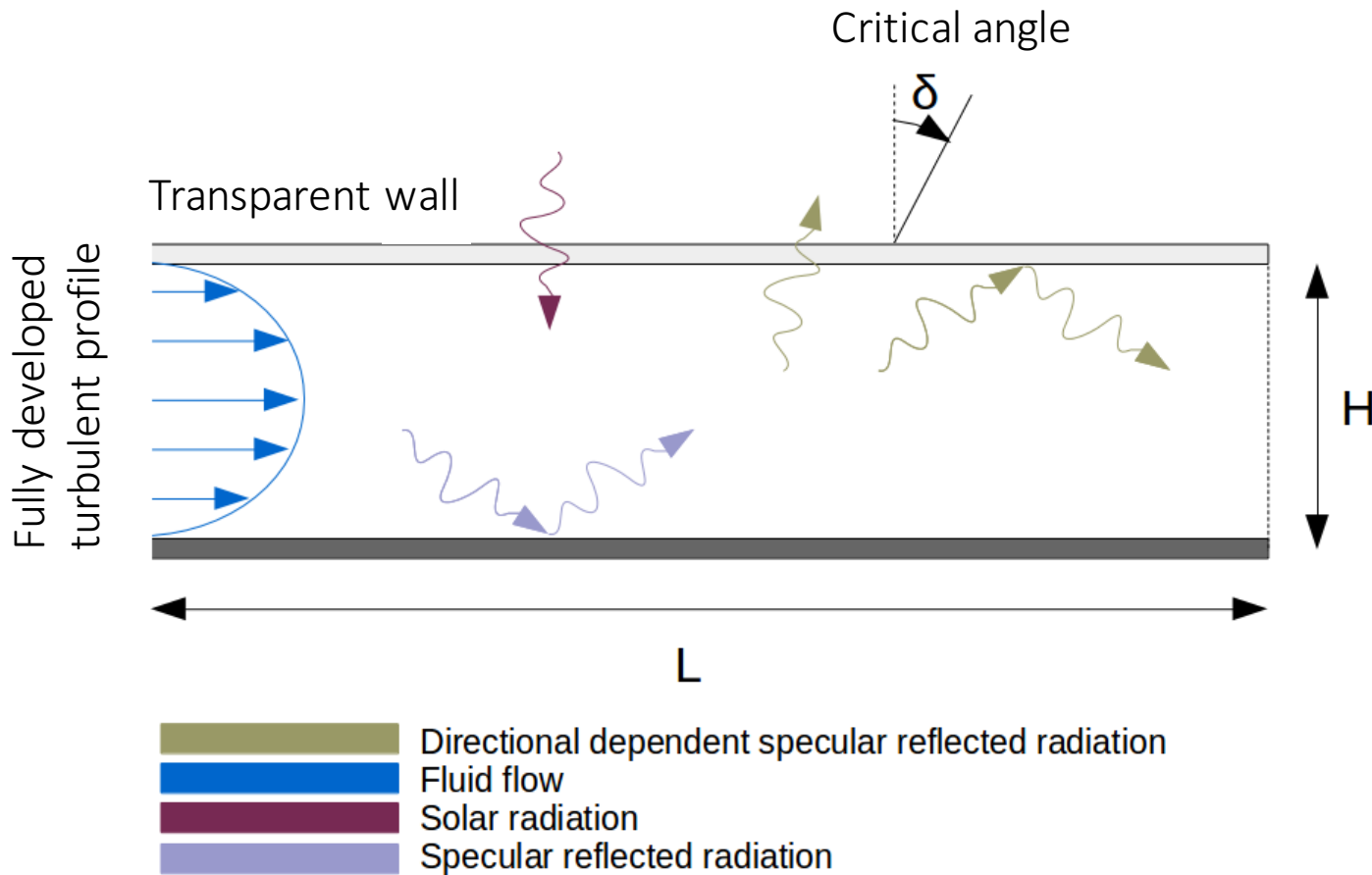


# Boundary conditions

# Boundary conditions: surface receiver



# Boundary conditions: volumetric receiver

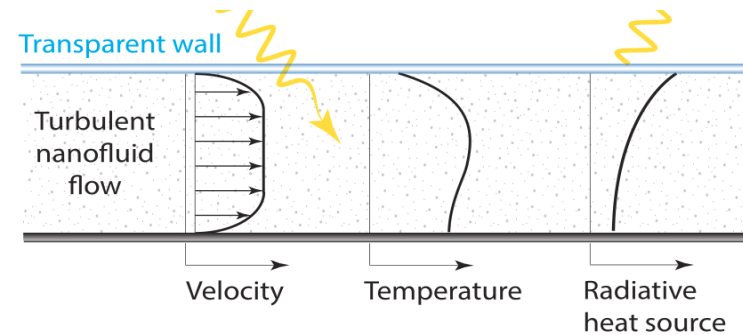
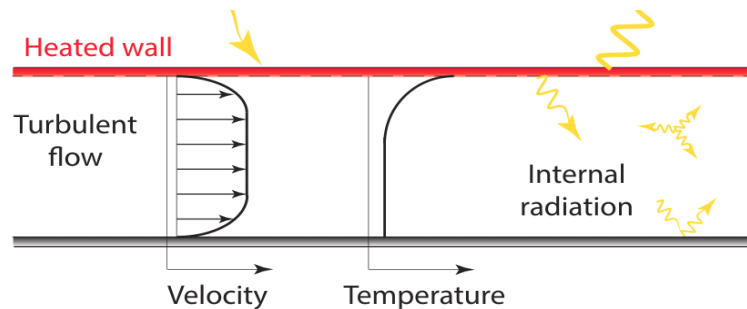


# Investigated parameter space

Overall boundary conditions	
Receiver inlet temperature	550 °C
Operating pressure	200 bar
Receiver length & height	4 m x 2.5 cm
Reynolds number	7000
Solar concentration factor	25 / 50 / 75

Surface receiver	
Front side emissivity	0.9 / 0.95 / 0.99
Back side emissivity	0.0 / 0.5 / 1.0

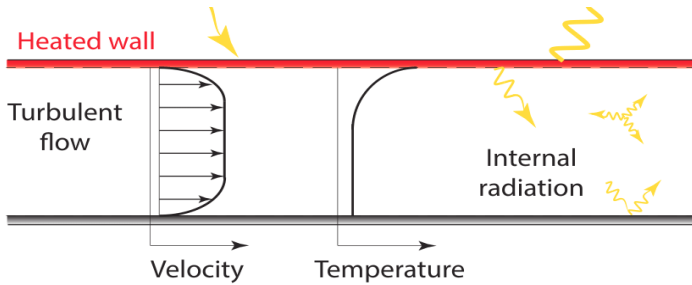
Volumetric receiver	
Volume fraction	$10^{-3}$ / $10^{-4}$ / $10^{-5}$
Critical angle	30° / 50° / 70°



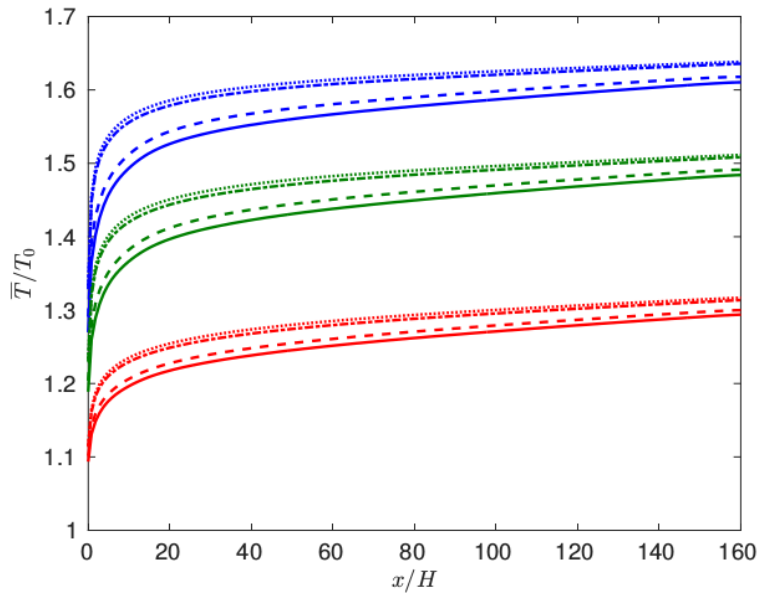
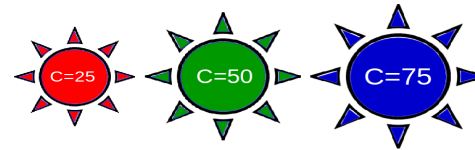
## Results

- Surface receiver
- Volumetric receiver

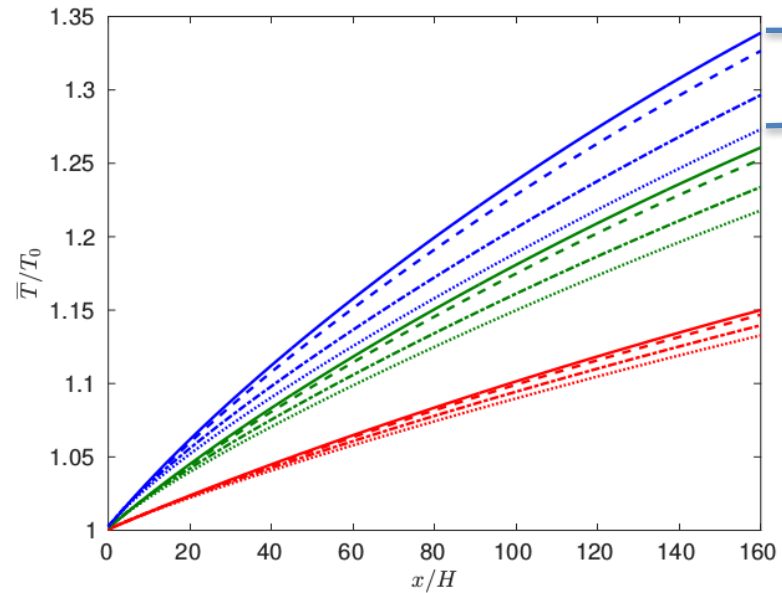
# Surface receiver: varying back side emissivity



- (—)  $\epsilon_b = 1$
- (- - -)  $\epsilon_b = 0.5$ ,
- (- · - ·)  $\epsilon_b = 0.0$
- (· · · ·) no radiative heat transfer



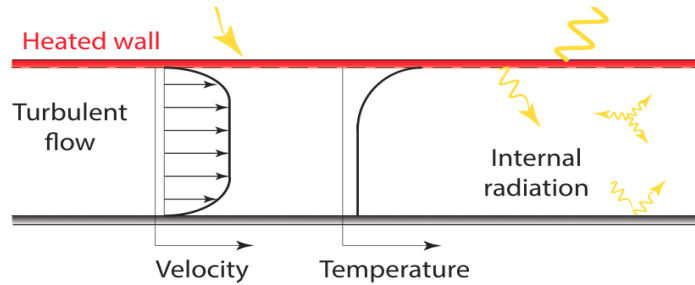
(a) Receiver plate temperature



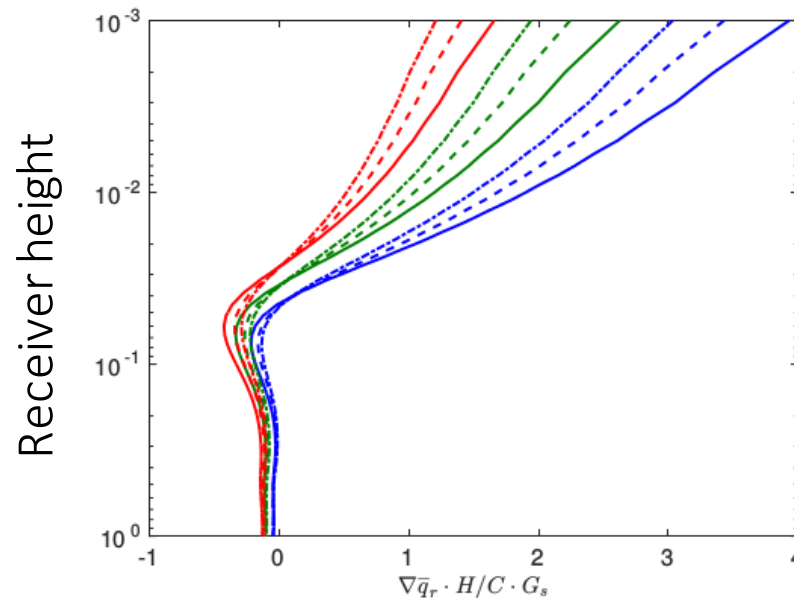
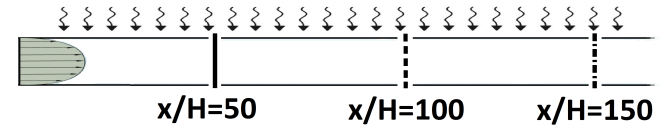
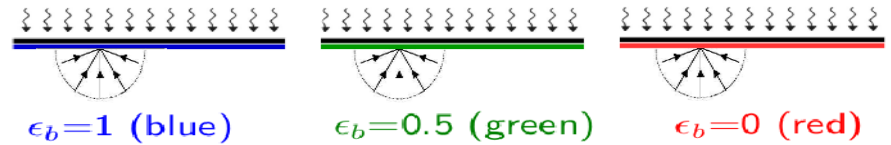
(b) Bulk temperature

8% higher with radiation

# Surface receiver: varying back side emissivity

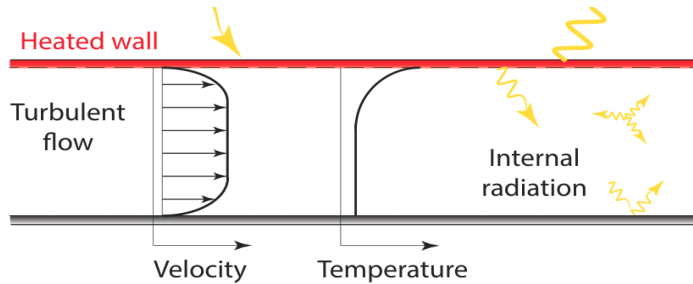


$[C=75, \epsilon_f = 0.9]$



(a) Volumetric heat source

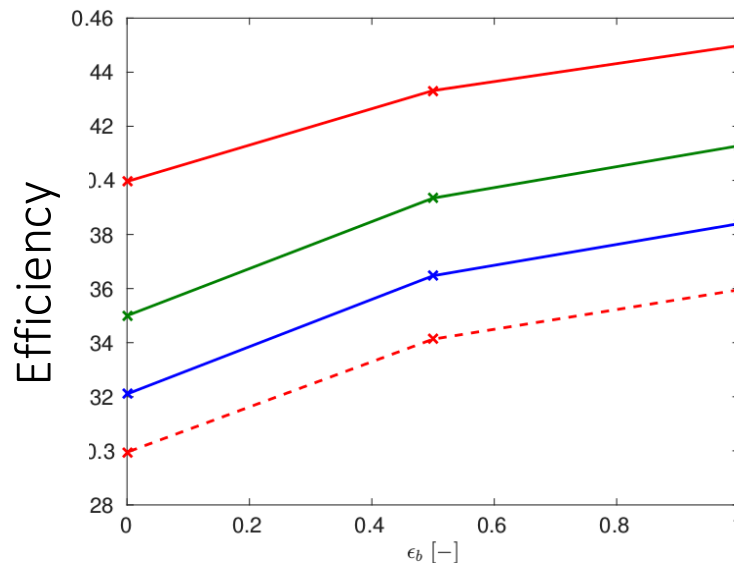
# Surface receiver: performance



$[C=75, \epsilon_f = 0.9]$

Efficiency definition:

$$\eta_{rec,x} = \frac{\dot{m}(h_x - h_{in})}{q_s A_{rec,x}}$$



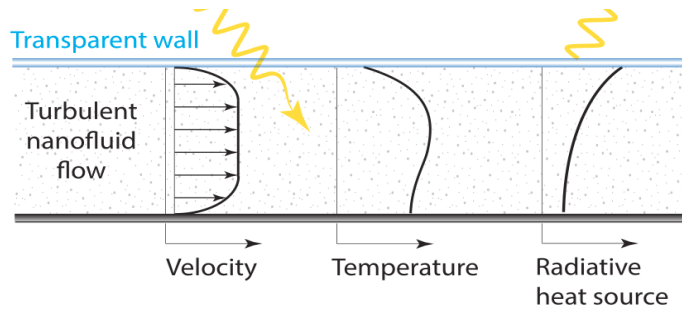
Desired receiver outlet temperature:

- x — 600 [°C]
- x — 650 [°C]
- x — 700 [°C]
- - x - - 750 [°C]

(a) Efficiency as a function of receiver plate back side emissivity  $[\epsilon_{f,1} = 0.9]$

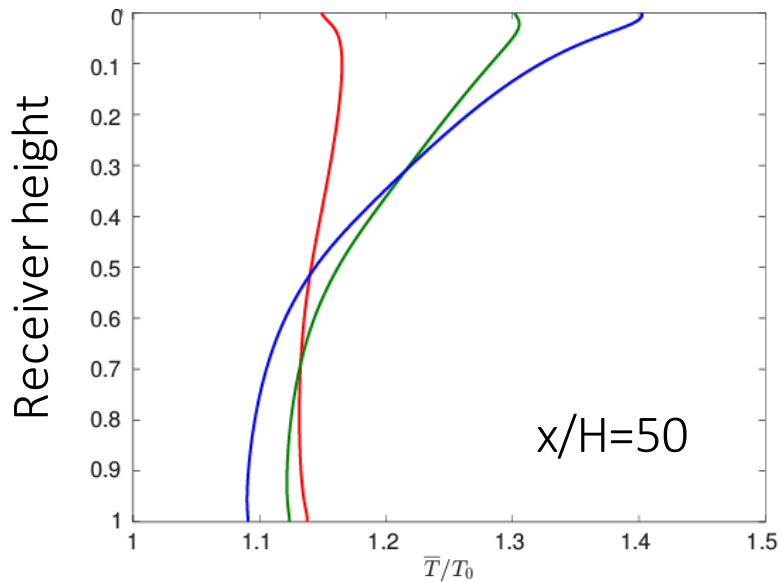
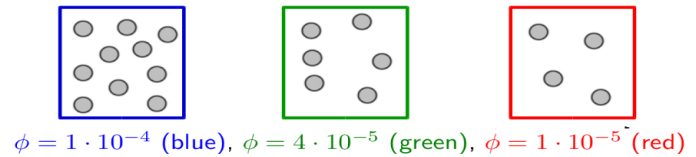


# Volumetric receiver: varying volume fraction

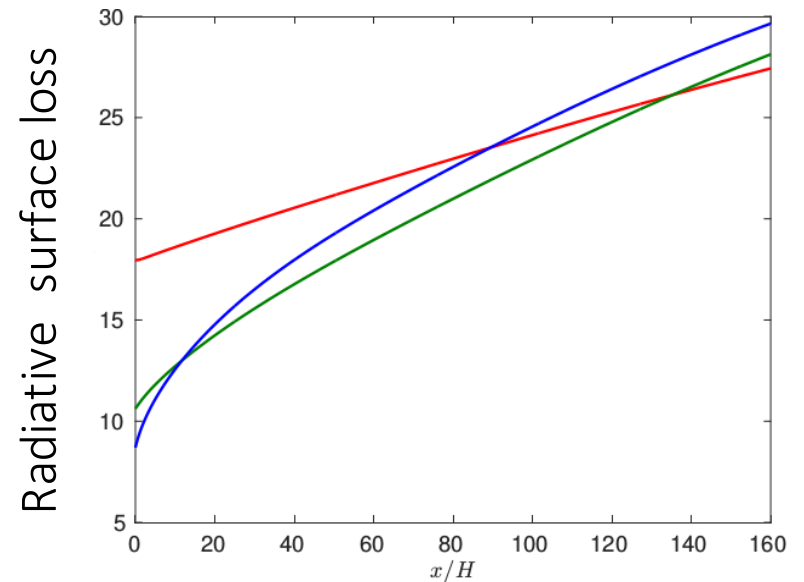


$$[C = 75, \delta_c = 70^\circ]$$

Volume fraction:

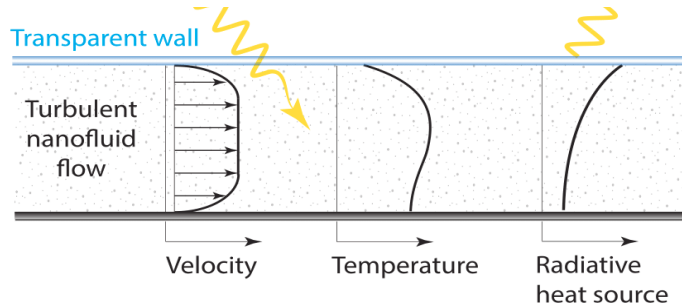


(a) Transverse contour lines of temperature:



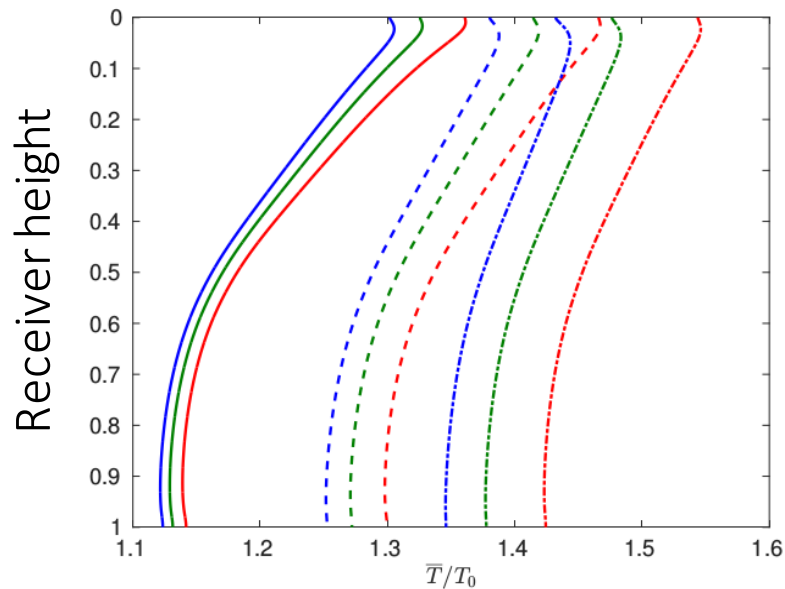
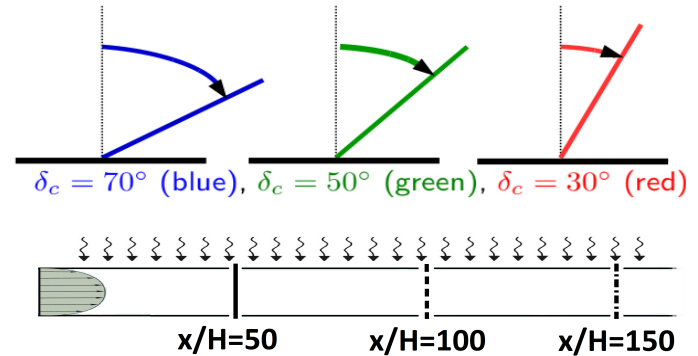
(b) Surface losses

# Volumetric receiver: varying critical angle

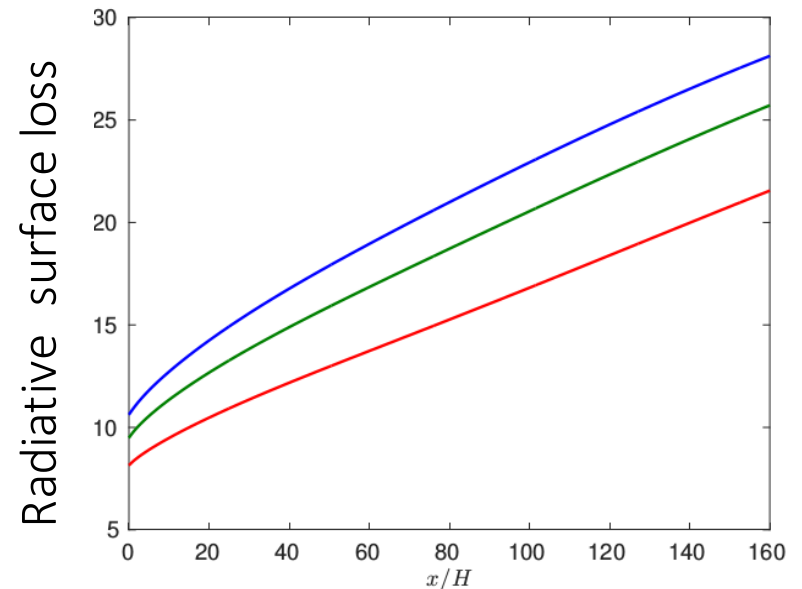


$$[C = 75, \phi = 4 \cdot 10^{-5}]$$

Critical angle:

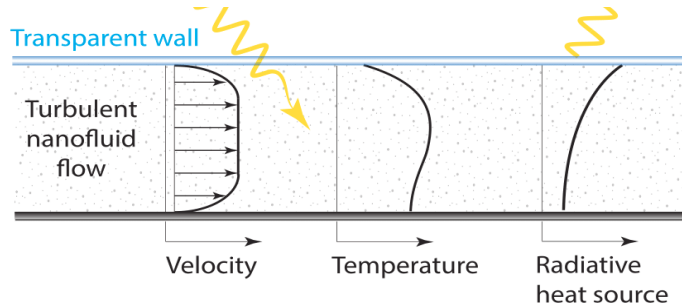


(a) Transverse contour lines of temperature:

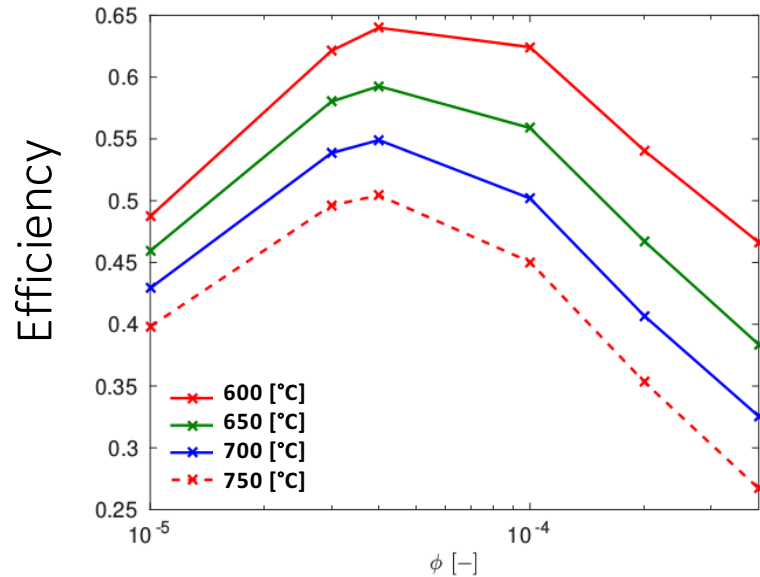
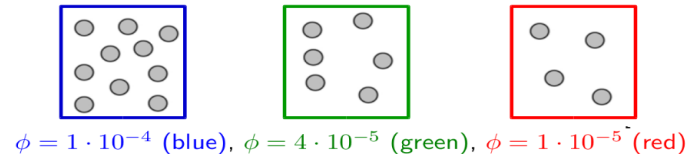


(b) Surface losses

# Volumetric receiver: performance

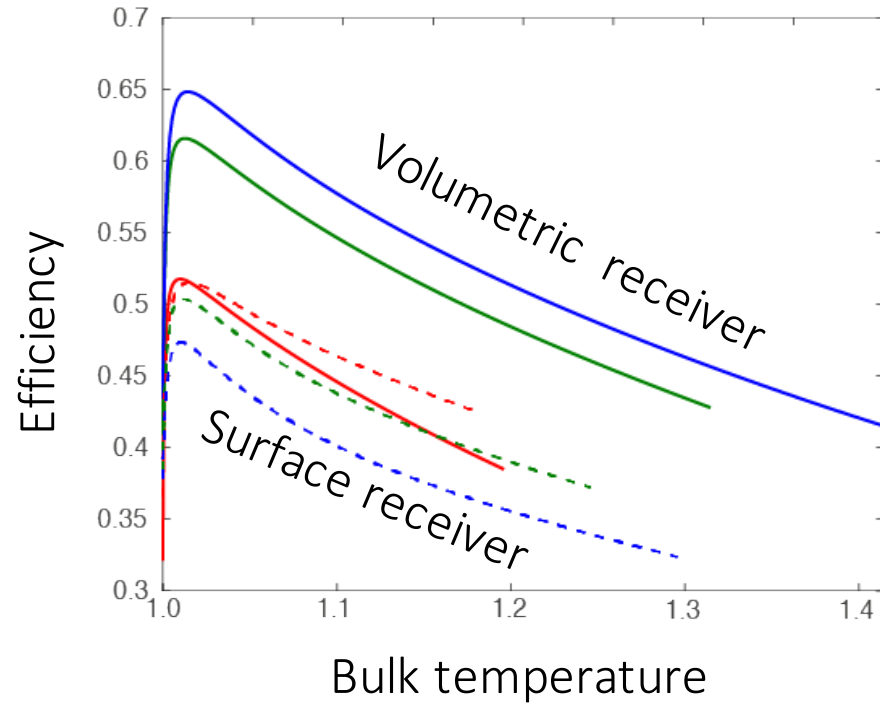


Volume fraction:



(a) Efficiency as a function of volume fraction  
 $[\delta_c = 70^\circ]$

# Performance comparison



Solar concentration:



# Conclusions

- Absorption peaks for CO<sub>2</sub> at 200 substantially broaden
- Radiative transfer has to be taken into account in surface receivers
  - efficiency 8% higher with radiation
- Substantial higher efficiencies for volumetric receivers for high solar concentration factors
  - Low radiative losses for volumetric receiver -> low surface temperature
  - Radiative losses limited to a thin layer close to the transparent surface

# Recommendations

- Turbulent radiation interaction for optically thick media is unknown
- Experiments are crucial to prove concept with nanoparticles in scCO<sub>2</sub>
  - Nanoparticle agglomeration and thermal/chemical stability, etc.
- Volumetric receiver design challenging for high pressures