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EXPERIMENT AND NUMERICAL STUDY OF SUPERCRITICAL CARBON DIOXIDE FLOW THROUGH LABYRINTH SEALS

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Outline

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- Introduction
- Experiment facility
- Numerical method
- Results discussion
- Conclusion

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Introduction

- Why labyrinth seal?
- Labyrinth seals are widely used in turbomachinery. Or used as a part of more complicated seals, such as dry gas seal.
- VENT SEPARATION SEAL GAS SUPPLY GAS SUPPLY PROCESS BEARING SIDE SIDE INNER PRIMARY BARRIER LABYRINTH GAS SEAL SEAL SEAL
- In Sandia SCO2 Brayton Cycle Experiment loop, simple labyrinth seal is used for compressor.





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Introduction

- Why new model?
- Old empirical methods based on incompressible fluid or ideal gas are not working for S-CO₂.
- As a result, new method is needed.
- This method should also applicable for future more complicated seal design for S-CO₂.
- Computational Fluid Dynamic code combined with S-CO₂ property is a good option.



Experiment facility

- A test facility was constructed at UW-Madison to measure the leakage rate through labyrinth seals.
- A Hydro-Pac single-stage compressor is used to compress CO₂ to supercritical state.
- Heater and cooler to help reach target inlet temperature.



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Experiment facility

- The test section is formed by a sub-assembly and a flange.
- Various labyrinth seal designs can be tested by changing the formation seals.
- A pressure vessel to help stabilize downstream condition.
- Flowmeters, thermocouples, pressure transducers are used to measure flow rate, temperature and pressure.



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Numerical method

- OpenFOAM is used as the framework of numerical study.
- It is an open source CFD code, which brings a lot of flexibility.
- The compressible solver was modified to use S-CO₂ properties in simulation.
- The S-CO₂ properties were provided by a library called FIT, which is developed by Northland Numerics. FIT use an interpolated representation of properties based on REFPROP.







Numerical method

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- Geometry.
- Currently, only see-through labyrinth seals are considered.
- As we can see, to define a seethrough labyrinth seal, we need seven parameters.
- However, with one inherent correlation: L_{total} =n*L_{seal} + (n-1)*L_{cavity}
- The degree of freedom is six.

| Description | Shaft diameter | Shaft clearance | Cavity Height(Cavity depth) | Cavity Length(Cavity width) | Seal Length(Tooth Width) | Total length | Number teeth | of |
|-------------|----------------|--------------------|-----------------------------------|-----------------------------------|--------------------------------|--------------------|-----------------|----|
| Notation | D | С | Н | L _{cavity} | L _{seal} | L _{total} | n | |
| Value | 3mm | 0.105mm | 0.88mm | 1.27mm | 1.27mm | 3.81mm | 2 | |





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Numerical method

- SIMPLEC algorithm is used for pressure and velocity coupling.
- Standard k-epsilon model.
- Re~200,000
- Boundary condition
- To know the leakage rate corresponding to different pressure drop, fixed pressure is given at inlet and outlet. Fixed temperature is given at inlet.
- At wall, non-slip and adiabatic condition are set.
- To save computational time, pseudo-axisymmetric method is used.
- Currently, the shaft is stationary





Numerical method

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- In the discharge process, two-phase may appears when downstream condition is inside two-phase dome.
- In this research, homogeneous equilibrium model is used for two-phase modeling.

$$v_{two-phase} = x \cdot v_{gas} + (1-x) \cdot v_{liquid}$$

$$h_{two-phase} = x \cdot h_{gas} + (1-x) \cdot h_{liquid}$$

$$\mu_{two-phase} = x \cdot \mu_{gas} + (1 - x) \cdot \mu_{liquid}$$

$$\kappa_{\mathrm{two-phase}} = x \cdot \kappa_{\mathrm{gas}} + (1 - x) \cdot \kappa_{\mathrm{liquid}}$$



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- Radial clearance (c)
- This parametric study is based on three-teeth seals.
- Inlet condition is (10MPa, 498 kg/m3), outlet pressure is fixed at 5MPa.
- These results show that as radial clearance increases, flow area increases, thus more mass can flow through the seal.







- Cavity length(L_{cavity})
- Total length is kept constant.
- L_{seal} is changed according to the change of L_{cavity}
- Larger L_{cavity} will results more radial expansion of main stream, which results a larger contraction form loss at next tooth.





Main stream

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Expansion angle

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- Cavity Height (H)
- The effect of cavity height is relative complicated.
- As an optimum value of H results in minimum leakage.











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- Number of tooth (n)
- The total length and seal width are kept constant. Tooth is inserted one by one to see the change.
- Inlet condition of (10 Mpa, 325 kg/m3), outlet pressure of 5 Mpa.
- As more tooth in seal, more contraction form loss is introduced.
- However, the space between teeth is also reduced.





Conclusion

- Experiment facility was build in UW-Madison for various seal test for S-CO₂
- Experiment data can serve as a data base for future researchers.
- Numerical method is calibrated by experiment results
- Parametric study of several geometric parameters provides guiding for design optimization.



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THANK YOU

